

A Piecewise Linear Isotropic-Kinematic Hardening Model with Semi-Implicit Rules for Cyclic Loading and Its Parameter Identification

M. Ohsaki¹, T. Miyamura² and J. Y. Zhang³

Abstract: A simple constitutive model, called semi-implicit model, for cyclic loading is proposed for steel materials used for structures such as building frames in civil engineering. The constitutive model is implemented in the E-Simulator, which is a software package for large-scale seismic response analysis. The constitutive relation is defined in an algorithmic manner based on the piecewise linear combined isotropic-kinematic hardening. Different rules are used for the first and subsequent loading states to incorporate characteristics such as yield plateau and Bauschinger effect of rolled mild steel materials. An optimization method is also presented for parameter identification from the results of cyclic and monotonic loading tests. Therefore, the proposed model is readily applicable to practical elastoplastic analysis of building frames. Accuracy of the model is demonstrated in an example of a cantilever subjected to various types of cyclic loading.

Keywords: Combined isotropic-kinematic hardening, Piecewise linear hardening, Steel material, Parameter optimization, Cyclic load

1 Introduction

In the design process of building frames, accurate estimation of elastoplastic responses is very important, because such structures are supposed to dissipate seismic energy through plastic deformation. Elastoplastic responses of building frames are usually simulated using macro models such as plastic hinges and fiber elements. However, owing to recent progress of computer technology and due to increasing

¹ Department of Architecture, Hiroshima University, 1-4-1 Kagamiyama, Higashi-Hiroshima 739-8527, Japan; Currently, Department of Architecture and Architectural Engineering, Kyoto University, Kyoto-Daigaku Katsura, Nishikyo, Kyoto 615-8540, Japan, ohsaki@archi.kyoto-u.ac.jp

² Department of Computer Science, Nihon University, 1 Nakagawara, Tokusada, Tamura, Koriyama 963-8642, Japan, miyamura@cs.ce.nihon-u.ac.jp

³ Department of Architecture and Urban Design, Nagoya City University, 2-1-10 Kita-Chigusa, Chigusa, Nagoya 464-0083, Japan, zhang@sda.nagoya-cu.ac.jp

demands for accurate estimation of seismic responses, finite element analysis using solid elements has been extensively investigated [Ohsaki, Miyamura, Kohiyama, Hori, Noguchi, Akiba, Kajiwara, Ine (2009)].

A software package called E-Simulator is under development at the National Research Institute of Earth Science and Disaster Prevention (NIED), Japan. E-simulator is a parallel finite element analysis software package for precisely simulating collapse behaviors of building and civil structures [Miyamura, Yamashita, Akiba, Ohsaki (2015); Miyamura, Ohsaki, Kohiyama, Isobe, Onda, Akiba, Hori, Kajiwara, Ine (2011)], and it is based on a commercial software package called ADVENTUREcluster [Allied Engineering (2015)]. We developed a constitutive model for large-scale analysis of steel building frames, and its prototype has been successfully used in a detailed and large-scale simulation of a steel frame discretized into solid elements of about 19 million degrees of freedom [Miyamura, Yamashita, Akiba, Ohsaki (2015)]. In this paper, we present the details of the constitutive model.

In the field of mechanical engineering and material science, various constitutive models, including Armstrong-Frederic (AF) model [Armstrong and Frederick (1966)], multi-layer model [Besseling (1958)], and bounding surface model [Dafalias and Popov (1975)], have been developed for simulating cyclic elastoplastic behavior of various types of steel materials. By contrast, in the field of civil engineering, rolled mild steel materials are widely used. One of the important and distinct properties of these materials is the existence of a sharp yield plateau, which leads to different characteristics between the first and subsequent loadings [Ucak and Tsopelas (2011); Rodzik (1999)]. Yoshida (2000) developed a viscoplastic model for simulation of cyclic responses of a material exhibiting yield plateau. Yoshida and Uemori (2002) and Shen, Tanaka, Mizuno, Usami (1992) presented constitutive models using combined isotropic-kinematic hardening and bounding surfaces for simulation of Bauschinger effect. However, they did not explicitly distinguish the first and subsequent loadings. Recently, Ucak and Tsopelas (2011) developed a cyclic model that can simulate yield plateau using the AF model and a bounding surface.

Ohno and Wang (1993) proposed an evolution rule of backstress for accurate simulation of dynamic recovery and ratcheting behavior, and derived the consistent tangent operator for implementation to a finite element analysis code [Kobayashi and Ohno (2002)]. In contrast to complex evolution rules, for which derivation of consistent tangent operators is complicated [Mühlich and Brocks (2003); Diegele, Jansohn, Tsakmakis (2000)], linear isotropic-kinematic hardening model has a very stable algorithm of consistent tangent based on analytical solution of consistency condition [Kossa and Szabó (2009); Auricchio and da Veiga (2003); Romashchenko, Lepikhin, Ivashchenko (1999)]. Therefore, our constitutive model based on linear

hardening can be applied to a large-scale simulation of steel structures.

Most of the existing studies on constitutive models of steel materials are intended to develop *explicit* evolution rules of stresses and strains in reference to the physical behavior of the material idealized as a continuum. There is another direction of research for developing an *implicit* constitutive relation, which is not defined based on an explicit differential equations of stresses, strains, and internal variables, but instead, defined in a heuristic and algorithmic manner. For example, Furukawa and Yagawa (1998) used a neural network for modeling implicit constitutive relation of a visco-elastoplastic material subjected to uniaxial cyclic loading. However, a completely implicit relation between the stress rates and strain rates cannot be obtained generally for multiaxial loading, because such relation depends on the loading history. Note that the terms explicit and implicit are not related to those for solution algorithm of differential equations as discussed in [Ellsiepen and Hartmann (2001)]. In the field of civil engineering, there are many macro (*phenomenological* or *physical*) models such as plastic hinges and fiber models for simulating cyclic plastic behavior of beams, columns, and braces [Uriz, Filippou, Mahin (2001); Kim and Lee (2001); Ishizawa and Iura (2006)]. Composite beam elements have also been developed [Bradford and Pi (2012)].

In this paper, an intermediate implicit and explicit model, called *semi-implicit model*, is presented for rolled mild steel materials. The purpose of this paper is summarized as follows:

1. Present a simple constitutive model for cyclic behavior of steel material used for civil engineering exhibiting yield plateau and Bauschinger effect.
2. Present an algorithm that can be used for finite element analysis with solid elements.
3. Present an optimization algorithm for identification of material parameters.

The constitutive relation is defined in an algorithmic manner based on the piecewise linear combined isotropic-kinematic hardening. Different rules are used for the first and subsequent loading states. We also present an efficient optimization method, based on a heuristic approach called tabu search (TS) [Glover (1989); Ohsaki (2010)], for parameter identification from the material test results. The accuracy of the constitutive model is discussed in examples of a cantilever subjected to symmetric and asymmetric cyclic loads.

2 Overview of J_2 plasticity

In this section, we summarize the basic equations of J_2 -plasticity for completeness of the paper.

Let $\boldsymbol{\sigma}$ and $\boldsymbol{\varepsilon}$ denote the stress and strain tensors, respectively. In the elastic range, the relation between $\boldsymbol{\sigma}$ and $\boldsymbol{\varepsilon}$ is defined by the linear elastic isotropic constitutive tensor \mathbf{C}^E as

$$\boldsymbol{\sigma} = \mathbf{C}^E : \boldsymbol{\varepsilon}. \quad (1)$$

Let \mathbf{s} and \mathbf{e} denote their deviatoric components; i.e.,

$$\mathbf{s} = \boldsymbol{\sigma} - \frac{1}{3}(\text{tr } \boldsymbol{\sigma})\mathbf{I}, \quad \mathbf{e} = \boldsymbol{\varepsilon} - \frac{1}{3}(\text{tr } \boldsymbol{\varepsilon})\mathbf{I}, \quad (2)$$

where \mathbf{I} is the second-order identity tensor, and $\text{tr}(\cdot)$ is the trace of a tensor. The equivalent stress $\hat{\sigma}$ corresponding to the deviatoric stress \mathbf{s} is defined as

$$\hat{\sigma} = \sqrt{\frac{3}{2}} \|\mathbf{s}\|, \quad (3)$$

where $\|\cdot\|$ is the norm of a tensor.

