

Symmetric Coupling of the Meshless Galerkin Boundary Node and Finite Element Methods for Elasticity

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Abstract: Combining moving least square (MLS) approximations and boundary integral equations, a symmetric and boundary-only meshless method, the Galerkin boundary node method (GBNM), is developed in this paper for two- and three-dimensional elasticity problems with mixed boundary conditions. Unlike other MLS-based meshless methods, boundary conditions in this meshless method can be applied directly and easily. In the GBNM, the stiffness matrices so obtained are symmetric. The property of symmetry is an added advantage in coupling the GBNM with the finite element method (FEM). Thus, a symmetric coupling of the GBNM and the FEM is also discussed for elasticity problems. Error analysis and convergence study of the GBNM and the coupled GBNM-FEM are given in Sobolev spaces. For demonstration purpose, some numerical examples are presented.

Keywords: Meshless method, Galerkin boundary node method, finite element method, coupled method, convergence, elasticity

1 Introduction

Meshless (or meshfree) methods have been proposed and achieved remarkable progress in the past two decades [Atluri (2004); Li and Liu (2004); Liu (2009)]. Compared with traditional mesh-based numerical methods such as the finite element method (FEM) and the boundary element method (BEM), meshless methods get rid of, or at least alleviate, the difficulty of meshing and remeshing the entire structure by simply adding or deleting nodes. Meshless methods have developed so fast that they are applied successfully to a variety of science and engineering problems.

The moving least square (MLS) is an approximation scheme of constructing continuous functions from a set of unorganized sampled point values. Since the numerical approximations start from scattered nodes instead of elements, the MLS scheme is

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one of the most extensively used schemes to form the meshless shape functions. Some MLS-based meshless methods, such as the element-free Galerkin (E-FG) method [Liu (2009)], the h - p meshless method [Duarte and Oden (1996)], the moving least square reproducing kernel method (MLSRKM) [Li and Liu (1996)] and the meshless local Petrov-Galerkin (MLPG) method [Sladek, Stanak, and Han (2013)] have been developed. They are domain type, as the FEM, in which the problem domain is discretized by nodes.

The boundary integral equation (BIE) is an important and attractive computational tool as it can reduce the dimensionality of the considered problem by one. The boundary type meshless methods are developed by the combination of the meshless idea with BIEs, such as the boundary node method (BNM) [Mukherjee and Mukherjee (2005)], the boundary cloud method (BCM) [Li and Aluru (2002)], the hybrid boundary node method (HBNM) [Miao, He, and Luo (2012)] and the boundary face method (BFM) [Zhang, Qin, Han, and Li (2009)]. In these methods, the MLS scheme is used to generate the shape functions on the boundary of a domain. These methods take the advantages of both the BIE in dimension reduction and the MLS scheme in elements elimination. However, since the MLS scheme lacks the delta function property, they cannot exactly satisfy boundary conditions. The technique used in the BNM to impose boundary conditions doubles the number of system equations. This technique is also used in the BCM, the HBNM and the BFM, together with the addition of a penalty formulation.

Liew, Cheng, and Kitipornchai (2006) developed an improved MLS scheme that uses weighted orthogonal polynomials as basis functions. The improved MLS scheme has been introduced into BIEs to develop a boundary element-free method (BEFM). Because the improved MLS scheme still lacks the delta function property, boundary conditions in the BEFM are implemented with constraints. To construct meshless shape functions with delta function properties, Li and Li (2014) discussed mathematically an improved interpolating MLS scheme and developed an interpolating BEFM for potential problems and unilateral problems. Besides, Liu *et al.* developed the point interpolation method (PIM) and introduced it into BIEs to produce boundary PIMs [Gu and Liu (2003); Liu (2009)]. Recently, Li (2014) developed a dual boundary node method (DBNM) for implementation of boundary conditions in BIEs-based meshless methods. In the DBNM, boundary conditions are introduced directly into dual BIEs including the conventional BIE and hypersingular BIE. Consequently, the DBNM can apply boundary conditions directly and easily, and the number of both unknowns and system equations in the DBNM is only half of that in the BNM.

Li and Zhu (2009b) and Li (2011a) developed a boundary type meshless method called the Galerkin boundary node method (GBNM). It combines the MLS scheme

with a variational (weak) version of BIEs. The MLS scheme is implemented for constructing the trial and test functions of the variational form, thus only the boundary of a problem domain is discretized by a set of scattered nodes instead of elements. Unlike other MLS-based methods mentioned above, boundary conditions in the GBNM do not present any difficulty and can be implemented with ease via multiplying the MLS shape function and integrating on the boundary. The GBNM has been applied to elastic problems with pure displacement boundary conditions [Li and Zhu (2009a)] and pure traction boundary conditions [Li and Li (2013)]. It is well known that mixed boundary value problems play an important role in many different applications of physics, mechanics and engineering. In this paper, the GBNM is further developed for solving elastic problems with mixed boundary conditions of displacement and traction type.

In contrast with other boundary type meshless methods aforementioned, another outstanding feature of the GBNM is the conservation of the symmetry and positive definiteness of the variational problems in the process of numerical implementation. The property of symmetry can be an added advantage in coupling the GBNM with other numerical methods. Some coupled methods, such as the coupled BEM and FEM [Brebbia and Georgion (1979); Stephan (2004); Beer (2001); Ganguly, Layton, and Balakrishna (2000); Haas and Kuhn (2003); Dong and Atluri (2012a,b, 2013)], the coupled EFG and FEM [Belytschko and Organ (1995)], the coupled MLPG and FEM [Liu (2009)], the coupled MLPG and BEM [Tadeu, Stanak, and Sladek (2013)], the coupled improved EFG and BEM [Zhang, Liew, and Cheng (2008)], and the coupled reproducing kernel particle boundary element-free method (RKPBEFM) and FEM [Qin and Cheng (2008)], have been developed. In this paper, based on the coupled techniques propose by Ganguly, Layton, and Balakrishna (2000), Haas and Kuhn (2003), Zhang, Liew, and Cheng (2008) and Qin and Cheng (2008), a direct symmetric coupling of the GBNM and the FEM is also developed for elasticity problems. In the present coupled method, the resulting coupling matrix is symmetric and positive definite.

Error analysis and convergence study, which ensure convergence of numerical methods, are crucial in meshless research. The associated mathematical proofs guarantee that meshless methods will converge to the true solution. Over the past two decades, it has been developed so fast in the areas of meshless research from both computational and mathematical point of views. A large amount of research has been devoted to deriving error estimation for MLS-based domain type meshless methods such as the h - p meshless method [Duarte and Oden (1996)], the MLSRK-M [Li and Liu (1996)] and the finite point method [Cheng and Cheng (2008)]. Nevertheless, although boundary type meshless methods perform very well in practice, not much is rigorously known on the mathematical foundation of these schemes.

Until now, a rigorous mathematical analysis of boundary type meshless methods was given for the GBNM for potential problems [Li and Zhu (2009b); Li (2011a, 2012)], for Stokes problems [Li and Zhu (2009c); Li (2011b)] and for elasticity problems with pure displacement or traction boundary conditions [Li and Zhu (2009a); Li and Li (2013)]. Thus, one aim of this paper is to provide a solid mathematical foundation to the GBNM for the mixed boundary value problems of elastostatics. Besides, the error analysis and convergence study of the coupled GBNM-FEM are also presented in Sobolev spaces.

An outline of this paper is as follows. In Section 2 we give a detailed numerical implementation and error analysis of the GBNM for elasticity problems with mixed boundary conditions. Section 3 deals with the GBNM-FEM coupling approach and the corresponding error analysis. Numerical examples are presented in Section 4. Section 5 contains conclusions.

2 The GBNM for mixed elasticity problems

2.1 BIEs

Let Ω be a bounded or unbounded domain in \mathbb{R}^d ($d = 2, 3$) with boundary $\Gamma = \Gamma_u \cup \Gamma_t$, $\Gamma_u \cap \Gamma_t = \emptyset$, $\Gamma_u \neq \emptyset$, with given displacement data on Γ_u , and traction data on Γ_t . In linear elasticity for isotropic materials, the governing equation is

$$\mu \Delta \mathbf{u} + (\lambda + \mu) \nabla (\nabla \cdot \mathbf{u}) = 0, \quad \text{in } \Omega \tag{1}$$

where $\mathbf{u} = (u_1, u_2, \dots, u_d)^T$ is the displacement field; λ and μ are classical Lamé constants; Δ , ∇ and $\nabla \cdot$ stand for the Laplacian, gradient and divergence operators, respectively. Suitable boundary conditions are associated with this field equation. They can be of the following types:

$$\mathbf{u} = \bar{\mathbf{u}}, \quad \text{on } \Gamma_u \tag{2}$$

$$\mathbf{t} = \boldsymbol{\sigma} \cdot \mathbf{n} = \bar{\mathbf{t}}, \quad \text{on } \Gamma_t \tag{3}$$

where $\mathbf{t} = (t_1, t_2, \dots, t_d)^T$ is the boundary traction, $\boldsymbol{\sigma}$ is the stress tensor, \mathbf{n} is the outward normal direction on Γ , and $\bar{\mathbf{u}} = (\bar{u}_1, \bar{u}_2, \dots, \bar{u}_d)^T$ and $\bar{\mathbf{t}} = (\bar{t}_1, \bar{t}_2, \dots, \bar{t}_d)^T$ are the given displacement and traction, respectively.