We use the von Mises type (J_2) yield criterion. The deviatoric components of back-stress tensor, which defines the center of yield surface, is denoted as $\boldsymbol{\alpha}$. Thus the relative stress $\boldsymbol{\xi}$ is given as

$$\boldsymbol{\xi} = \mathbf{s} - \boldsymbol{\alpha}, \quad (4)$$

and $\bar{\sigma}$ is defined using the norm of $\boldsymbol{\xi}$:

$$\bar{\sigma} = \sqrt{\frac{3}{2}} \|\boldsymbol{\xi}\|. \quad (5)$$

The deviatoric component of plastic strain-rate tensor is denoted as \mathbf{d}^P , and the equivalent plastic strain \bar{e}^P at time t is given as

$$\bar{e}^P = \int_0^t \sqrt{\frac{2}{3}} \|\mathbf{d}^P(\tau)\| d\tau. \quad (6)$$

The yield condition is given as follows, by comparing $\bar{\sigma}$ with the radius κ of yield surface:

$$f(\boldsymbol{\sigma}) = \bar{\sigma} - \kappa(\bar{e}^P) = 0. \quad (7)$$

We use associated flow rule for rate form of stress-strain relation. Let $(\dot{\cdot})$ indicate differentiation with respect to time t , although rate-dependency is not considered in the constitutive relation. The deviatoric component of plastic strain-rate tensor is obtained from

$$\mathbf{d}^p = \dot{\bar{e}}^p \mathbf{n}, \quad (8)$$

where \mathbf{n} is the unit normal vector of the yield surface defined as

$$\mathbf{n} = \frac{\boldsymbol{\xi}}{\|\boldsymbol{\xi}\|}. \quad (9)$$

Properties of isotropic hardening and kinematic hardening are defined by $H_I(\bar{e}^p)$ and $H_K(\bar{e}^p)$, respectively. For isotropic hardening,

$$\kappa(\bar{e}^p) = \kappa_0 + H_I(\bar{e}^p), \quad (10)$$

where κ_0 is the initial value of κ . The evolution rules for isotropic hardening and kinematic hardening are defined as

$$\dot{\kappa} = H_I' \dot{\bar{e}}^p, \quad (11a)$$

$$\dot{\boldsymbol{\alpha}} = \sqrt{\frac{2}{3}} H_K' \mathbf{d}^p = \sqrt{\frac{2}{3}} H_K' \dot{\bar{e}}^p \mathbf{n}, \quad (11b)$$

where $(\cdot)'$ denotes differentiation with respect to the equivalent plastic strain.

We use the standard procedure of the elastic predictor-radial return for stress integration [Simo and Taylor (1985)]. The values at the i th incremental step of analysis is denoted by the subscript $(\cdot)_i$. The elastically predicted (trial) value of $\bar{\boldsymbol{\sigma}}$ at the $(i+1)$ th step is denoted by $\bar{\boldsymbol{\sigma}}_{i+1}^{\text{trial}}$. Then the consistency condition is written as

$$g(\Delta \bar{e}_{i+1}^p) = -\kappa(\bar{e}_{i+1}^p) + \bar{\boldsymbol{\sigma}}_{i+1}^{\text{trial}} - [3\mu \Delta \bar{e}_{i+1}^p + H_K(\bar{e}_{i+1}^p) - H_K(\bar{e}_i^p)] = 0, \quad (12)$$

where μ is the shear modulus. The increment $\Delta \bar{e}_{i+1}^p$ of \bar{e}^p is obtained from (12) analytically for the piecewise linear hardening model, and \bar{e}^p is updated as

$$\bar{e}_{i+1}^p = \bar{e}_i^p + \Delta \bar{e}_{i+1}^p. \quad (13)$$

3 Piecewise linear semi-implicit constitutive model

3.1 Semi-implicit constitutive model

In most of the constitutive models, the evolution rules of state variables including $\boldsymbol{\epsilon}$ and $\boldsymbol{\sigma}$ are defined using the internal variables such as κ and $\boldsymbol{\alpha}$, for which evolution

rules are given. These rules are regarded to be *explicit*, because all variables are updated using differential equations. For example, when the input variables are \bar{e}^p , \dot{e}^p , σ , $\dot{\epsilon}$, κ , and α , and the output variables are $\dot{\sigma}$, $\dot{\kappa}$, and $\dot{\alpha}$, the explicit rule is symbolically written as follows:

$$(\dot{\sigma}, \dot{\kappa}, \dot{\alpha}) = \Phi_E(\bar{e}^p, \dot{e}^p, \sigma, \dot{\epsilon}, \kappa, \alpha). \quad (14)$$

A steel material is intrinsically a polycrystal material, and continuum model is an approximation of a complex discrete behavior. Therefore, it is difficult to describe all properties of complex behavior explicitly using differential equations.

By contrast, a rule is said to be *implicit*, if it is not defined using explicit equations and differential equations. Furukawa and Yagawa (1998) developed an implicit rule for visco-elastoplastic material using a neural network. For a simple uniaxial problem in one-dimensional stress space, the input variables are the viscoplastic strain ϵ^{vp} , stress σ , backstress α , and the parameter R for isotropic hardening. The output variables are $\dot{\epsilon}^{vp}$, $\dot{\alpha}$, and \dot{R} . Hence, the implicit mapping is symbolically written as

$$(\dot{\epsilon}^{vp}, \dot{\alpha}, \dot{R}) = \Phi_I(\epsilon^{vp}, \sigma, \alpha, R). \quad (15)$$

They trained the network using a numerical simulation of an existing explicit rule with a specified strain rate in the uniaxial deformation. Therefore, the model is effective for uniaxial model.

For a rate independent plasticity problem in multi-axial stress state, it may not be possible to develop a completely implicit model considering all possible states of the variables and all cases of loading/unloading conditions. Therefore, it is natural to utilize reliable hardening rules defined by differential equations.

An intermediate approach that is a combination of the explicit and implicit rules can be constructed, i.e., some variables are updated explicitly and the others are updated implicitly. This approach is called a *semi-implicit* constitutive model. In the present paper, a semi-implicit constitutive model for a rate independent plasticity problem is proposed. It is based on the conventional explicit constitutive model using the piecewise linear isotropic and kinematic hardening rules with J_2 flow rules. The explicit constitutive model is extended to the semi-implicit one by introducing several implicit rules depending on internal variables such as the maximum equivalent stress experienced so far, equivalent plastic strain at the previous unloading point, etc., which are represented by a vector \mathbf{a} . Then, the semi-implicit rule is written as

$$(\dot{\sigma}, \dot{\kappa}, \dot{\alpha}) = \Phi_{SI}(\bar{e}^p, \dot{e}^p, \sigma, \dot{\epsilon}, \kappa, \alpha; \mathbf{a}). \quad (16)$$

This rule is based on the explicit isotropic-kinematic hardening rule, and variables in the vector \mathbf{a} are updated using implicit or algorithmic rules. Note that other existing explicit constitutive models such as AF model [Armstrong and Frederick (1966)] and Ohno-Wang model [Ohno and Wang (1993)] have parameters that can be regarded as state variables with implicit update rules; i.e., the parameters defining the size of the surface of critical state of Ohno-Wang model and the ratio of dynamic recovery of AF model can be included in the vector \mathbf{a} as internal variables.

3.2 *Implicit modeling of isotropic-kinematic hardening*

It is convenient to use an isotropic or kinematic hardening model with von Mises yield criterion, because the consistency condition (12) is solved analytically. However, a simple isotropic or kinematic model cannot predict accurately the cyclic responses observed in physical tests of steel materials. Therefore, we present a formulation based on piecewise linear combined isotropic-kinematic hardening with implicit rules. Different rules are used for the first and subsequent loading states. Since \bar{e}^p increases monotonically even in cyclic loading state, we introduce a parameter \bar{e}_0^p to record the starting point of \bar{e}^p for each different loading state. The increment of \bar{e}^p from \bar{e}_0^p is denoted by

$$\bar{e}_d^p = \bar{e}^p - \bar{e}_0^p, \tag{17}$$

and the hardening properties become functions of \bar{e}_d^p .

The total hardening coefficient $H'(\bar{e}_d^p)$ is given as the sum of isotropic hardening $H'_I(\bar{e}_d^p)$ and kinematic hardening $H'_K(\bar{e}_d^p)$:

$$H'(\bar{e}_d^p) = H'_I(\bar{e}_d^p) + H'_K(\bar{e}_d^p). \tag{18}$$

Let coefficients c_I and c_K denote the ratios of the isotropic and kinematic hardening effects, respectively, satisfying

$$c_I + c_K = 1. \tag{19}$$

Accordingly, $H'(\bar{e}_d^p)$ is defined as

$$H'(\bar{e}_d^p) = c_K H'_K(\bar{e}_d^p) + c_I H'_I(\bar{e}_d^p). \tag{20}$$

Since $H'(\bar{e}_d^p)$ is a piecewise constant function of \bar{e}_d^p , the pure isotropic and kinematic hardening models are defined by $(c_I, c_K) = (1.0, 0.0)$ and $(0.0, 1.0)$, respectively.