Eqs. (1)-(3) compose the standard mixed boundary value problem of linear elasticity. The associated BIE is [Zhu and Yuan (2009)]

$$\mathbf{u}(\mathbf{x}) = \int_{\Gamma} \mathbf{U}(\mathbf{x}, \mathbf{y}) \mathbf{t}(\mathbf{y}) d\Gamma(\mathbf{y}) - \int_{\Gamma} \mathbf{T}_y(\mathbf{x}, \mathbf{y}) \mathbf{u}(\mathbf{y}) d\Gamma(\mathbf{y}), \quad \mathbf{x} \in \Omega \tag{4}$$

where $\mathbf{x} = (x_1, x_2, \dots, x_d)^T$ is the source point, $\mathbf{y} = (y_1, y_2, \dots, y_d)^T$ is the field point, and $\mathbf{U}(\mathbf{x}, \mathbf{y}) = [\mathbf{U}_{ij}(\mathbf{x}, \mathbf{y})]_{i,j=1}^d$ is the fundamental solution of the Lamé system (1),

$$\mathbf{U}(\mathbf{x}, \mathbf{y}) = \frac{\lambda + 3\mu}{4\pi(d-1)\mu(\lambda + 2\mu)} \left[E(\mathbf{x}, \mathbf{y}) \mathbf{I} + \frac{\lambda + \mu}{\lambda + 3\mu} \frac{(\mathbf{x} - \mathbf{y})(\mathbf{x} - \mathbf{y})^T}{r^d} \right]$$

Here, \mathbf{I} is the identity matrix, $r = |\mathbf{x} - \mathbf{y}|$, $E(\mathbf{x}, \mathbf{y}) = -\ln r$ for $d = 2$ and $E(\mathbf{x}, \mathbf{y}) = 1/r$ for $d = 3$. Let \mathcal{T} be the differential operator which transforms a displacement field in Ω into the corresponding traction on its boundary. When the derivatives are taken with respect to \mathbf{x} or \mathbf{y} , then we denote it by $\mathcal{T}_{\mathbf{x}}$ and $\mathcal{T}_{\mathbf{y}}$, respectively. Under this notation, $\mathbf{T}_{\mathbf{y}}(\mathbf{x}, \mathbf{y}) = (\mathcal{T}_{\mathbf{y}}\mathbf{U}(\mathbf{x}, \mathbf{y}))^T$ is the strongly singular fundamental solution.

In Eq. (4), letting \mathbf{x} tend to Γ , we obtain the strongly singular displacement BIE

$$\frac{1}{2}\mathbf{u}(\mathbf{x}) = (\mathcal{V}_{\Gamma}\mathbf{t})(\mathbf{x}) - (\mathcal{K}_{\Gamma}\mathbf{u})(\mathbf{x}), \quad \mathbf{x} \in \Gamma \quad (5)$$

Then applying the operator $\mathcal{T}_{\mathbf{x}}$ to Eq. (5) yields the hypersingular traction BIE

$$\frac{1}{2}\mathbf{t}(\mathbf{x}) = (\mathcal{K}'_{\Gamma}\mathbf{t})(\mathbf{x}) + (\mathcal{D}_{\Gamma}\mathbf{u})(\mathbf{x}), \quad \mathbf{x} \in \Gamma \quad (6)$$

In Eqs. (5) and (6), we have used the standard notations for the boundary integral operators defined on Γ ,

$$(\mathcal{V}_{\Gamma}\mathbf{t})(\mathbf{x}) = \int_{\Gamma} \mathbf{U}(\mathbf{x}, \mathbf{y}) \mathbf{t}(\mathbf{y}) d\Gamma(\mathbf{y})$$

$$(\mathcal{K}_{\Gamma}\mathbf{u})(\mathbf{x}) = \int_{\Gamma} \mathbf{T}_{\mathbf{y}}(\mathbf{x}, \mathbf{y}) \mathbf{u}(\mathbf{y}) d\Gamma(\mathbf{y})$$

$$(\mathcal{K}'_{\Gamma}\mathbf{u})(\mathbf{x}) = \int_{\Gamma} \mathbf{T}_{\mathbf{x}}(\mathbf{x}, \mathbf{y}) \mathbf{u}(\mathbf{y}) d\Gamma(\mathbf{y})$$

and

$$(\mathcal{D}_{\Gamma}\mathbf{u})(\mathbf{x}) = - \int_{\Gamma} \mathbf{S}(\mathbf{x}, \mathbf{y}) \mathbf{u}(\mathbf{y}) d\Gamma(\mathbf{y})$$

Here, $\mathbf{T}_{\mathbf{x}}(\mathbf{x}, \mathbf{y}) = \mathcal{T}_{\mathbf{x}}\mathbf{U}(\mathbf{x}, \mathbf{y})$ and $\mathbf{S}(\mathbf{x}, \mathbf{y}) = \mathcal{T}_{\mathbf{x}}\mathbf{T}_{\mathbf{y}}(\mathbf{x}, \mathbf{y})$ are the strongly singular and hypersingular fundamental solutions, respectively.

Obviously, the kernel functions are symmetric, i.e., $\mathbf{U} = \mathbf{U}^T$, $\mathbf{S} = \mathbf{S}^T$ and $\mathbf{T}_{\mathbf{y}} = \mathbf{T}_{\mathbf{x}}^T$. Thus, to find the complete Cauchy data $[\mathbf{u}, \mathbf{t}]_{\Gamma}$ and to achieve a symmetric formulation, Eq. (5) is used where the boundary traction \mathbf{t} is unknown, while Eq.

(6) is used where the boundary displacement \mathbf{u} is unknown. Then according to boundary conditions (2) and (3), we get the following BIEs:

$$(\mathcal{V}_{\Gamma_u} \mathbf{t})(\mathbf{x}) - (\mathcal{K}_{\Gamma_t} \mathbf{u})(\mathbf{x}) = \frac{1}{2} \bar{\mathbf{u}}(\mathbf{x}) + (\mathcal{K}_{\Gamma_u} \bar{\mathbf{u}})(\mathbf{x}) - (\mathcal{V}_{\Gamma_t} \bar{\mathbf{t}})(\mathbf{x}), \quad \mathbf{x} \in \Gamma_u \tag{7}$$

$$(\mathcal{K}'_{\Gamma_u} \mathbf{t})(\mathbf{x}) + (\mathcal{D}_{\Gamma_t} \mathbf{u})(\mathbf{x}) = \frac{1}{2} \bar{\mathbf{t}}(\mathbf{x}) - (\mathcal{K}'_{\Gamma_t} \bar{\mathbf{t}})(\mathbf{x}) - (\mathcal{D}_{\Gamma_u} \bar{\mathbf{u}})(\mathbf{x}), \quad \mathbf{x} \in \Gamma_t \tag{8}$$

2.2 Variational formulation

Let

$$H^\tau(\Gamma) = \left\{ u|_\Gamma : u \in H^{\tau+1/2}(\mathbb{R}^d) \right\}, \quad \mathbf{H}^\tau(\Gamma) := (H^\tau(\Gamma))^d, \quad \tau \geq 0$$

denote the usual Sobolev space of functions defined on Γ [Zhu and Yuan (2009)]. In the following, we often write $\|\cdot\|_{\tau,\Gamma}$ for the Sobolev norm $\|\cdot\|_{\mathbf{H}^\tau(\Gamma)}$.

Besides, let $\mathbf{H}^{-\tau}(\Gamma)$ be the dual space of $\mathbf{H}^\tau(\Gamma)$ with respect to the duality $\langle \cdot, \cdot \rangle_\Gamma$ which is defined for functions w and v by

$$\langle w, v \rangle_\Gamma = \int_\Gamma w(\mathbf{y}) v(\mathbf{y}) d\Gamma(\mathbf{y})$$

Moreover, let

$$\mathcal{H}^\tau(\Gamma) = \mathbf{H}^\tau(\Gamma_u) \times \mathbf{H}^{\tau+1}(\Gamma_t), \quad \tau \in \mathbb{R}$$

with norm

$$\|(\mathbf{t}, \mathbf{u})\|_{\tau,\Gamma}^2 = \|\mathbf{t}\|_{\tau,\Gamma_u}^2 + \|\mathbf{u}\|_{\tau+1,\Gamma_t}^2$$

Then Eqs. (7) and (8) lead to the following variational problem:

$$\begin{cases} \text{Find } (\mathbf{t}, \mathbf{u}) \in \mathcal{H}^{-1/2}(\Gamma) \text{ such that} \\ B((\mathbf{t}, \mathbf{u}), (\mathbf{t}', \mathbf{u}')) = L(\mathbf{t}', \mathbf{u}'), \quad \forall (\mathbf{t}', \mathbf{u}') \in \mathcal{H}^{-1/2}(\Gamma) \end{cases} \tag{9}$$

where

$$B((\mathbf{t}, \mathbf{u}), (\mathbf{t}', \mathbf{u}')) = \langle \mathcal{V}_{\Gamma_u} \mathbf{t}, \mathbf{t}' \rangle_{\Gamma_u} - \langle \mathcal{K}_{\Gamma_t} \mathbf{u}, \mathbf{t}' \rangle_{\Gamma_u} + \langle \mathcal{K}'_{\Gamma_u} \mathbf{t}, \mathbf{u}' \rangle_{\Gamma_t} + \langle \mathcal{D}_{\Gamma_t} \mathbf{u}, \mathbf{u}' \rangle_{\Gamma_t}$$

$$L(\mathbf{t}', \mathbf{u}') = \left\langle \frac{1}{2} \bar{\mathbf{u}} + \mathcal{K}_{\Gamma_u} \bar{\mathbf{u}} - \mathcal{V}_{\Gamma_t} \bar{\mathbf{t}}, \mathbf{t}' \right\rangle_{\Gamma_u} + \left\langle \frac{1}{2} \bar{\mathbf{t}} - \mathcal{K}'_{\Gamma_t} \bar{\mathbf{t}} - \mathcal{D}_{\Gamma_u} \bar{\mathbf{u}}, \mathbf{u}' \right\rangle_{\Gamma_t}$$

The unique solvability of the variational problem (9) follows from the continuity of the boundary integral operators introduced above and the coerciveness of the operators \mathcal{V} and \mathcal{D} .