For example, Fig. 1 shows the relation between Cauchy stress and logarithmic s-strain obtained from cyclic uniaxial loading experiments of two rolled mild steel

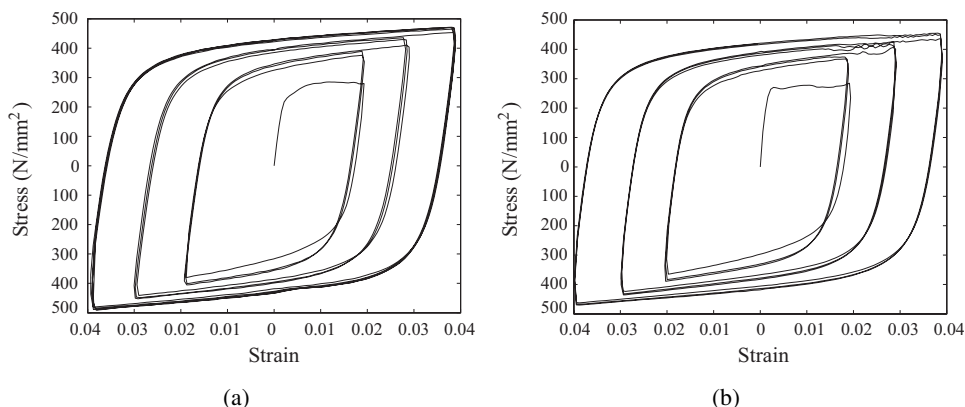


Figure 1: Uniaxial cyclic stress-strain relations of two typical steel materials in civil engineering; (a) SN400, (b) SS400 [Yamada, Imaeda, Okada (2002)].

materials called SS400 and SN400 in Japanese specification, which are widely used for building frames [Yamada, Imaeda, Okada (2002)]. The relation up to the second loading is illustrated in Fig. 2. The following properties can be observed from the experimental results:

1. There exists a wide yield plateau, especially for SS400, in the first plastification (initial yielding); however, no clear yield plateau exists in the subsequent plastifications.
2. Yielding in the second plastic loading (first compressive plastification at ‘C’ in Fig. 2) occurs at a smaller absolute value of stress than the first tensile loading (‘A’) due to *Bauschinger effect*. Furthermore, the difference between the stresses at first unloading (‘B’) and second loading (‘C’) is smaller than twice of the initial yield stress at ‘A’; however, the yield stress gradually increases in the subsequent loading cycles due to isotropic hardening.
3. The stress-strain curves converge to the same loop within a constant range of specified strain; i.e., the material yields at almost the same stress, implying that cyclic isotropic hardening effect is insignificant for this material. Note that the proposed constitutive model can be modified if cyclic hardening with constant magnitude of strain is not negligible.

From the observations of the physical tests, we need at least two types of stress-strain curves, as shown in Fig. 2, to accurately describe the properties of the rolled mild steel materials; one for the first plastic loading with a yield plateau and the

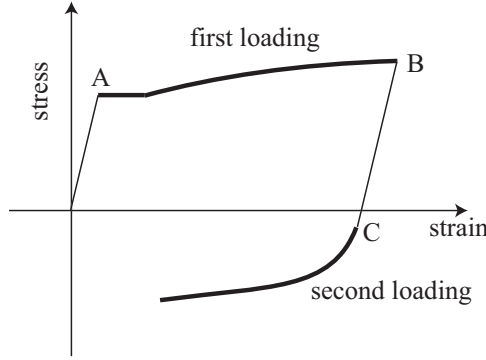


Figure 2: Illustration of two different plastic loading states in uniaxial cyclic loading.

other for the subsequent plastic loading without yield plateau with smooth curvature.

Moreover, as observed in the experimental results, isotropic hardening and kinematic hardening have different coefficients in different loading states. Therefore, the following heuristic and implicit rules are proposed:

1. In the first plastic loading, we use an artificial negative value of the ratio c_I for isotropic hardening with positive H' to simulate the shrinkage of the yield surface as illustrated in Fig. 3.
2. For the subsequent plastic loadings, we have to consider two cases based on the equivalent stress $\hat{\sigma}$ and the maximum value $\hat{\sigma}^{\max}$ experienced in the preceding loading history:
 - If $\hat{\sigma} < \hat{\sigma}^{\max}$, then c_I should have a very small positive value, because isotropic hardening effect is insignificant.
 - If $\hat{\sigma} \geq \hat{\sigma}^{\max}$, then neither isotropic nor kinematic hardening effect can be ignored; hence, c_I and c_K have values around 0.5.

The contribution of yield plateau and Bauschinger effect can be simulated using the AF model, where the back stress $\boldsymbol{\alpha}$ is decomposed to m components as

$$\boldsymbol{\alpha} = \sum_{i=1}^m \boldsymbol{\alpha}_i. \tag{21}$$

The evolution of $\boldsymbol{\alpha}_i$ is defined as

$$\dot{\boldsymbol{\alpha}}_i = \sqrt{\frac{2}{3}} H'_i \mathbf{d}^p - \zeta_i \dot{e}^p \boldsymbol{\alpha}, \tag{22}$$

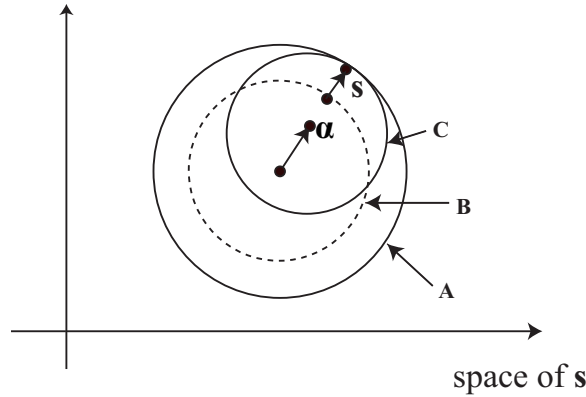


Figure 3: Illustration of artificial shrinkage of yield surface; circle A: true initial yield surface, circle B: artificial initial yield surface that is a little smaller than the true one, circle C: yield surface with small radius κ and nonzero backstress.

where H'_i is the hardening coefficient that is a function of the equivalent plastic strain. The ratio between kinematic and isotropic hardening is controlled by the parameter ζ_i . However, the shrinkage of yield surface after initial yielding cannot be effectively controlled using the AF model, because α initially vanishes and the second term in (22) is negligibly small in the state of first yielding. To overcome this difficulty, Ucak and Tsopelas (2011) developed an approach combining AF model and a bounding surface model.

In our model, the shrinkage of yield surface is controlled by the conventional framework of J_2 -plasticity, for which a stable stress integration algorithm has been developed. We introduce a small interval of $\bar{\epsilon}^p$ with large H' and negative c_I just below the yield plateau at the first plastic loading; i.e., the yield stress is a little smaller than the true value in experiment, and a small artificial plastic region exists below the initial yielding. The process of shrinkage is illustrated in Fig. 3, where circle A is the true initial yield surface, circle B is the artificial initial yield surface that is a little smaller than the true one, and circle C is the yield surface with small radius κ and nonzero backstress. In this way, the size κ of the yield surface is reduced during the first yielding.

To summarize, there are three stress-strain curves in our constitutive model for rolled mild steel materials: curve 1 for the first plastic loading and curves 2 and 3 for the subsequent loadings; and three sets of ratios of hardening: $\{c_I^{(1)}, c_K^{(1)}\}$ for the first loading, and $\{c_I^{(2)}, c_K^{(2)}\}$ and $\{c_I^{(3)}, c_K^{(3)}\}$, respectively, for the cases $\hat{\sigma} < \hat{\sigma}^{\max}$ and $\hat{\sigma} \geq \hat{\sigma}^{\max}$ in the subsequent loadings. Note that $c_I^{(i)} + c_K^{(i)} = 1.0$ ($i = 1, 2, 3$) always holds.

3.3 Radial return mapping

In the proposed semi-implicit constitutive model, a set of piecewise constant values are used for H' in (20). Suppose the computation has converged at the i th incremental step of analysis. The radial return mapping algorithm for updating stresses and plastic strains at an integration point at the $(i + 1)$ th step of analysis, with the given increment $\Delta\boldsymbol{\varepsilon}_{i+1}$ of stain, is summarized in Algorithm 1 below. Procedures for updating the parameters for the semi-implicit constitutive model are also shown in the algorithm.

Algorithm 1: Radial return mapping

Step 1: Elastic predictor

Compute the trial stress as

$$\boldsymbol{\sigma}_{i+1}^{\text{trial}} = \boldsymbol{\sigma}_i + \mathbf{C}_{i+1}^{\text{E}} : \Delta\boldsymbol{\varepsilon}_{i+1}, \quad (23)$$

where $\mathbf{C}_{i+1}^{\text{E}}$ is the elastic constitutive tensor.

Compute $\mathbf{s}_{i+1}^{\text{trial}} = \boldsymbol{\sigma}_{i+1}^{\text{trial}} - \frac{1}{3}(\text{tr } \boldsymbol{\sigma}_{i+1}^{\text{trial}})\mathbf{I}$ by (2), $\boldsymbol{\xi}_{i+1}^{\text{trial}} = \mathbf{s}_{i+1}^{\text{trial}} - \boldsymbol{\alpha}_i$ by (4), $\bar{\sigma}_{i+1}^{\text{trial}} = \sqrt{\frac{3}{2}}\|\boldsymbol{\xi}_{i+1}^{\text{trial}}\|$ by (5), and \mathbf{n} by (9).

Step 2: Plastic corrector

To compute increment $\Delta\bar{e}_{i+1}^{\text{p}}$ of the equivalent plastic strain, we have the following two cases depending on the state of $\bar{\sigma}_{i+1}^{\text{trial}}$.