2.3 Approximation

Let $\{\mathbf{x}_i\}_{i=1}^N$ be a set of N boundary nodes $\mathbf{x}_i \in \Gamma$ and let

$$h = \max_{1 \leq i \leq N} \min_{1 \leq j \leq N} |\mathbf{x}_i - \mathbf{x}_j|$$

represent the nodal spacing. To simplify the representation, we further assume that the first N_u boundary nodes $\{\mathbf{x}_i\}_{i=1}^{N_u} \subset \Gamma_u$ and the rest $N - N_u$ nodes $\{\mathbf{x}_i\}_{i=N_u+1}^N \subset \Gamma_t$.

Based on $\{\mathbf{x}_i\}_{i=1}^N$ alone, the MLS approximation of \mathbf{u} and \mathbf{t} can be written as

$$\mathbf{t}(\mathbf{x}) \approx \mathcal{M}\mathbf{t}(\mathbf{x}) = \sum_{i=1}^{N_u} \Phi_i(\mathbf{x}) \mathbf{t}_i, \quad \mathbf{x} \in \Gamma_u \tag{10}$$

$$\mathbf{u}(\mathbf{x}) \approx \mathcal{M}\mathbf{u}(\mathbf{x}) = \sum_{i=N_u+1}^N \Phi_i(\mathbf{x}) \mathbf{u}_i, \quad \mathbf{x} \in \Gamma_t \tag{11}$$

where \mathcal{M} is an approximation operator, \mathbf{t}_i and \mathbf{u}_i are the nodal values, and Φ_i is the shape function of the MLS approximation, which can be defined as [Li and Zhu (2009b); Li (2011a)]

$$\Phi_i(\mathbf{x}) = \begin{cases} \sum_{j=1}^{\beta} P_j(\mathbf{s}) \left[\mathbf{A}^{-1}(\mathbf{s}) \mathbf{B}(\mathbf{s}) \right]_{jk}, & i = I_k \in \wedge(\mathbf{x}), \\ 0, & i \notin \wedge(\mathbf{x}), \end{cases} \quad 1 \leq i \leq N \tag{12}$$

where \mathbf{s} is a local coordinate of the boundary point \mathbf{x} on Γ , $P_j(\mathbf{s})$ is a basis of order β consisting of monomials in \mathbf{s} , $\wedge(\mathbf{x}) = \{I_1, I_2, \dots, I_\kappa\}$ is the set of the global sequence numbers of boundary nodes that lie on the influence domain of \mathbf{x} , and the matrices $\mathbf{A}(\mathbf{s})$ and $\mathbf{B}(\mathbf{s})$ are defined by

$$\mathbf{A}(\mathbf{s}) = \sum_{j \in \wedge(\mathbf{x})} w_j(\mathbf{s}) \mathbf{P}(\mathbf{s}_j) \mathbf{P}^T(\mathbf{s}_j)$$

$$\mathbf{B}(\mathbf{s}) = [w_{I_1}(\mathbf{s}) \mathbf{P}(\mathbf{s}_{I_1}), w_{I_2}(\mathbf{s}) \mathbf{P}(\mathbf{s}_{I_2}), \dots, w_{I_\kappa}(\mathbf{s}) \mathbf{P}(\mathbf{s}_{I_\kappa})]$$

in which $\mathbf{P}(\mathbf{s}) = [P_1(\mathbf{s}), P_2(\mathbf{s}), \dots, P_\beta(\mathbf{s})]^T$, $w_j(\mathbf{s})$ is a weight function and \mathbf{s}_i is the curvilinear coordinate of \mathbf{x}_i .

In what follows, we assume that there exists a positive number $\gamma \geq 1/2$ such that the chosen weight function w_i is γ -times continuously differentiable and the boundary Γ is γ -times piecewise continuous. Then we can conclude that the MLS shape function Φ_i is γ -times continuously differentiable [Li and Zhu (2009b); Li (2011a)].

Let

$$\mathcal{H}_h = (\text{span} \{ \Phi_i, 1 \leq i \leq N \})^d \tag{13}$$

be the meshless space. Then the approximation of the variational problem (9) is

$$\begin{cases} \text{Find } (\mathbf{t}_h, \mathbf{u}_h) \in \mathcal{H}_h \text{ such that} \\ B((\mathbf{t}_h, \mathbf{u}_h), (\mathbf{t}', \mathbf{u}')) = L(\mathbf{t}', \mathbf{u}'), \quad \forall (\mathbf{t}', \mathbf{u}') \in \mathcal{H}_h \end{cases} \tag{14}$$

2.4 Discretization

Inserting Eqs. (10) and (11) into the variational problem (14), we get the following linear algebraic equations

$$\begin{pmatrix} \mathbf{V} & -\mathbf{A} \\ \mathbf{A}^T & \mathbf{D} \end{pmatrix} \begin{pmatrix} \mathbf{X}_t \\ \mathbf{X}_u \end{pmatrix} = \begin{pmatrix} \mathbf{F}_u \\ \mathbf{F}_t \end{pmatrix} \tag{15}$$

where \mathbf{X}_t is the vector of dN_u unknown nodal values $\mathbf{t}_i = (t_{1i}, t_{2i}, \dots, t_{di})^T$, \mathbf{X}_u is the vector of $d(N - N_u)$ unknown nodal values $\mathbf{u}_j = (u_{1j}, u_{2j}, \dots, u_{dj})^T$. The block matrices used in Eq. (15) are given by

$$\mathbf{V} = [\mathbf{V}_{pq}]_{p,q=1}^d, \quad \mathbf{V}_{pq}[k, i] = \int_{\Gamma_u} \int_{\Gamma_u} \Phi_i(\mathbf{y}) \mathbf{U}_{pq}(\mathbf{x}, \mathbf{y}) \Phi_k(\mathbf{x}) d\Gamma(\mathbf{y}) d\Gamma(\mathbf{x}) \tag{16}$$

$$\mathbf{A} = [\mathbf{A}_{pq}]_{p,q=1}^d, \quad \mathbf{A}_{pq}[k, j] = \int_{\Gamma_u} \int_{\Gamma_t} \Phi_j(\mathbf{y}) (\mathbf{T}_y)_{pq}(\mathbf{x}, \mathbf{y}) \Phi_k(\mathbf{x}) d\Gamma(\mathbf{y}) d\Gamma(\mathbf{x}) \tag{17}$$

$$\mathbf{D} = [\mathbf{D}_{pq}]_{p,q=1}^d, \quad \mathbf{D}_{pq}[m, j] = - \int_{\Gamma_t} \int_{\Gamma_t} \Phi_j(\mathbf{y}) \mathbf{S}_{pq}(\mathbf{x}, \mathbf{y}) \Phi_m(\mathbf{x}) d\Gamma(\mathbf{y}) d\Gamma(\mathbf{x}) \tag{18}$$

for all $k, i = 1, 2, \dots, N_u$ and $m, j = N_u + 1, N_u + 2, \dots, N$. The components of the right-hand side are given by

$$\mathbf{F}_u = [\mathbf{F}_u^p]_{p=1}^d, \quad \mathbf{F}_t = [\mathbf{F}_t^p]_{p=1}^d$$

with

$$\begin{aligned} \mathbf{F}_u^p[k] &= \frac{1}{2} \int_{\Gamma_u} \bar{u}_p(\mathbf{x}) \Phi_k(\mathbf{x}) d\Gamma(\mathbf{x}) + \int_{\Gamma_u} \int_{\Gamma_u} (\mathbf{T}_y)_{pq}(\mathbf{x}, \mathbf{y}) \bar{u}_p(\mathbf{y}) \Phi_k(\mathbf{x}) d\Gamma(\mathbf{y}) d\Gamma(\mathbf{x}) \\ &\quad - \int_{\Gamma_u} \int_{\Gamma_t} \mathbf{U}_{pq}(\mathbf{x}, \mathbf{y}) \bar{t}_p(\mathbf{y}) \Phi_k(\mathbf{x}) d\Gamma(\mathbf{y}) d\Gamma(\mathbf{x}) \end{aligned} \tag{19}$$

$$\begin{aligned} \mathbf{F}_t^p[m] &= \frac{1}{2} \int_{\Gamma_t} \bar{t}_p(\mathbf{x}) \Phi_m(\mathbf{x}) d\Gamma(\mathbf{x}) - \int_{\Gamma_t} \int_{\Gamma_t} (\mathbf{T}_x)_{pq}(\mathbf{x}, \mathbf{y}) \bar{t}_p(\mathbf{y}) \Phi_m(\mathbf{x}) d\Gamma(\mathbf{y}) d\Gamma(\mathbf{x}) \\ &\quad + \int_{\Gamma_t} \int_{\Gamma_u} \mathbf{S}_{pq}(\mathbf{x}, \mathbf{y}) \bar{u}_p(\mathbf{y}) \Phi_m(\mathbf{x}) d\Gamma(\mathbf{y}) d\Gamma(\mathbf{x}) \end{aligned} \tag{20}$$

for all $k = 1, 2, \dots, N_u$ and $m = N_u + 1, N_u + 2, \dots, N$.

From the symmetry of the kernel functions and the coerciveness of the operators \mathcal{V} and \mathcal{D} , we conclude that the block matrices \mathbf{V} and \mathbf{D} are symmetric and positive definite. Hence, the stiffness matrix in Eq. (15) is block skew-symmetric and positive definite. Then, one can solve Eq. (15) by a generalized Krylov subspace method such as the generalized minimum residual method (GMRES). Since this method can not utilize symmetry and positive definiteness simultaneously and sufficiently, equivalent system equations deserve to be established for the practical numerical implementation.

On the other hand, the symmetry and positive definiteness of the block matrix \mathbf{V} indicate that it is invertible. Thus, \mathbf{X}_t can be obtained from the first of Eq. (15) as

$$\mathbf{X}_t = \mathbf{V}^{-1} (\mathbf{F}_u + \mathbf{A}\mathbf{X}_u) \tag{21}$$

Then, inserting Eq. (21) into the second of Eq. (15) leads to the Schur complement system

$$(\mathbf{A}^T \mathbf{V}^{-1} \mathbf{A} + \mathbf{D}) \mathbf{X}_u = \mathbf{F}_t - \mathbf{A}^T \mathbf{V}^{-1} \mathbf{F}_u \tag{22}$$

The stiffness matrix in Eq. (22) is symmetric and positive definite, a property that enables the use of more efficient equation solvers and therefore leads to substantial reductions in solution time. Moreover, this property can be an added advantage in coupling the GBNM with the FEM. After solving the reduced system (22), the unknown vector \mathbf{X}_t can be computed in a postprocessing step via Eq. (21) from the now known vector \mathbf{X}_u . Finally, the yet unknowns \mathbf{t} on Γ_u and \mathbf{u} on Γ_t can be computed using Eqs. (10) and (11), respectively. Then, the approximate solution \mathbf{u}_h of the mixed elastic problem (1)-(3) can be computed from Eq. (4) as

$$\begin{aligned} \mathbf{u}_h(\mathbf{x}) = & \int_{\Gamma_u} \mathbf{U}(\mathbf{x}, \mathbf{y}) \mathbf{t}_h(\mathbf{y}) d\Gamma(\mathbf{y}) + \int_{\Gamma_t} \bar{\mathbf{t}}(\mathbf{y}) \mathbf{U}(\mathbf{x}, \mathbf{y}) d\Gamma(\mathbf{y}) \\ & - \int_{\Gamma_u} \bar{\mathbf{u}}(\mathbf{y}) \mathbf{T}_y(\mathbf{x}, \mathbf{y}) d\Gamma(\mathbf{y}) - \int_{\Gamma_t} \mathbf{u}_h(\mathbf{y}) \mathbf{T}_y(\mathbf{x}, \mathbf{y}) d\Gamma(\mathbf{y}), \quad \mathbf{x} \in \Omega \end{aligned} \tag{23}$$

Eqs. (16)-(20) and (23) have integrations over the boundary. As in many other meshless methods such as the EFG and the BNM, cells are used in this research to approximate the boundary and carry out numerical integration. It is worth mentioning that cells are used just for integration, and pose no restriction on shape or compatibility. In these integrations, if \mathbf{x} and \mathbf{y} belong to distinct cells, the integrands are regular and thus, the associated double integrals can be evaluated by usual Gaussian quadrature formulas. Otherwise, these double integrals are weakly singular, strongly singular or hypersingular. There have been various regularization procedures proposed in the past to handle various singular integrals. Li (2012)

developed a technique to tackle the weakly singular, strongly singular and hyper-singular integrations simultaneously. This technique is attractive and is used to carry out the singular integrations in this research.

2.5 Error analysis

In this subsection, we will estimate the error of using the GBNM for solving the mixed elastic problem (1)-(3). In what follows, by C we will denote a general constant which is independent of h and may have different values at different occurrences.

Lemma 2.1 (Li and Zhu (2009b); Li (2011a)) *Let \mathcal{M} be the MLS approximation operator and let \mathcal{P} be the L^2 -projection onto \mathcal{H}_h , then for any $\mathbf{v} \in \mathbf{H}^{m+1}(\Gamma)$ with $0 \leq m \leq \gamma$, we have*

$$\|\mathbf{v} - \mathcal{M}\mathbf{v}\|_{k,\Gamma} \leq Ch^{m+1-k} \|\mathbf{v}\|_{m+1,\Gamma}, \quad 0 \leq k \leq m$$

$$\|\mathbf{v} - \mathcal{P}\mathbf{v}\|_{k,\Gamma} \leq Ch^{m+1-k} \|\mathbf{v}\|_{m+1,\Gamma}, \quad -(\gamma+1) \leq k \leq m$$

Theorem 2.1 *Let (\mathbf{t}, \mathbf{u}) and $(\mathbf{t}_h, \mathbf{u}_h)$ be the solutions of variational problems (9) and (14), respectively. Then if $(\mathbf{t}, \mathbf{u}) \in \mathcal{H}^m(\Gamma)$, we have*

$$\|(\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h)\|_{-1/2,\Gamma} \leq Ch^{m+1/2} \|(\mathbf{t}, \mathbf{u})\|_{m,\Gamma}, \quad 1/2 \leq m \leq \gamma$$

Proof. Subtraction Eq. (14) from Eq. (9) leads to

$$B((\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h), (\mathbf{t}', \mathbf{u}')) = 0, \quad (\mathbf{t}', \mathbf{u}') \in \mathcal{H}_h \tag{24}$$

then using $(\mathcal{P}\mathbf{t}, \mathcal{M}\mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h) \in \mathcal{H}_h$ yields

$$B((\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h), (\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h)) = B((\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h), (\mathbf{t}, \mathbf{u}) - (\mathcal{P}\mathbf{t}, \mathcal{M}\mathbf{u})) \tag{25}$$

According to the coerciveness and continuity of the bilinear form $B(\cdot, \cdot)$, we have

$$B((\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h), (\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h)) \geq C \|(\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h)\|_{-1/2,\Gamma}^2 \tag{26}$$

$$\begin{aligned} & B((\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h), (\mathbf{t}, \mathbf{u}) - (\mathcal{P}\mathbf{t}, \mathcal{M}\mathbf{u})) \\ & \leq C \|(\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h)\|_{-1/2,\Gamma} \|(\mathbf{t}, \mathbf{u}) - (\mathcal{P}\mathbf{t}, \mathcal{M}\mathbf{u})\|_{-1/2,\Gamma} \end{aligned} \tag{27}$$

Gathering Eqs. (25)-(27) and using Lemma 2.1 we finally obtain

$$\begin{aligned} \|(\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h)\|_{-1/2, \Gamma} &\leq C \|(\mathbf{t}, \mathbf{u}) - (\mathcal{P}\mathbf{t}, \mathcal{M}\mathbf{u})\|_{-1/2, \Gamma} \\ &= C \left(\|\mathbf{t} - \mathcal{P}\mathbf{t}\|_{-1/2, \Gamma_u}^2 + \|\mathbf{u} - \mathcal{M}\mathbf{u}\|_{1/2, \Gamma_t}^2 \right)^{1/2} \\ &\leq C \left(h^{2m+1} \|\mathbf{t}\|_{m, \Gamma_u}^2 + h^{2m+1} \|\mathbf{u}\|_{m+1, \Gamma_t}^2 \right)^{1/2} \\ &= Ch^{m+1/2} \|(\mathbf{t}, \mathbf{u})\|_{m, \Gamma} \end{aligned}$$

which completes the proof.

Theorem 2.2 *Under the conditions of Theorem 2.1,*

$$\|(\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h)\|_{-k, \Gamma} \leq Ch^{m+k} \|(\mathbf{t}, \mathbf{u})\|_{m, \Gamma}, \quad 1/2 \leq k \leq m \leq \gamma$$

Proof. From the duality argument it follows that

$$\|(\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h)\|_{-k, \Gamma} \leq C \sup_{(\boldsymbol{\tau}, \boldsymbol{\mu}) \in \mathcal{H}^{k-1}(\Gamma)} \frac{|B((\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h), (\boldsymbol{\tau}, \boldsymbol{\mu}))|}{\|(\boldsymbol{\tau}, \boldsymbol{\mu})\|_{k-1, \Gamma}} \quad (28)$$

Since $\mathcal{P}(\boldsymbol{\tau}, \boldsymbol{\mu}) \in \mathcal{H}_h$, from Eq. (24) one gets

$$B((\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h), (\boldsymbol{\tau}, \boldsymbol{\mu})) = B((\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h), (\boldsymbol{\tau}, \boldsymbol{\mu}) - \mathcal{P}(\boldsymbol{\tau}, \boldsymbol{\mu}))$$

Then using the continuity of $B(\cdot, \cdot)$, Theorem 2.1 and Lemma 2.1 yields

$$\begin{aligned} B((\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h), (\boldsymbol{\tau}, \boldsymbol{\mu})) &\leq C \|(\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h)\|_{-1/2, \Gamma} \|(\boldsymbol{\tau}, \boldsymbol{\mu}) - \mathcal{P}(\boldsymbol{\tau}, \boldsymbol{\mu})\|_{-1/2, \Gamma} \\ &\leq Ch^{m+k} \|(\mathbf{t}, \mathbf{u})\|_{m, \Gamma} \|(\boldsymbol{\tau}, \boldsymbol{\mu})\|_{k-1, \Gamma} \end{aligned} \quad (29)$$

Finally, inserting Eq. (29) into Eq. (28) ends the proof.