Case 1: If $\bar{\sigma}_{i+1}^{\text{trial}} < \kappa(\bar{e}_i^{\text{p}})$, then the material is in elastic state and we have

$$\Delta\bar{e}_{i+1}^{\text{p}} = 0. \quad (24)$$

Furthermore, if the material is in plastic state at the i th step, then elastic unloading occurs, and we record the previous equivalent plastic strain as

$$\bar{e}_0^{\text{p}} = \bar{e}_i^{\text{p}}. \quad (25)$$

Case 2: If $\bar{\sigma}_{i+1}^{\text{trial}} \geq \kappa(\bar{e}_i^{\text{p}})$, then the material is in plastic loading state and correction should be carried out to satisfy the consistency condition. To select the proper hardening curve and coefficients, we consider the following cases:

1. First plastic loading (curve 1):

$$H = H_1, \quad c_{\text{I}} = c_{\text{I}}^{(1)}, \quad c_{\text{K}} = c_{\text{K}}^{(1)}; \quad (26)$$

2. Subsequent loading:

$$H = H_2, \tag{27}$$

and

(a) $\hat{\sigma}_i < \hat{\sigma}^{\max}$ (curve 2):

$$c_I = c_I^{(2)}, \quad c_K = c_K^{(2)}; \tag{28}$$

(b) $\hat{\sigma}_i \geq \hat{\sigma}^{\max}$ (curve 3):

$$c_I = c_I^{(3)}, \quad c_K = c_K^{(3)}. \tag{29}$$

$\Delta \bar{e}_{i+1}^p$ can then be computed by solving (12) analytically [Simo and Taylor (1985)].

Step 3: *State updater*

Update \bar{e}_{i+1}^p , α_{i+1} , s_{i+1} , and σ_{i+1} as

$$\bar{e}_{i+1}^p = \bar{e}_i^p + \Delta \bar{e}_{i+1}^p, \tag{30}$$

$$\kappa_{i+1} = \kappa_i + H'_1(\bar{e}_i^p) \Delta \bar{e}_{i+1}^p, \tag{31}$$

$$\alpha_{i+1} = \alpha_i + \sqrt{\frac{2}{3}} H'_K(\bar{e}_i^p) \Delta \bar{e}_{i+1}^p \mathbf{n}, \tag{32}$$

$$s_{i+1} = \alpha_{i+1} + \sqrt{\frac{2}{3}} \kappa(\bar{e}_{i+1}^p) \mathbf{n}, \tag{33}$$

$$\sigma_{i+1} = s_{i+1} + \frac{1}{3} (\text{tr } \sigma_{i+1}^{\text{trial}}) \mathbf{I}. \tag{34}$$

If $\hat{\sigma}_{i+1}$ computed from (3) is larger than the currently recorded maximum value $\hat{\sigma}^{\max}$ in the loading history, then update $\hat{\sigma}^{\max}$ as

$$\hat{\sigma}^{\max} = \hat{\sigma}_{i+1} = \sqrt{\frac{3}{2}} \|s_{i+1}\|. \tag{35}$$

If the stresses obtained from Algorithm 1 satisfy the equilibrium equations at each node, then the $(i + 1)$ th step of the analysis converges; otherwise, calculate a new increment of strain by considering the unbalanced forces and apply the algorithm again.

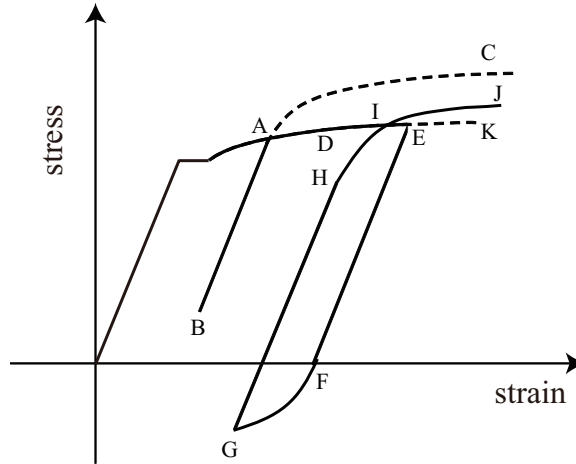


Figure 4: Stress-strain curve under asymmetric uniaxial cyclic loading.

3.4 Response to asymmetric cyclic loading

It is well known that simulation of responses to asymmetric cyclic loading is more difficult than that to symmetric loading. Fig. 4 illustrates a stress-strain relation of an asymmetric uniaxial cyclic loading. The solid lines are the correct stress-strain relations, while the dashed lines are the ‘wrong’ relations due to improper modeling of hardening effect:

- In the process of reloading after no plastic loading in the opposite direction, the stress-strain path returns onto the extension of the stress-strain curve experienced in the previous plastic loading. Thus, after unloading at ‘A’ in Fig. 4 and reversal of direction at ‘B’, the stress-strain curve returns to ‘A’, and follows the original curve to ‘D’. Hence, curve ‘AC’ shown in dashed line is inappropriate.
- When plastification occurs after unloading at ‘E’, plastic strain accumulates between ‘F’ and ‘G’, and the stress-strain hysteresis loops are open so that they resemble a creep effect under a large number of plastic loading cycles [Chaboche (2008)]. Thus, the reloading occurs at ‘I’ that is a different point from the unloading point ‘E’, and stress-strain relation follows curve ‘IJ’ which is different from the original curve. However, if the plastic deformation between ‘F’ and ‘G’ is small, the stress-strain path returns onto the original curve, i.e., it follows the curve ‘IEK’.

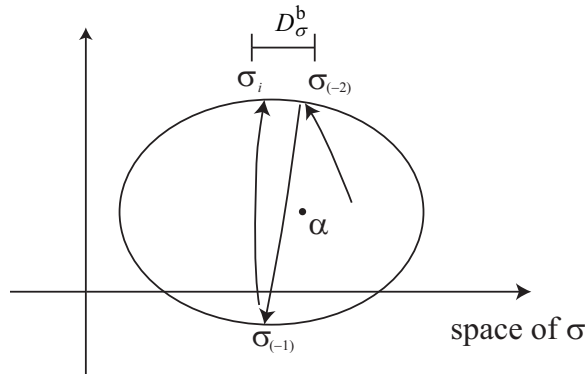


Figure 5: History of stress under cyclic loading.

In order to accurately simulate the responses to asymmetric loading, we use σ and \bar{e}^p at the preceding unloading points as the internal parameters. Let $\sigma_{(-j)}$ and $\bar{e}_{(-j)}^p$ denote the values of σ and \bar{e}^p at the j th previous unloading point as illustrated in Fig. 5. The following procedures are added after Case 1 in Step 2 of Algorithm 1:

$$\begin{aligned}
 \sigma_{(-1)} &:= \sigma_i, \\
 e_{(-1)}^p &:= e_i^p, \\
 \sigma_{(-j-1)} &:= \sigma_{(-j)}, \quad j = 1, 2, \dots, \\
 e_{(-j-1)}^p &:= e_{(-j)}^p, \quad j = 1, 2, \dots
 \end{aligned}
 \tag{36}$$

The stress-strain relation depends on the variable vector \mathbf{a} in (14) consisting of $\sigma_{(-1)}$, $\sigma_{(-2)}$, $\bar{e}_{(-1)}^p$, $\bar{e}_{(-2)}^p$, $\hat{\sigma}^{\max}$, and an integer indicating the current curve number. To simulate responses against asymmetric cyclic loading, the following Algorithm 2 is added at the end of Case 2 in Step 2 of Algorithm 1:

Algorithm 2: Simulation of responses to asymmetric loading

If the material is currently in plastic state, then consider the following two cases:

1. If the material is in elastic state at the previous step, and the norm of difference between the current stress σ_i and the stress $\sigma_{(-1)}$ at the previous unloading point is smaller than a specified small positive value D_σ^a ; i.e.,

$$\|\sigma_i - \sigma_{(-1)}\| \leq D_\sigma^a,
 \tag{37}$$

then reloading occurs without plastic loading in the opposite direction; hence, we set

$$\bar{e}_0^p = \bar{e}_{(-2)}^p,
 \tag{38}$$

and update

$$\begin{aligned} \bar{e}_{(-j)}^p &:= \bar{e}_{(-j-1)}^p, \quad j = 1, 2, \dots, \\ \sigma_{(-j)} &:= \sigma_{(-j-1)}, \quad j = 1, 2, \dots, \end{aligned} \tag{39}$$

so as to follow the curve of the previous plastic loading as illustrated as the path $A \rightarrow B \rightarrow A \rightarrow D$ in Fig. 4.

2. If the difference between the equivalent plastic strains $\bar{e}_{(-1)}^p$ and $\bar{e}_{(-2)}^p$ at the two successive previous unloading points is smaller than a specified small positive value $D_{\bar{e}^p}$; i.e.,

$$\bar{e}_{(-1)}^p - \bar{e}_{(-2)}^p \leq D_{\bar{e}^p}, \tag{40}$$

and the difference between the current stress σ_i and the stress $\sigma_{(-2)}$ at the second previous unloading points is smaller than a specified small value D_{σ}^b , as illustrated in Fig. 5; i.e.,

$$\|\sigma_i - \sigma_{(-2)}\| \leq D_{\sigma}^b, \tag{41}$$

then the plastification in the opposite direction is negligibly small. Therefore, we set

$$\bar{e}_0^p = \bar{e}_{(-3)}^p + (\bar{e}_i^p - \bar{e}_{(-2)}^p), \tag{42}$$

and update

$$\begin{aligned} \bar{e}_{(-j)}^p &:= \bar{e}_{(-j-2)}^p, \quad j = 1, 2, \dots, \\ \sigma_{(-j)} &:= \sigma_{(-j-2)}, \quad j = 1, 2, \dots, \end{aligned} \tag{43}$$

so as to follow the previous plastic loading curve.