Theorem 2.3 *Let \mathbf{u} and \mathbf{u}_h be defined by Eqs. (4) and (23), respectively. Assume that $(\mathbf{t}, \mathbf{u}) \in \mathcal{H}^m(\Gamma)$ with $1/2 \leq m \leq \gamma$. Then for any $\mathbf{x} \in \Omega$ with $\ell_{\mathbf{x}} = \min_{\mathbf{y} \in \Gamma} |\mathbf{x} - \mathbf{y}| \geq \delta > 0$, we have*

$$|\mathbf{u}(\mathbf{x}) - \mathbf{u}_h(\mathbf{x})| \leq C \sum_{j=1}^{\gamma+2} \ell_{\mathbf{x}}^{-j} h^{m+\gamma} \|(\mathbf{t}, \mathbf{u})\|_{m, \Gamma}$$

Proof. Subtraction Eq. (23) from Eq. (4) yields

$$\begin{aligned} &|\mathbf{u}(\mathbf{x}) - \mathbf{u}_h(\mathbf{x})| \\ &= \left| \int_{\Gamma_u} \mathbf{U}(\mathbf{x}, \mathbf{y}) (\mathbf{t}(\mathbf{y}) - \mathbf{t}_h(\mathbf{y})) \, d\Gamma(\mathbf{y}) - \int_{\Gamma_t} (\mathbf{u}(\mathbf{y}) - \mathbf{u}_h(\mathbf{y})) \mathbf{T}_{\mathbf{y}}(\mathbf{x}, \mathbf{y}) \, d\Gamma(\mathbf{y}) \right| \\ &\leq \|\mathbf{U}(\mathbf{x}, \mathbf{y})\|_{\gamma, \Gamma_u} \|\mathbf{t} - \mathbf{t}_h\|_{-\gamma, \Gamma_u} + \|\mathbf{T}_{\mathbf{y}}(\mathbf{x}, \mathbf{y})\|_{\gamma-1, \Gamma_t} \|\mathbf{u} - \mathbf{u}_h\|_{1-\gamma, \Gamma_t} \end{aligned} \quad (30)$$

Using $\ell_{\mathbf{x}} \geq \delta > 0$, we have

$$\|\mathbf{U}(\mathbf{x}, \mathbf{y})\|_{\gamma, \Gamma_u} \leq C \sum_{j=1}^{\gamma+2} \ell_{\mathbf{x}}^{-j}, \quad \left\| \partial^\lambda T_{\mathbf{y}}(\mathbf{x}, \mathbf{y}) \right\|_{\gamma-1, \Gamma_t} \leq C \sum_{j=1}^{\gamma+2} \ell_{\mathbf{x}}^{-j} \quad (31)$$

Thus, substituting Eq. (31) into Eq. (30) and invoking Theorem 2.2 end the proof. Theorem 2.3 indicates that the approximate solution obtained by the meshless GB-NM converges to the analytical solution of the elastic problem (1)-(3). The same type of estimate can be obtained for the stress tensor σ . More specifically, we have

Theorem 2.4 *Let σ be the exact stress solution of the elastic problem (1)-(3) and let σ_h be the corresponding GBNM solution, then under conditions of Theorem 2.3,*

$$|\sigma(\mathbf{x}) - \sigma_h(\mathbf{x})| \leq C \sum_{j=1}^{\gamma+2} \ell_{\mathbf{x}}^{-j-1} h^{m+\gamma} \|(\mathbf{t}, \mathbf{u})\|_{m, \Gamma}$$

Theorems 2.3 and 2.4 indicate that the errors of stress and displacement in the meshless GBNM are all of the same convergence rate.

Furthermore, the convergence can be established in energy norms.

Theorem 2.5 *Let \mathbf{u} and \mathbf{u}_h be defined by Eqs. (4) and (23), respectively. If $(\mathbf{t}, \mathbf{u}) \in \mathcal{H}^m(\Gamma)$ with $1/2 \leq m \leq \gamma$, then*

$$\|\mathbf{u} - \mathbf{u}_h\|_{1, \Omega} \leq Ch^{m+1/2} \|(\mathbf{t}, \mathbf{u})\|_{m, \Gamma}$$

Proof. Since Eq. (4) defines an isomorphism from $\mathcal{H}^{-1/2}(\Gamma)$ onto $\mathbf{H}^1(\Omega)$, we have

$$\|\mathbf{u} - \mathbf{u}_h\|_{1, \Omega} \leq \|(\mathbf{t}, \mathbf{u}) - (\mathbf{t}_h, \mathbf{u}_h)\|_{-1/2, \Gamma}$$

The proof is completed via invoking Theorem 2.1.

3 Coupling of the GBNM and the FEM

3.1 Coupled formulation

As shown in Fig. 1, a bounded or unbounded problem domain Ω is decomposed into two disjoint sub-domains, Ω_G and Ω_F , with the GBNM-FEM coupling interface Γ_I . The GBNM is used in Ω_G and the FEM is used in Ω_F . Without loss of generality, it is assumed that both displacement and traction boundary conditions are given on Γ_G and Γ_F . The boundaries of Ω_G and Ω_F are denoted as $\Gamma_G = \Gamma_{uG} \cup \Gamma_{tG} \cup \Gamma_I$

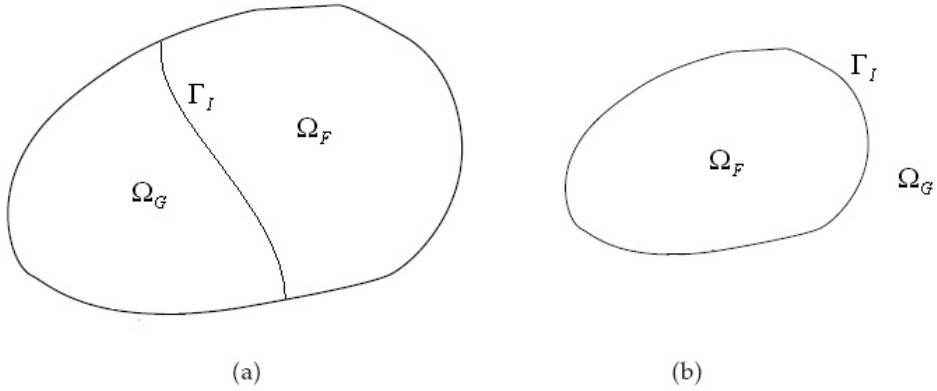


Figure 1: The coupling domain of the GBNM and the FEM. (a) the problem domain Ω is bounded and (b) the problem domain Ω is unbounded.

and $\Gamma_F = \Gamma_{uF} \cup \Gamma_{tF} \cup \Gamma_I$, respectively. Note that if the problem domain Ω is unbounded, as shown in Fig. 1(b), both the displacement boundary Γ_u and traction boundary Γ_t are empty. We consider the model boundary value problems as

$$\nabla \cdot \boldsymbol{\sigma} = \mathbf{0}, \quad \text{in } \Omega_G \tag{32}$$

$$\nabla \cdot \boldsymbol{\sigma} = \mathbf{b}, \quad \text{in } \Omega_F \tag{33}$$

$$u = \bar{\mathbf{u}}, \quad \text{on } \Gamma_u = \Gamma_{uG} \cup \Gamma_{uF} \tag{34}$$

$$\mathbf{t} = \boldsymbol{\sigma} \cdot \mathbf{n} = \bar{\mathbf{t}}, \quad \text{on } \Gamma_t = \Gamma_{tG} \cup \Gamma_{tF} \tag{35}$$

where \mathbf{b} is the body force, and $\boldsymbol{\sigma}$ is the stress tensor given by $\boldsymbol{\sigma} = \mathcal{A}(\mathbf{u})$. Here, the operator \mathcal{A} is uniformly monotone and Lipschitz continuous. In Ω_G , this operator is linear as $\mathcal{A}(\mathbf{u}) = \lambda \text{tr} \boldsymbol{\varepsilon}(\mathbf{u}) \mathbf{I} + 2\mu \boldsymbol{\varepsilon}(\mathbf{u})$, where \mathbf{I} is the unit matrix, tr is the trace operator and $\boldsymbol{\varepsilon}(\mathbf{u}) = \frac{1}{2} (\nabla \mathbf{u} + (\nabla \mathbf{u})^T)$ is the strain. In Ω_F , the operator \mathcal{A} may be linear or nonlinear. Examples of nonlinear cases can be found in [Stephan (2004)]. In the GBNM sub-domain Ω_G , applying Eq. (5) for $\mathbf{x} \in \Gamma_{uG}$ and Eq. (6) for $\mathbf{x} \in \Gamma_{tG}$, according to boundary conditions (34) and (35), we gain

$$\mathcal{H}_{\Gamma_{uG}} \mathbf{t} + \mathcal{H}_{\Gamma_t} \mathbf{t} - \mathcal{K}_{\Gamma_{tG}} \mathbf{u} - \mathcal{K}_{\Gamma_t} \mathbf{u} = \frac{1}{2} \bar{\mathbf{u}} + \mathcal{K}_{\Gamma_{uG}} \bar{\mathbf{u}} - \mathcal{H}_{\Gamma_{tG}} \bar{\mathbf{t}} \tag{36}$$

$$\mathcal{K}'_{\Gamma_{uG}} \mathbf{t} + \mathcal{K}'_{\Gamma_t} \mathbf{t} + \mathcal{D}_{\Gamma_{tG}} \mathbf{u} + \mathcal{D}_{\Gamma_t} \mathbf{u} = \frac{1}{2} \bar{\mathbf{t}} - \mathcal{K}'_{\Gamma_{tG}} \bar{\mathbf{t}} - \mathcal{D}_{\Gamma_{uG}} \bar{\mathbf{u}} \tag{37}$$

Then, as in Sections 2.2 and 2.4, evaluating Eqs. (36) and (37) in the sense of Galerkin yields

$$\begin{pmatrix} \mathbf{V}_{ul} & \mathbf{V}_{uu} & -\mathbf{A}_{ul} & -\mathbf{A}_{ut} \\ \mathbf{A}_{tl}^T & \mathbf{A}_{tu}^T & \mathbf{D}_{tl} & \mathbf{D}_{tt} \end{pmatrix} \begin{pmatrix} \mathbf{T}^l \\ \mathbf{T}^u \\ \mathbf{U}^l \\ \mathbf{U}^t \end{pmatrix} = \begin{pmatrix} \mathbf{f}_u^u \\ \mathbf{f}_t^l \end{pmatrix} \quad (38)$$

where \mathbf{T}^l and \mathbf{T}^u are the traction vector at the nodes on Γ_l and Γ_{uG} , respectively; \mathbf{U}^l and \mathbf{U}^t are the displacement vector at the nodes on Γ_l and Γ_{tG} , respectively. In the matrices \mathbf{V}_{ij} , \mathbf{A}_{ij} and \mathbf{D}_{ij} , the first index denotes the position of the source point \mathbf{x} and the second index stands for the position of the field point \mathbf{y} .

Since both the displacements and tractions are unknown on the interface Γ_l , applying Eqs. (5) and (6) for $\mathbf{x} \in \Gamma_l$ we gain

$$-\frac{1}{2}\mathbf{u} + \mathcal{V}_{\Gamma_{uG}}\mathbf{t} + \mathcal{V}_{\Gamma_l}\mathbf{t} - \mathcal{K}_{\Gamma_{tG}}\mathbf{u} - \mathcal{K}_{\Gamma_l}\mathbf{u} = \mathcal{K}_{\Gamma_{uG}}\bar{\mathbf{u}} - \mathcal{V}_{\Gamma_{tG}}\bar{\mathbf{t}} \quad (39)$$

$$-\frac{1}{2}\mathbf{t} + \mathcal{K}'_{\Gamma_{uG}}\mathbf{t} + \mathcal{K}'_{\Gamma_l}\mathbf{t} + \mathcal{D}_{\Gamma_{tG}}\mathbf{u} + \mathcal{D}_{\Gamma_l}\mathbf{u} = -\mathcal{K}'_{\Gamma_{tG}}\bar{\mathbf{t}} - \mathcal{D}_{\Gamma_{uG}}\bar{\mathbf{u}} \quad (40)$$

Then evaluating Eqs. (39) and (40) in the sense of Galerkin yields

$$\begin{pmatrix} \mathbf{V}_{ll} & \mathbf{V}_{lu} & -\frac{1}{2}\Phi - \mathbf{A}_{ll} & -\mathbf{A}_{lt} \\ -\frac{1}{2}\Phi + \mathbf{A}_{ll}^T & \mathbf{K}_{lu}^T & \mathbf{D}_{ll} & \mathbf{D}_{lt} \end{pmatrix} \begin{pmatrix} \mathbf{T}^l \\ \mathbf{T}^u \\ \mathbf{U}^l \\ \mathbf{U}^t \end{pmatrix} = \begin{pmatrix} \mathbf{f}_u^l \\ \mathbf{f}_t^l \end{pmatrix} \quad (41)$$

where $\Phi = [\Phi^p]_{p=1}^d$ contains the integrals accompanying the free terms as

$$\Phi^p[k, i] = \int_{\Gamma_l} \Phi_i(\mathbf{x}) \Phi_k(\mathbf{x}) d\Gamma(\mathbf{x})$$

According to the scheme used for evaluating an equivalent nodal force [Beer (2001)], we can define a vector of equivalent nodal forces on the interface Γ_l as

$$\mathbf{F}_G^l = \Phi \mathbf{T}^l$$

Then gathering Eqs. (38) and (41) provides

$$\begin{pmatrix} \mathbf{V}_{ll} & \mathbf{V}_{lu} & -\frac{1}{2}\Phi - \mathbf{A}_{ll} & -\mathbf{A}_{lt} \\ \mathbf{V}_{ul} & \mathbf{V}_{uu} & -\mathbf{A}_{ul} & -\mathbf{A}_{ut} \\ \frac{1}{2}\Phi + \mathbf{A}_{ll}^T & \mathbf{A}_{lu}^T & \mathbf{D}_{ll} & \mathbf{D}_{lt} \\ \mathbf{A}_{ll}^T & \mathbf{A}_{tu}^T & \mathbf{D}_{tl} & \mathbf{D}_{tt} \end{pmatrix} \begin{pmatrix} \mathbf{T}^l \\ \mathbf{T}^u \\ \mathbf{U}^l \\ \mathbf{U}^t \end{pmatrix} = \begin{pmatrix} \mathbf{f}_u^l \\ \mathbf{f}_u^u \\ \mathbf{f}_t^l + \mathbf{F}_G^l \\ \mathbf{f}_t^t \end{pmatrix} \quad (42)$$

As in Section 2.4, Eq. (42) can be transformed to the following Schur complement system by eliminating the traction vectors,

$$\begin{pmatrix} \mathbf{K}_G^{II} & \mathbf{K}_G^{IG} \\ \mathbf{K}_G^{GI} & \mathbf{K}_G^{GG} \end{pmatrix} \begin{pmatrix} \mathbf{U}^I \\ \mathbf{U}^t \end{pmatrix} = \begin{pmatrix} \mathbf{f}_G^I + \mathbf{F}_G^I \\ \mathbf{f}_G^G \end{pmatrix} \quad (43)$$

Eq. (43) correlates the nodal displacements with nodal forces.

On the other hand, the FEM sub-domain Ω_F yields the following system of equation by the well-known finite element implementation,

$$\mathbf{K}_F \mathbf{U} = \mathbf{F}_F$$

where \mathbf{K}_F is the domain stiffness matrix, \mathbf{U} and \mathbf{F}_F are the nodal displacements and nodal forces, respectively. This equation can be split into two parts corresponding to a region containing the interfacial degrees of freedom and a region containing the non-interfacial degrees of freedom,

$$\begin{pmatrix} \mathbf{K}_F^{FF} & \mathbf{K}_F^{FI} \\ \mathbf{K}_F^{IF} & \mathbf{K}_F^{II} \end{pmatrix} \begin{pmatrix} \mathbf{U}^F \\ \mathbf{U}^I \end{pmatrix} = \begin{pmatrix} \mathbf{F}_F^F \\ \mathbf{F}_F^I \end{pmatrix} \quad (44)$$

where \mathbf{K}_F^{II} is the stiffness matrix corresponding to the interfacial region displacements \mathbf{U}^I , \mathbf{K}_F^{IF} is the stiffness matrix corresponding to the non-interfacial region \mathbf{U}^F , \mathbf{F}_F^I is the load vector for the interface region alone that is obtained using the FEM, and \mathbf{F}_F^F is the load vector for the non-interface degrees of freedom.

Moreover, the compatibility and equilibrium conditions on the coupling interface Γ_I must be satisfied. Therefore, the displacements on Γ_I for Ω_F and Ω_G should be equal, i.e.

$$\mathbf{u}_I^F = \mathbf{u}_I^G, \quad \text{on } \Gamma_I \quad (45)$$

where \mathbf{u}_I^F and \mathbf{u}_I^G are the displacement on Γ_I that is obtained using the FEM and the GBNM, respectively. Besides, the summation of nodal forces on Γ_I for Ω_F and Ω_G should be zero, i.e.

$$\mathbf{F}_G^I + \mathbf{F}_F^I = \mathbf{0}, \quad \text{on } \Gamma_I \quad (46)$$

Finally, combining Eqs. (43) and (44), in view of Eqs. (45) and (46), we obtain the coupled equations as follows:

$$\begin{pmatrix} \mathbf{K}_F^{FF} & \mathbf{K}_F^{FI} & \mathbf{0} \\ \mathbf{K}_F^{IF} & \mathbf{K}_F^{II} + \mathbf{K}_G^{II} & \mathbf{K}_G^{IG} \\ \mathbf{0} & \mathbf{K}_G^{GI} & \mathbf{K}_G^{GG} \end{pmatrix} \begin{pmatrix} \mathbf{U}^F \\ \mathbf{U}^I \\ \mathbf{U}^t \end{pmatrix} = \begin{pmatrix} \mathbf{F}_F^F \\ \mathbf{f}_G^I \\ \mathbf{f}_G^G \end{pmatrix} \quad (47)$$

Since the FEM matrix in Eq. (44) is obtained from the usual energy based FEM approaches, the resultant matrix is symmetric and positive definite. The sum of the symmetric and positive definite matrices continues to be symmetric and positive definite. Thus the resulting coupling matrix presented in Eq. (47) is symmetric and positive definite.