According to Algorithm 2, the yield plateau does not appear at reloading after unloading from the plateau even if the equivalent plastic strain is very small. However, as noted in Ucak and Tsopelas (2011) and Rodzik (1999), the yield plateau remains if the equivalent plastic strain is small enough. Therefore, the following rule is added:

Algorithm 3: Simulation of unloading from yield plateau

Use curve 1 if $\bar{e}^p \leq \bar{e}^{YP}$, where \bar{e}^{YP} is a parameter representing the bound for the accumulated width of the yield plateau for the cyclic loading.

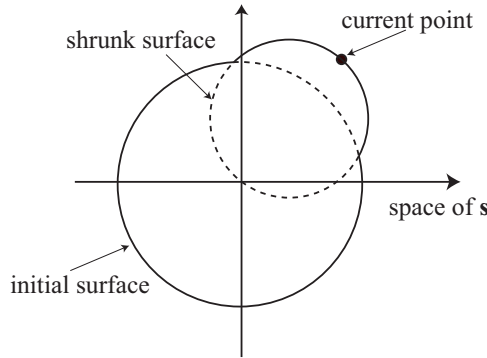


Figure 6: Illustration of union of initial yield surface and shrunk surface.

However, the size of yield surface has been shrunk after first yielding. Therefore, the stress of reloading at the yield surface is smaller than the real value after slightly experiencing plastic deformation in opposite direction. To prevent this unrealistic situation, we use the union of initial and shrunk surfaces as the yield surface as illustrated in Fig. 6 until \bar{e}^p exceeds \bar{e}^{YP} .

In the following applications, Algorithms 1–3 are implemented in the material library of the E-Simulator. The accuracy and stability of the algorithms have been confirmed through various numerical experiments including the seismic response analysis of a model with about 19 million degrees of freedom [Miyamura, Yamashita, Akiba, Ohsaki (2015)].

4 Parameter identification from material test

This section proposes an identification method for the parameters of the semi-implicit constitutive model using results of cyclic uniaxial material tests. The parameters to be identified, which are the variables of an optimization problem, include the initial yield stress, the hardening coefficients of the piecewise linear stress-strain curves, and the ratios of isotropic-kinematic hardening. The square error between analysis and experiment results is minimized to find the optimal parameter values.

4.1 Problem description

The hardening curve is divided into several intervals with respect to $\bar{e}_{(-j)}^p$. Let $\bar{e}_{[i]}^p$ denote the value of \bar{e}^p at the end of the i th interval. Then, the i th interval is defined by

$$\bar{e}_{[i-1]}^p \leq \bar{e}^p \leq \bar{e}_{[i]}^p, \tag{44}$$

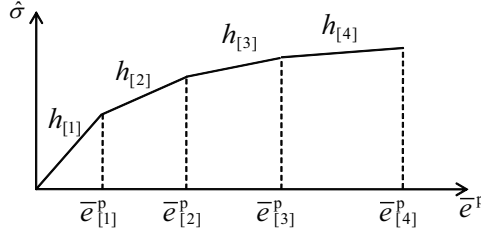


Figure 7: Piecewise linear relation between equivalent plastic strain and equivalent stress for a monotonic loading with constant hardening coefficient $h_{[i]}$ in the i th interval.

with $\bar{e}_{[0]}^p = 0$. In the following, we use h instead of H' for simple presentation of variables. The strain hardening coefficient in the i th interval is denoted by $h_{[i]}$, which is assumed to have a constant value in each interval as illustrated in Fig. 7. Note that \bar{e}^p should be replaced by \bar{e}_d^p in (17) in a reloading process.

Let $n^{(k)}$ ($k = 1, 2, 3$) denote the number of intervals in the k th loading curve. The values $\bar{e}_{[i]}^{p(k)}$ ($i = 1, \dots, n^{(k)}$) are fixed. The ratios $c_1^{(k)}$ ($k = 1, 2, 3$) of isotropic hardening and the initial yield stress σ_y^0 are also considered as variables. The hardening coefficients of curve 3 are the same as those of curve 2, and the constant values $c_1^{(2)}$ and $c_1^{(3)}$ are used for all intervals of curves 2 and 3, respectively. The ratio $c_1^{(1)}$ represents the artificial negative value before initial yielding, and $c_1^{(2)}$ is used for the remaining intervals of curve 1. Therefore, we have the following $n_1 + n_2 + 4$ variables:

$$\begin{aligned} &\sigma_y^0, \\ &c_1^{(1)}, c_1^{(2)}, c_1^{(3)}, \\ &h_{[i]}^{(k)} \quad (i = 1, \dots, n^{(k)}; k = 1, 2), \end{aligned} \tag{45}$$

which are represented by a vector \mathbf{x} . Note that the elastic modulus is identified from the initial stiffness only. Fixed values are assigned for D_σ^a , D_σ^b , and $D_{\bar{e}^p}$.

Let $\tilde{\sigma}(t_i)$ and $\tilde{e}(t_i)$ denote the stress and strain at time step t_i of cyclic uniaxial test. We carry out cyclic uniaxial analysis of a single element with the same history of the forced strain as the experiment to obtain the uniaxial stress $\hat{\sigma}^a(t_i)$ and strain $e(t_i)$ at time step t_i .

We minimize the error between $\hat{\sigma}^a(t_i)$ and $\tilde{\sigma}(t_i)$ defined as

$$E(\mathbf{x}) = \sqrt{\frac{1}{|\mathcal{D}|} \sum_{i \in \mathcal{D}} [\hat{\sigma}^a(t_i) - \tilde{\sigma}(t_i)]^2}, \tag{46}$$

where the summation is carried out in the set of steps \mathcal{P} in plastic loading state. Constraints are given, as follows, so that the hardening coefficient is a non-increasing function of the equivalent plastic strain:

$$h_{[i]}^{(k)} \geq h_{[i+1]}^{(k)}, \quad (k = 1, 2; i = 1, \dots, n^{(k)}). \tag{47}$$

The first interval of curve 1 has a negative value of $c_1^{(1)}$ to simulated the Bauschinger effect. If yield plateau exists, it is represented by the second interval of curve 1 (coefficient $h_{[2]}^{(1)}$), and the hardening after yield plateau is represented by the coefficients $h_{[i]}^{(1)}$ ($i \geq 3$). In this case, the constraint $h_{[2]}^{(1)} \geq h_{[3]}^{(1)}$ is not given.

Since the optimization problem is highly nonlinear, we discretize the variables into integer values, and use TS [Glover (1989)], which is a heuristic approach. Let x_i represent one of the parameters to be identified. The upper and lower bounds of x_i are denoted by x_i^U and x_i^L , respectively, which are assigned as shown in Tab. 1. Each variable is discretized into s equally spaced values x_{ij} ($j = 1, \dots, s$) as

$$x_{ij} = x_i^L + \frac{j-1}{s}(x_i^U - x_i^L), \quad (j = 1, \dots, s). \tag{48}$$

The algorithm of TS is summarized as follows:

- Step 1** Assign an initial seed solution for \mathbf{x} , and initialize the tabu list T to be empty.
- Step 2** Generate neighborhood solutions of the current seed solution and move to the best feasible solution \mathbf{x}^* among them that is not included in the list T .
- Step 3** Add \mathbf{x}^* to T . Remove the oldest solution in T if the length of the list exceeds the specified limit.
- Step 4** Accept \mathbf{x}^* as the next seed solution and go to Step 2 if the number of steps is less than the specified limit; otherwise, output the best solution and terminate the process.

The neighborhood solutions are generated using a random number $\tau \in [0, 1]$. Each variable is increased by 1 if $\tau \geq 2/3$, decreased by 1 if $\tau < 1/3$, and is not modified if $1/3 \leq \tau < 2/3$.

In order to improve accuracy, TS is carried out from several different initial random seeds, after re-defining the upper and lower bounds of variables, as follows, around the best solution of each trial. Let x_i^k denote the i th variable of the k th optimal

solution. To reduce the range of search region, the bounds of x_i are reassigned to x_i^{U*} and x_i^{L*} as

$$x_i^{U*} = x_i^k + \frac{x_i^U - x_i^L}{s}, \quad x_i^{L*} = x_i^k - \frac{x_i^U - x_i^L}{s} \quad (49)$$

Each parameter is discretized again into s integer values in a similar manner as (48), and TS is carried out again from several different random seeds.