3.2 Error analysis

In this subsection, we will estimate the error of using the symmetric coupled GBNM-FEM for solving the mixed elastic problem (32)-(35). In what follows, let \mathbf{u} be the exact displacement solution of the elastic problem and let \mathbf{u}_h be the approximated displacement obtained by the GBNM, the FEM or the coupled GBNM-FEM.

In the GBNM sub-domain Ω_G , using Theorem 2.5 we have

Theorem 3.1 *Let h_G be the spacing of boundary nodes in the GBNM sub-domain Ω_G . Then*

$$\|\mathbf{u} - \mathbf{u}_h\|_{1,\Omega_G} \leq Ch_G^{m+1/2}$$

In the FEM sub-domain Ω_F , let us use a regular partition of the interior domain Ω_F and let h_F denote the maximum of the longest element sides. On these elements, let \mathbf{H}_F denote a finite dimensional subspaces of $\mathbf{H}^1(\Omega_F)$, satisfying

$$\inf_{\mathbf{v}_h \in \mathbf{H}_F} \|\mathbf{u} - \mathbf{v}_h\|_{1,\Omega_F} \leq Ch_F^\alpha \|\mathbf{u}\|_{\alpha+1,\Omega_F}, \quad \forall \mathbf{u} \in \mathbf{H}^1(\Omega_F)$$

Then,

Theorem 3.2 *The error estimation in the FEM sub-domain Ω_F is*

$$\|\mathbf{u} - \mathbf{u}_h\|_{1,\Omega_F} \leq Ch_F^\alpha$$

As stated in the previous section, when the coupled GBNM-FEM is used, the problem domain Ω is decomposed into two disjoint sub-domains, Ω_G and Ω_F . Thus,

$$\|\mathbf{u} - \mathbf{u}_h\|_{1,\Omega} = \|\mathbf{u} - \mathbf{u}_h\|_{1,\Omega_F} + \|\mathbf{u} - \mathbf{u}_h\|_{1,\Omega_G}$$

As a consequence, the error estimation of the coupled GBNM-FEM can be established by combining Theorems 3.1 and 3.2.

Theorem 3.3 *The error estimation in the problem domain Ω is*

$$\|\mathbf{u} - \mathbf{u}_h\|_{1,\Omega} \leq C \left(h_G^{m+1/2} + h_F^\alpha \right)$$

4 Numerical examples

4.1 Examples of the GBNM

Two examples are selected to demonstrate the applicability of the GBNM for elasticity problems.

The first example that is considered is a three-dimensional problem in a cubic domain. The cube is bounded by the planes $x_1 = \pm 1$, $x_2 = \pm 1$ and $x_3 = \pm 1$. The following analytical solution is used,

$$u_1 = -2x_1^2 + 3x_2^2 + 3x_3^2, \quad u_2 = 3x_1^2 - 2x_2^2 + 3x_3^2, \quad u_3 = 3x_1^2 + 3x_2^2 - 2x_3^2 \quad (48)$$

Displacements are imposed on faces $x_3 = \pm 1$ and boundary tractions on all other faces. The material constants that are used in our analysis are Young’s modulus $E = 1.0$ and Poisson’s ration $\nu = 0.25$.

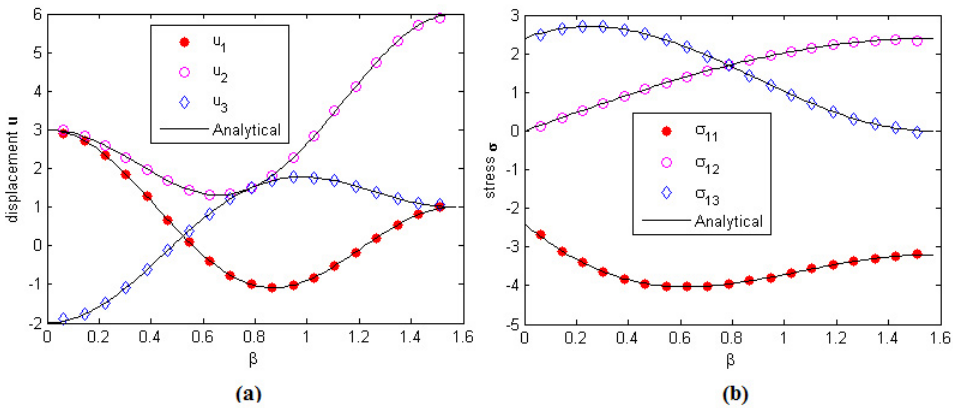


Figure 2: Results of (a) displacement \mathbf{u} and (b) stress σ for the cubic problem.

Fig. 2 shows a comparison between the numerical results with the analytical solutions for displacement \mathbf{u} and stress σ along the arc given by the formulas $x_1 = \sin \beta$, $x_2 = 0$, $x_3 = \cos(2\beta)$, $\beta \in [0, 0.5\pi]$. In this analysis, the cubic surface is discretized using 48 distributed nodes. It is clearly shown that the numerical results agree very well with the analytical ones.

To investigate the convergence of the present method, three different nodal arrangements of 48, 192 and 768 boundary nodes have been used. Fig. 3 shows the log-log plot of errors with respect to the nodal spacing. As we expected, the numerical results from the proposed meshless method gradually converge to the analytical values along with the decrease of the nodal spacing.

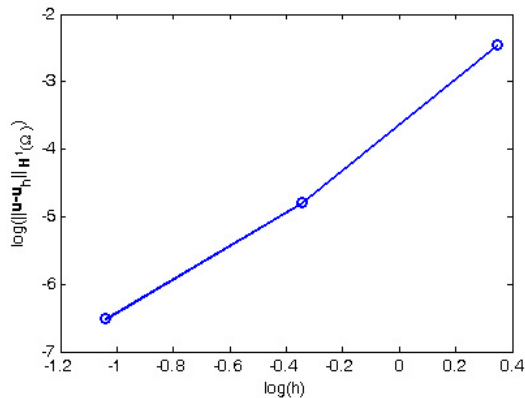


Figure 3: Convergence of the GBNM.

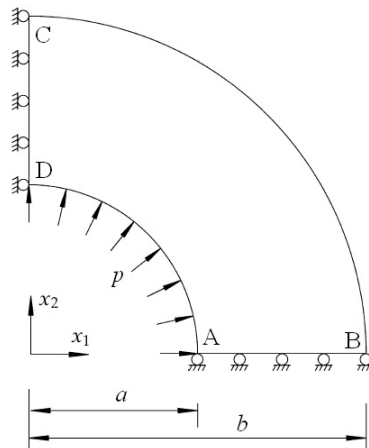


Figure 4: A cylindrical tube subjected to uniform internal pressure.

The second example to be considered is a cylindrical tube subjected to uniform internal pressure. Due to symmetry, only the upper right quadrant of the structure is modeled as shown in Fig. 4. The plane stress case is considered, and the parameters are chosen as Young's modulus $E = 10$, Poisson's ratio $\nu = 0.25$ and internal pressure $p = 1.0$. Besides, the geometry is chosen as $a = 1$ and $b = 2$. In the polar coordinate system (r, θ) , the analytical displacements are [Timoshenko and Goodier (1970)]

$$u_r = \frac{a^2 p r}{E(b^2 - a^2)} \left[1 - \nu + \frac{b^2}{r^2} (1 + \nu) \right], \quad u_\theta = 0$$

and the corresponding stresses are

$$\sigma_r = \frac{a^2 p}{b^2 - a^2} \left(1 - \frac{b^2}{r^2} \right), \quad \sigma_\theta = \frac{a^2 p}{b^2 - a^2} \left(1 + \frac{b^2}{r^2} \right)$$

The numerical results by the GBNM are plotted in Fig. 5. We again verify these results with the available analytical solution. In this analysis, the boundary is discretized by 60 boundary nodes (12 nodes on AB, CD and AD, and 24 nodes on BC). As expected, these numerical results agree well with the analytical values.