4.2 Parameter identification using cyclic test

In the following, the units of length and force are mm and N, respectively. The parameters are identified from the uniaxial cyclic material test of SS400 [Yamada, Imaeda, Okada (2002)] as shown in Fig. 1(b). We cannot identify the hardening coefficients beyond yield plateau of curve 1, because the first unloading occurs from the yield plateau in Fig. 1(b). Furthermore, we fix the hardening coefficient at yield plateau, and only the hardening coefficient in the small interval at the first yielding is considered as variable; i.e, n_1 in (45) is 1. The number of intervals in the subsequent stress-strain relation is seven; i.e., $n_2 = 7$ in (45). We have twelve variables in total including σ_y^0 , $c_1^{(1)}$, $c_1^{(2)}$, and $c_1^{(3)}$.

Optimization using TS is carried out for identification of parameters. Values of the fixed parameters are set as

- Elastic modulus: $1.7505 \times 10^5 \text{ N/mm}^2$
- Poisson’s ratio: 0.3
- Hardening coefficient $h_{[2]}^{(1)}$ for the yield plateau: 100.0 N/mm^2 .
- Equivalent plastic strains at the boundaries of intervals:
 - Curve 1: $\{\bar{e}_{[1]}^{p(1)}, \bar{e}_{[2]}^{p(1)}\} = \{0.0000315, 1.0\}$
 - Curves 2 and 3: $\{\bar{e}_{[1]}^{p(2)}, \dots, \bar{e}_{[7]}^{p(2)}\} = \{0.0015, 0.0044, 0.01, 0.018, 0.025, 0.05, 1.0\}$

The variables are discretized into $s = 20$ values. The numbers of neighborhood solutions and total steps for TS are 12 and 50, respectively. The length of tabu list is large enough so that each solution is selected only once. TS is carried out from five different initial random seeds to find five approximate solutions. Parameters are re-discretized to 20 values around each approximate solution, and TS is carried out again from five different random seeds. The five approximate solutions, denoted by

Table 1: Bounds of parameters.

	Lower bound	Upper bound
$c_1^{(1)}$	-20.0	-10.0
$c_1^{(2)}$	0.0	0.2
$c_1^{(3)}$	0.3	0.8
$h_{[1]}^{(1)}$	1.0×10^4	21.0×10^5
$h_{[1]}^{(2)}$	1.0×10^4	21.0×10^5
$h_{[2]}^{(2)}$	1.0×10^4	21.0×10^5
$h_{[3]}^{(2)}$	5000.0	15000.0
$h_{[4]}^{(2)}$	1000.0	5000.0
$h_{[5]}^{(2)}$	100.0	5000.0
$h_{[6]}^{(2)}$	100.0	5000.0
$h_{[7]}^{(2)}$	100.0	5000.0
σ_y^0	260.0	270.0

Opt k ($k = 1, \dots, 5$), obtained as the best solution after the second step are listed in Tab. 2.

Among five solutions, Opt 3 has the smallest value of $E(\mathbf{x})$, and its stress-strain relation is plotted in dotted line in Fig. 8. The solid line shows the experimental result by Yamada, Imaeda, Okada (2002). As seen from the figure, the numerical results have high accuracy of simulating cyclic uniaxial loading of the steel material.

In the following examples of a cantilever, materials for flange and web are classified as SS400; however, they are fabricated from different lots, and only monotonic test results are available. We use the well-known fact that the plot of equivalent stress with respect to accumulated value of $\bar{\epsilon}^p$ under cyclic deformation resembles the plot with respect to $\bar{\epsilon}^p$ under monotonic deformation [Yamada, Imaeda, Okada (2002)]. Hardening properties of curve 1, after removing the yield plateau, are used for identifying the parameters $h_{[i]}^{(2)}$ ($i = 1, \dots, 7$) and σ_y^0 for flange and web. For the cyclic properties, the parameter values of Opt 3 in Tab. 2 are used for $c_1^{(1)}$, $c_1^{(2)}$, $c_1^{(3)}$, and $h_{[1]}^{(1)}$. The same cyclic pattern as Fig. 1(b) is applied to a single element, and the error between the accumulated plastic strain and the experimental result is minimized using the same approach as cyclic loading.

The results for flange and web are listed in Tab. 3. The monotonic stress-strain relation for flange and web are plotted in Figs. 9(a) and (b), respectively, where solid lines are the experimental results, and dotted lines are the numerical results

Table 2: Results of second step of parameter identification by TS.

	Opt 1	Opt 2	Opt 3	Opt 4	Opt 5
$c_I^{(1)}$	-14.921	-19.921	-16.816	-13.342	-17.8684
$c_I^{(2)}$	0.0010263	0.0013421	0.00150000	0.000500	0.0005
$c_I^{(3)}$	0.58289	0.63553	0.61447	0.56184	0.66711
$h_{[1]}^{(1)}$	1.7053×10^5	1.5579×10^5	1.5579×10^5	1.6526×10^5	1.2526×10^5
$h_{[1]}^{(2)}$	8.2105×10^4	1.0737×10^5	7.4737×10^4	6.5263×10^4	5.5789×10^4
$h_{[2]}^{(2)}$	1.3973×10^4	3.0526×10^4	3.3684×10^4	3.2632×10^4	3.0526×10^4
$h_{[3]}^{(2)}$	1.3973×10^4	5763.2	9394.7	7763.2	9868.4
$h_{[4]}^{(2)}$	3465.5	5002.6	3534.2	3055.3	4339.5
$h_{[5]}^{(2)}$	3465.5	2485.5	1995.5	2691.8	2176.1
$h_{[6]}^{(2)}$	3465.5	2176.1	1995.5	2691.8	2176.1
$h_{[7]}^{(2)}$	3465.5	1943.9	1196.1	2691.8	912.37
σ_y^0	270.13	265.61	265.18	268.71	263.24
$E(\mathbf{x})$	23.721	11.475	10.528	16.232	11.272

corresponding to Opt 2-1 for flange and Opt 2-5 for web. It can be observed from the figures that good approximation has been achieved.

5 Illustrative example and FE analysis of cantilever

We demonstrated in the previous section that the proposed semi-implicit constitutive model has high accuracy in simulation of cyclic uniaxial loading for rolled mild steel materials in civil engineering. In this section, we first demonstrate the necessity of implementation of Algorithms 2 and 3 in simulation of cyclic loading. Cyclic analysis is carried out for a cantilever as shown in Fig. 10 as an example to show the accuracy of the proposed model used for structural analysis.

5.1 Cyclic analysis of single element

To illustrate the importance of implementation of Algorithms 2 and 3 for cyclic loading, we consider a simple example with only one hexahedral solid element. The parameters for Algorithms 2 and 3 are given as

- Small bounds for Algorithm 2:
 $D_\sigma^a = 5.0$, $D_\sigma^b = 15.0$, $D_{\bar{e}^p} = 0.004$
- Threshold plastic strain of the yield plateau for Algorithm 3: $\bar{e}^{YP} = 0.005$

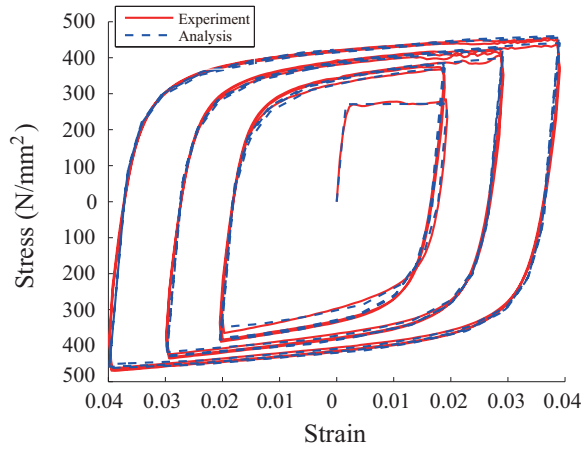


Figure 8: Stress-strain relation of for the cyclic uniaxial material test of SS400 using the identified parameter values.

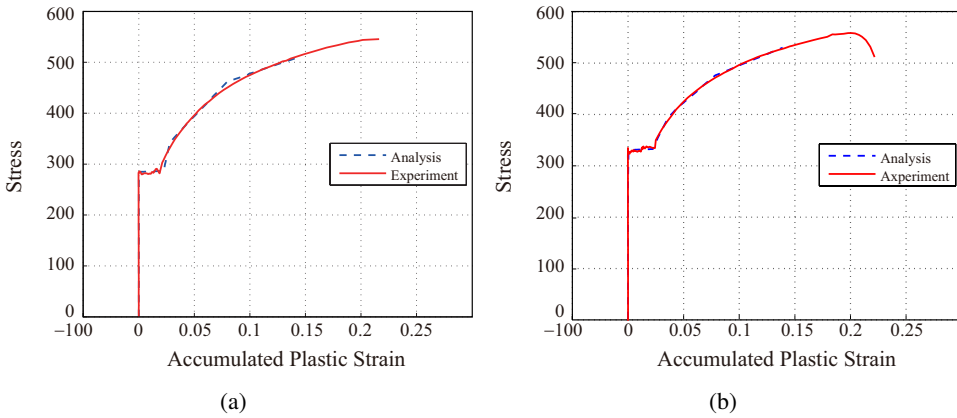


Figure 9: Stress-strain curves for flange and web after identification of hardening coefficients and yield stresses from uniaxial material test, corresponding to Opt 3; (a) flange, (b) web.

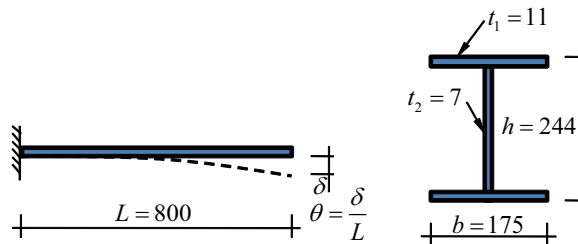


Figure 10: Geometry of cantilever and its cross-section; (Unit: mm).

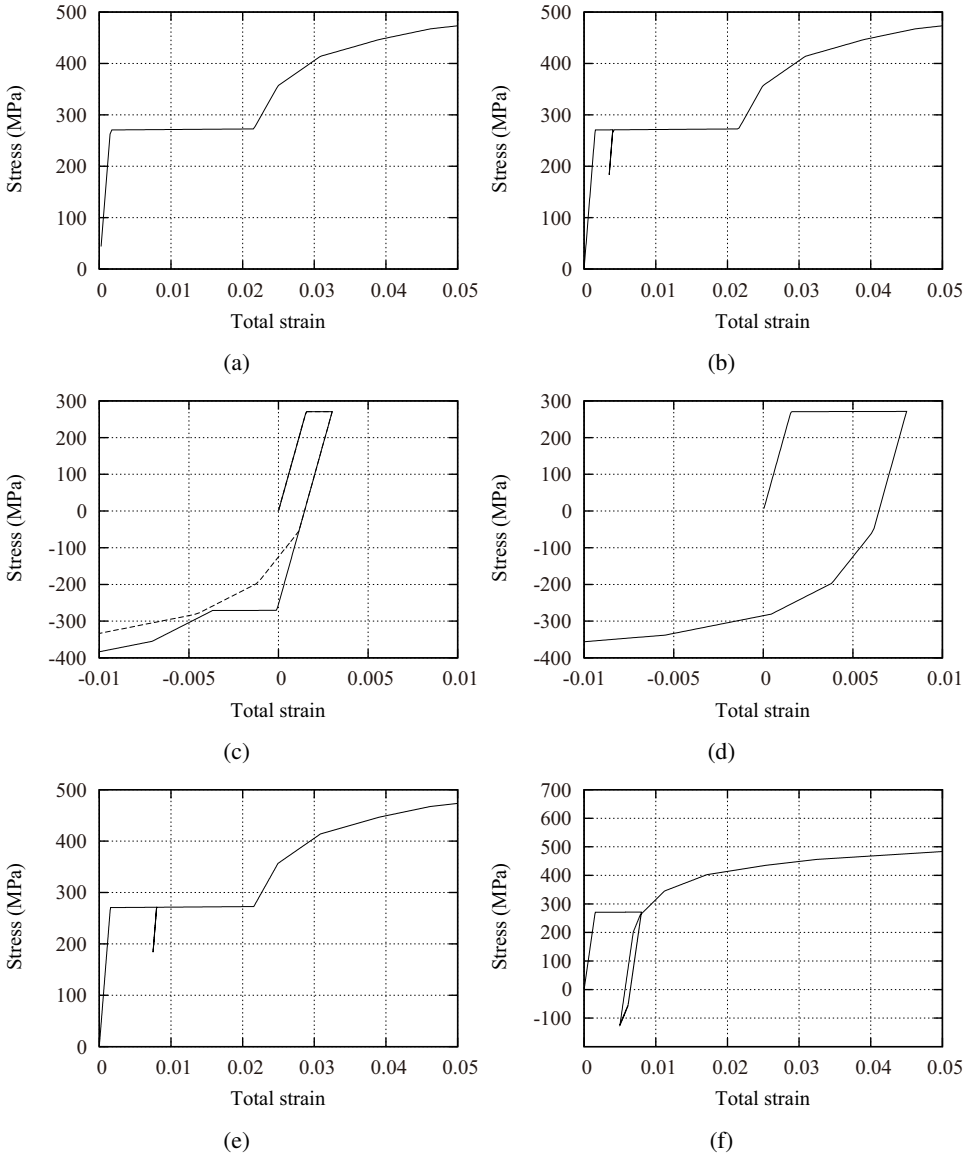


Figure 11: Cyclic behavior with reloading from yield plateau; (a) monotonic loading, (b) reloading without loading in the opposite direction, (c) reloading at small equivalent plastic strain after loading in the opposite direction, (d) reloading at large equivalent plastic strain after loading in the opposite direction, (e) reloading at large equivalent plastic strain without loading in the opposite direction, (f) reloading at large equivalent plastic strain after slight loading in the opposite direction.

Table 3: Results of parameter identification of curve 2 and yield stress of flange and web using monotonic uniaxial tests.

		Opt 2-1	Opt 2-2	Opt 2-3	Opt 2-4	Opt 2-5
Flange	$h_{[1]}^{(2)}$	86447.3	89342.1	89342.1	86973.6	88815.7
	$h_{[2]}^{(2)}$	29526.3	28473.6	29947.3	28052.6	28473.6
	$h_{[3]}^{(2)}$	11348.6	10851.3	11277.6	10851.3	10993.4
	$h_{[4]}^{(2)}$	2777.63	2759.21	2667.10	2740.78	2678.94
	$h_{[5]}^{(2)}$	2678.94	2752.63	2594.73	2657.89	2678.94
	$h_{[6]}^{(2)}$	2584.21	2584.21	2594.73	2605.26	2584.21
	$h_{[7]}^{(2)}$	848.552	859.868	825.921	871.184	882.500
	σ_y^0	279.736	278.684	273.947	279.210	279.210
	$E(\mathbf{x})$	5.78927	6.13947	7.22870	6.02356	6.01854
	Web	$h_{[1]}^{(2)}$	86184.2	86973.6	88815.7	88026.3
$h_{[2]}^{(2)}$		26473.6	26894.7	27842.1	26789.4	26263.1
$h_{[3]}^{(2)}$		5877.63	5806.57	6517.10	5877.63	5877.63
$h_{[4]}^{(2)}$		5006.57	4951.31	4877.63	5006.57	4877.63
$h_{[5]}^{(2)}$		2784.21	2763.15	2700.00	2678.94	2784.21
$h_{[6]}^{(2)}$		2121.05	2089.47	1994.73	2100.00	2110.52
$h_{[7]}^{(2)}$		1046.18	1034.86	1000.92	1012.23	1023.55
σ_y^0		325.000	325.000	323.947	325.000	325.000
$E(\mathbf{x})$		2.89199	2.89152	3.07475	2.90781	2.87298

Note that the width of yield plateau in the monotonic loading test is about 0.02 for flange and 0.026 for web. The values of D_{σ}^a and D_{σ}^b are about 1.8 % and 5.4 % of the yield stress, and $D_{\bar{e}^p}$ is about 2.3 times as large as the yield strain. It has been confirmed that the numerical results below are not sensitive to these values.

Fig. 11(a) shows a monotonic stress-strain relation of an element subjected to normal stress, which has a clear yield plateau. Fig. 11(b) shows the cyclic behavior involving unloading from the yield plateau. It is seen that the yield plateau is traced after reloading without loading in the opposite direction. Fig. 11(c) demonstrates the loading in the opposite direction after unloading from the yield plateau. The value of \bar{e}^p at the unloading point is 0.0014532, which is less than \bar{e}^{YP} . Therefore, yield plateau exists in the opposite direction. The stress-strain relation follows the dotted line if Algorithm 3 is not used.

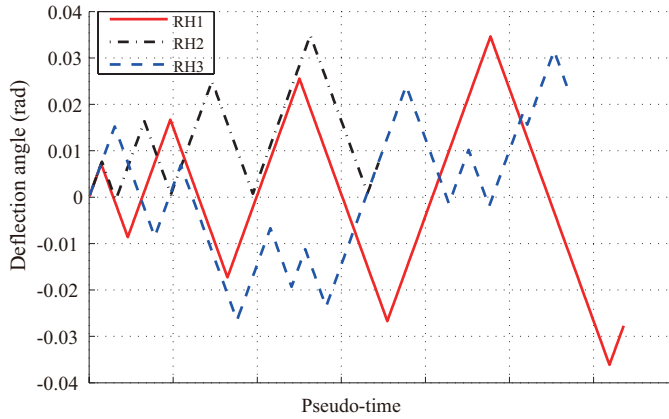


Figure 12: Loading patterns for the cantilever.

If unloading occurs at $\bar{e}^p = 0.0064503$, which is larger than \bar{e}^{YP} , the yield plateau disappears and the stress-strain relation exhibits Bauschinger effect as shown in Fig. 11(d), which confirms the effectiveness of using multiple curves. If unloading and reloading occurs at $\bar{e}^p = 0.0064503$ in the same direction without loading in the opposite direction, the stress-strain curve follows curve 2 as shown in Fig. 11(e). However, if loading is experienced in the opposite direction, then yields plateau disappears as shown in Fig. 11(f).