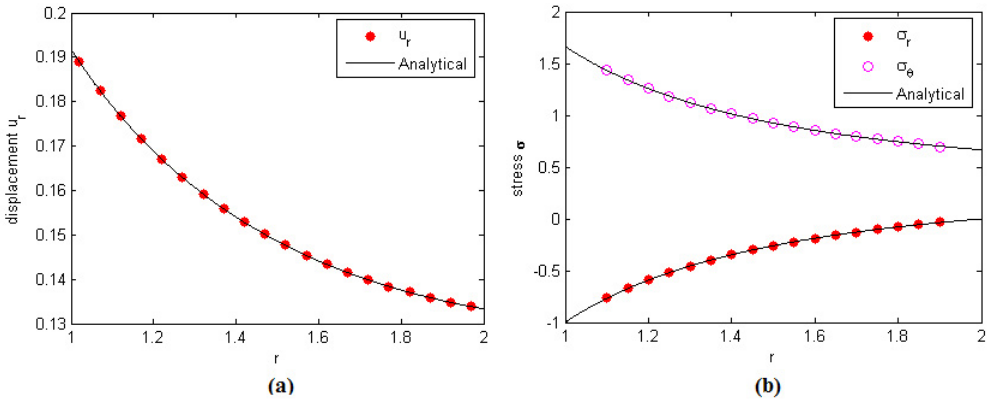


Figure 5: Results of (a) radial displacement u_r and (b) stress σ along the radius.

4.2 Examples of the coupled GBNM-FEM

In this subsection, we will present two numerical examples to show the accuracy and efficiency of the coupled GBNM-FEM of this paper.

Consider a beam subjected to a parabolic traction at the free end as shown in Fig. 6. The beam is of length L and height H , and has a unit thickness. The beam is assumed to be in a state of plane stress. The analytical solution for this problem is [Timoshenko and Goodier (1970)]

$$u_1 = \frac{Px_2}{6EI} \left[(6L - 3x_1)x_1 + (2 + \nu) \left(x_2^2 - \frac{H^2}{4} \right) \right]$$

$$u_2 = -\frac{P}{6EI} \left[3\nu x_2^2 (L - x_1) + (4 + 5\nu) \frac{H^2 x_1}{4} + (3L - x_1)x_1^2 \right]$$

$$\sigma_{11} = \frac{P(L - x_1)x_2}{I}, \quad \sigma_{22} = 0, \quad \sigma_{12} = -\frac{P}{2I} \left(\frac{H^2}{4} - x_2^2 \right)$$

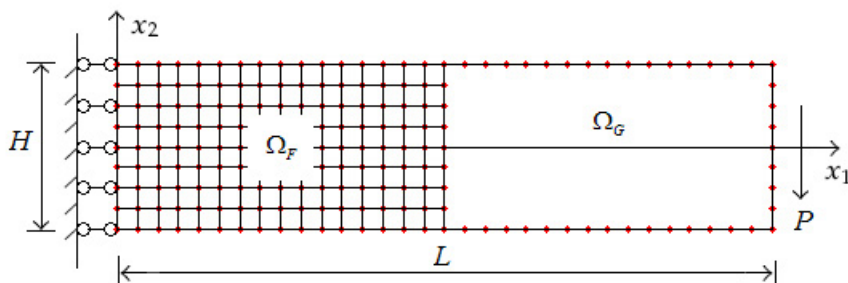


Figure 6: A beam and its computational model.

where $I = H^3/12$ is the moment of inertia of the beam.

The beam is separated into two parts. The GBNM is used in the right part and the FEM is used in the left part. The parameters are taken as $E = 3.0 \times 10^7 \text{kPa}$, $\nu = 0.3$, $L = 48\text{m}$, $H = 12\text{m}$ and $P = 1000\text{kN}$ in the computation.

The numerical results, which are furnished by the coupled method, are shown in Fig. 7 together with the analytical solutions. In this analysis, 48 boundary nodes are used in the GBNM region, and 128 quadrangular FEM elements are used in the FEM region. From this figure, we can find that the numerical solutions are in excellent agreement with the analytical solutions.

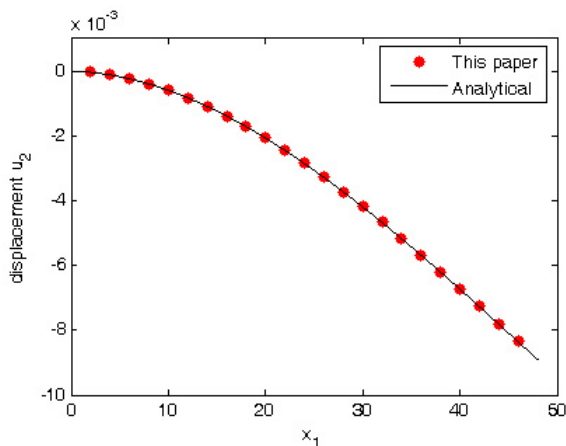


Figure 7: Results of displacement u_2 at $x_2 = 0$.

The convergence is presented in Fig. 8, where h is equivalent to the maximum element size in the FEM in this case. We can find that the greater precision of the

solution will be obtained when more nodes are selected.

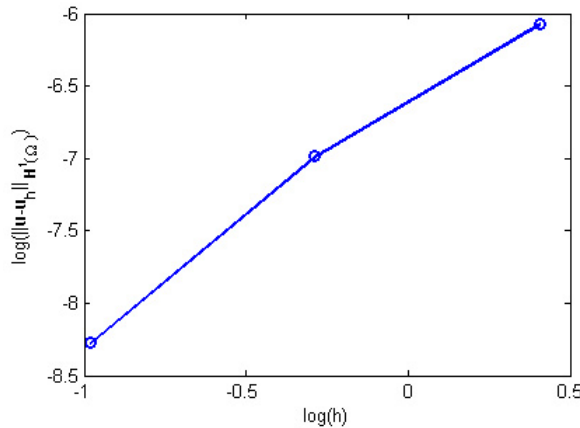


Figure 8: Convergence of the coupled GBNM-FEM.

Next, we consider a semi-infinite soil-structure interaction problem. As shown in Fig. 9 (a), the FEM is used in the structure region Ω_F , and the GBNM is used in the infinite soil foundation region Ω_G . As in [Brebbia and Georgion (1979); Qin and Cheng (2008)], the infinite foundation can be treated by truncating the semi-infinite plane at a finite distance from the structure. The computational model is plotted in Fig. 9: 36 boundary nodes are used in the GBNM region, and 48 triangular elements are used in the FEM region.

Consider five concentrated vertical loads on the top of the structure. Table 1 gives the vertical displacement on the top of the structure. The results obtained using the FEM [Brebbia and Georgion (1979)], the coupled BEM-FEM [Brebbia and Georgion (1979)] and the coupled RKPBEFM-FEM [Qin and Cheng (2008)] are also given in the table for comparison. Although no analytical solutions exist for such a complex problem, the solutions of the presented coupled method are in good agreement with the results of other methods.

5 Conclusions

The meshless GBNM is developed in this paper for the numerical solution of mixed elasticity problems in two and three dimensions. In this method, an equivalent variational form of BIEs is used, thus boundary conditions are applied directly and easily. Another prominent feature of the present approach is that the resulting system matrix is not only symmetric but also positive definite. This paper also examines an

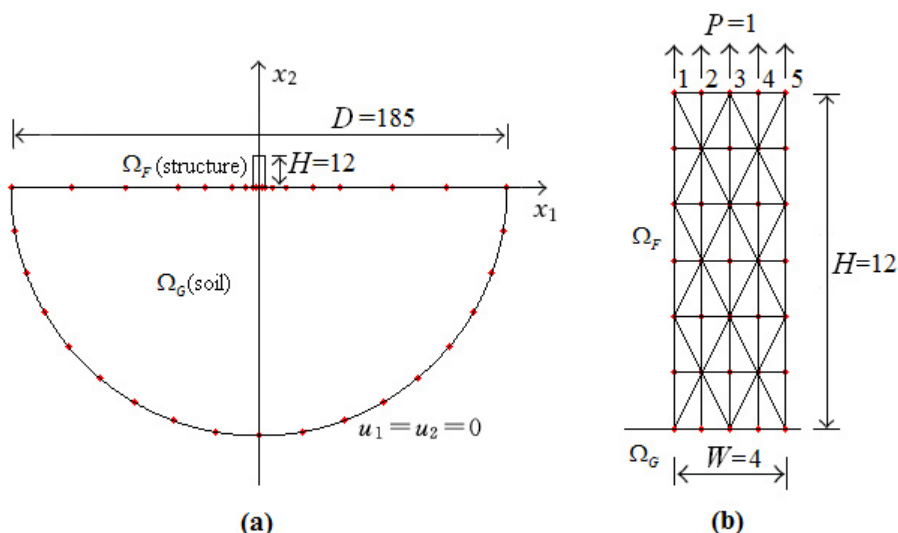


Figure 9: Schematic diagram for the problem of a structure standing on a semi-infinite foundation. (a) The coupled GBNM/FEM model. (b) Meshes and loads on the FEM region.

Table 1: Vertical displacement ($\times 10^{-4}$) along top of the structure

Node No.	FEM	BEM-FEM	RKPBEFM-FEM	GBNM-FEM
1	1.41	1.40	1.42	1.43
2	1.34	1.33	1.33	1.33
3	1.32	1.32	1.31	1.32
4	1.34	1.33	1.33	1.33
5	1.41	1.40	1.42	1.43

efficient symmetric coupling of the GBNM with the FEM. In the coupled method, the resulting coupling matrix is symmetric and positive definite. Theoretical error estimates of the GBNM and the coupled GBNM-FEM are established. From the error analysis, it is shown that the error bound of the numerical solution is directly related to the nodal spacing. Some numerical examples have been given and the numerical results are accurate and are in agreement with the theoretical analysis.

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