5.2 *Cyclic analysis of cantilever*

To demonstrate the effectiveness of the proposed constitutive model for analysis of structures in building engineering, we carry out elastoplastic cyclic analysis of a cantilever consisting of rolled wide-flange section H-244×175×7×11 as shown in Fig. 10. The web and flange are made of the same material SS400 with different yield stresses. The left end of cantilever is clamped, and forced vertical displacement is given at the right end. The average deflection angle θ of the beam is defined by dividing the tip displacement δ by the beam length $L = 800$ mm.

Yamada, Imaeda, Okada (2002) conducted physical experiments under three different loading patterns RH1, RH2, and RH3 described in terms of deflection angles as shown in Fig. 12. The loading pattern RH1 is symmetric, RH2 gradually deflects in one direction, and RH3 is asymmetric. The cantilever is discretized using hexahedral finite elements with quadratic interpolation function. The flanges and web are divided into three layers as shown in Fig. 13. Each element has 20 nodes, and the total numbers of elements and nodes are 2700 and 14167, respectively. Sufficiently small increment is used to simulate the history of complex behavior of the material using the semi-implicit constitutive model. The total number of incremental steps

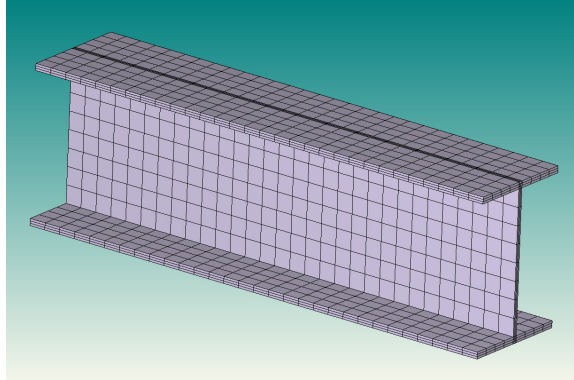


Figure 13: Finite element mesh of the cantilever.

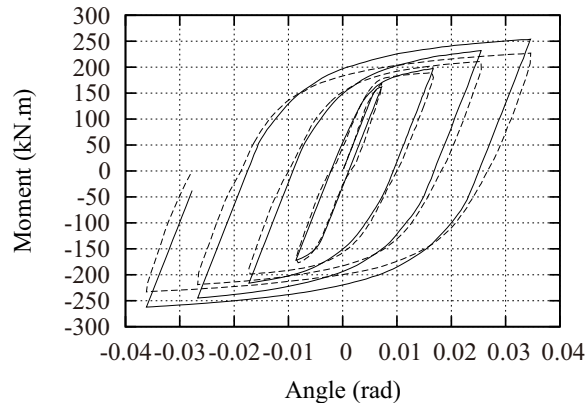


Figure 14: Relation between average deflection angle and bending moment for loading pattern RH1; solid line: analysis, dotted line: experiment.

for the loading patterns RH1, RH2, and RH3 are 555, 362, and 443, respectively.

We use the parameter values identified in Sec. 4. For the symmetric loading pattern RH1, the relation between the deflection angle and the bending moment at the fixed end is shown in Fig. 14, where the solid and dotted lines, respectively, correspond to numerical and experimental results. The numerical analysis reproduces the behavior of the cantilever with good accuracy in the subsequent cycles, although the strength is slightly larger than the experimental result.

For the more complex loading patterns RH2 and RH3, the numerical results together with the experimental results are respectively plotted in Figs. 15(a) and (b). As seen from the figures, the responses evaluated using the proposed model has

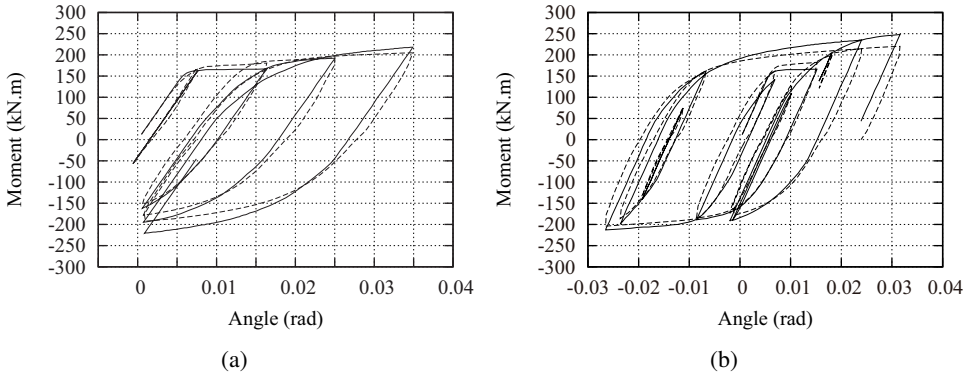


Figure 15: Results of asymmetric loading patterns; solid line: analysis, dotted line: experiment; (a) RH2, (b) RH3.

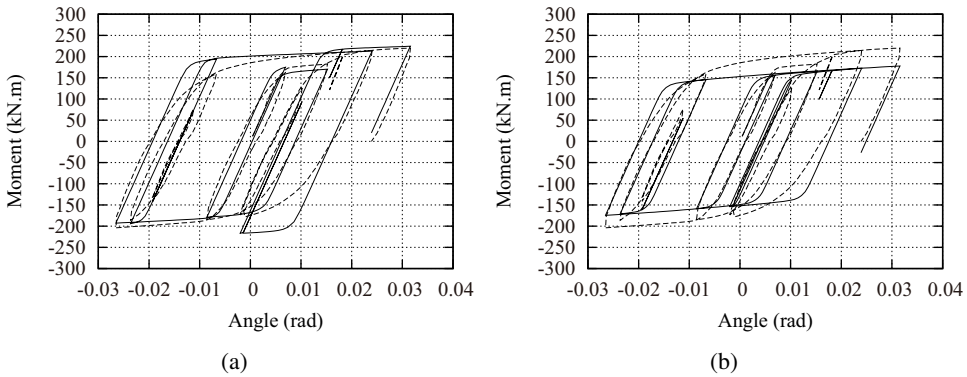


Figure 16: Results of simple hardening rules for loading pattern RH3; solid line: analysis, dotted line: experiment; (a) isotropic, (b) kinematic.

good accuracy for even these asymmetric loading patterns. It should be noted here that the constitutive parameters are identified using only the cyclic and monotonic material tests, and no tuning has been made in view of the results of cantilever subjected to three different loading conditions.

For comparison purpose, we also carried out numerical analysis for RH3 using simple isotropic and kinematic hardening rules, respectively, where the bilinear hardening models are used and the hardening coefficient is equal to 1/100 of the elastic modulus. The numerical results for the two cases are shown in Fig. 16, also with the experimental results for reference. It is obvious from Figs. 15(b) and 16 that the proposed constitutive model has much higher accuracy than simple isotropic or kinematic hardening model in simulation of behavior of the cantilever subjected to

a symmetric cyclic loading. Obviously, the expansion of yield surface cannot be simulated by the kinematic hardening model, while the isotropic hardening cannot simulate the Bauschinger effect.

6 Conclusions

An intermediate implicit and explicit constitutive model, called semi-implicit model, has been proposed for the rate-independent elastoplastic constitutive models of rolled mild steel materials used in the field of civil engineering. The model simulates the behaviors of steel materials subjected to cyclic loading, while existence of yield plateau and Bauschinger effect is considered.

The constitutive model has been implemented in the E-Simulator, which is a software package for large-scale seismic response analysis and is currently under development at the National Research Institute of Earth Science and Disaster Prevention (NIED), Japan. The accuracy and stability of the prototype of the constitutive model have already been confirmed through various numerical experiments including the seismic response analysis of a model with about 19 million degrees of freedom.

The constitutive relation is defined in an algorithmic manner based on the piecewise-linear combined isotropic-kinematic hardening. Different rules are used for the first and subsequent loading states to consider the wide yield plateau in the first loading, and a rule is also included for simulating reloading after unloading from the yield plateau. Moreover, the Bauschinger effect is simulated by shrinkage of the yield surface using a negative ratio for isotropic hardening.

The conventional formulations of radial return mapping for predictor-corrector incremental algorithm and the consistent tangent stiffness proposed in [Simo and Taylor (1985)] can be used without any modification incorporating explicit solution of the consistency condition. Therefore, the proposed model can be implemented easily by simply adding the new hardening functions to an existing and reliable code, and very stable computation without divergence can be achieved in the framework of J_2 -plasticity.

An optimization approach using TS has been presented for identification of the constitutive parameters from uniaxial material tests. The ratio of isotropic hardening as well as the parameter values for modeling Bauschinger effect is identified from a cyclic material test. The hardening coefficient as function of plastic strain can be identified from uniaxial material test, if cyclic test is not available, utilizing the accumulated equivalent plastic strain under cyclic analysis. Hence, the proposed approach can be widely used in a practical situation, because cyclic test is much more difficult than monotonic test.

The accuracy of the model has been demonstrated through numerical examples of a

cantilever subjected to three different types of cyclic loads. It has been shown that the elastoplastic responses under complex asymmetric cyclic loads can be simulated accurately using the proposed model without any specific tuning for the structural model; only material tests for identification of the parameters are needed for accurate estimation of elastoplastic cyclic responses.

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