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Finite Element Simulation Analysis of a Novel 3D-FRSPA for Crawling Locomotion

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ABSTRACT

A novel three-dimensional-fiber reinforced soft pneumatic actuator (3D-FRSPA) inspired by crab claw and human hand structure that can bend and deform independently in each segment is proposed. It has an omni-directional bending configuration, and the fibers twined symmetrically on both sides to improve the bending performance of FRSPA. In this paper, the static and kinematic analysis of 3D-FRSPA are carried out in detail. The effects of fiber, pneumatic chamber and segment length, and circular air chamber radius of 3D-FRSPA on the mechanical performance of the actuator are discussed, respectively. The soft mobile robot composed of 3D-FRSPA has the ability to crawl. Finally, the crawling processes of the soft mobile robot on different road conditions are studied, respectively, and the motion mechanism of the mobile actuator is shown. The numerical results show that the soft mobile robots have a good comprehensive performance, which verifies the correctness of the proposed model. This work shows that the proposed structures have great potential in complex road conditions, unknown space detection and other operations.

KEYWORDS

3D-FRSPA; bionic structure; theoretical modeling; crawling application; obstacle avoidance analysis; numerical investigation

1 Introduction

The rigid crawling mobile robots inspired by biological structure are widely used in various industrial practices and auxiliary equipment, such as cargo handling robots, lower limb exoskeleton rehabilitation robots, etc. It has brought great help to industrial production and rehabilitation medical treatment [1,2]. However, the traditional rigid mobile robot is limited by the size of the connecting rod and connecting mechanism, so its flexibility is not enough. In order to improve flexibility, it is necessary to improve the degree of freedom of the structure or match it with complex control algorithms, which increases the research and development costs and limits the scope of application [3,4]. Moreover, for complex road conditions with obstacles, the mobile robot may not be able to pass the specified locomotion path smoothing, thus limiting its corresponding functions. Moreover, due to the structural limitations of rigid links, mobile robots such as exoskeleton rehabilitation robots



have poor environmental adaptability, safety and reliability [5,6]. Therefore, in order to overcome the shortcomings of the rigid mobile robot and improve the interaction ability with the multi-environment, the robot with high security and reliability and low price has a huge research prospect.

Natural creatures, such as animals and plants, provide a lot of design inspiration to human beings, which greatly promotes the development of soft mobile robots [7,8], and have become the current research hotspot and development frontier. The soft mobile robot usually consists of a soft actuator and a rigid bracket. Among them, the actuator composed of flexible materials can carry out large deformation movement, which is driven by a variety of modes, such as pneumatic [9], combustion drive [10], shape memory alloy (SMA) [11] and so on. However, the risk of combustion drive is high, and the displacement is not easy to control. SMA driven by electricity has the disadvantage of a low power ratio. Due to the characteristics of large deformation of soft materials, the energy utilization rate of pneumatic actuator is also a problem that needs to be paid attention to. In contrast, pneumatic driven actuators have the characteristics of fast response speed, high power density, low cost and easy control [12,13]. Therefore, pneumatic drive is still widely used in the field of soft robots. By changing the position and size of the air chamber structure and volume of the actuator, different motion modes can be formed, and the soft mobile robot can be driven to complete specific locomotion, such as crawling, turning locomotion of animals [14,15]. For example, the caterpillar crawls in the bow [16] and the earthworm wriggle [17], which have strong maneuvering ability and strong adaptability to the environment. Due to the cushioning effect of flexible materials, which further improves the safety in the operation process, soft mobile robots can be widely used in post-disaster rescue, pipeline detection and other fields [18,19].

The main body of a pneumatic driven actuator is mainly composed of soft materials (such as rubber, silicone, etc.), which has great flexibility. Its own body shape and movement can better adapt to the external environment, with a wide range of applications [20,21]. With the rapid development of bionics, many actuators with novel structures have been fully studied. Onal et al. [22] developed a snake-like soft robot, which consists of four bidirectional flexible parts. But it does not have autonomous steering capability. Rafsanjani et al. [23] proposed a snake-like soft robot based on the texture structure of snake scale skin and the Kirigami principle. Must et al. [24] designed a soft robot with variable stiffness to mimic the tentacles according to the internal penetration principle of plant tentacles, which can complete the extension locomotion along the rods. Zhang et al. [25] introduced an inchworm-like soft actuator, which has two functions of crawling and climbing and can complete locomotion in both terrestrial and underwater environments, Jeong et al. [26] designed a bellows shaped soft actuator with multiple air chambers according to the structure of the human hand, which has multiple bending functions and a high degree of fit with it. Although the soft actuator with bellows multi-chamber structure can improve its bending performance to a certain extent, due to the low stiffness of the soft material, it will lead to the radial main elongation ratio λ_1 greater than 1, which will consume part of the work done by the air pressure on SPA. Therefore, in order to improve energy efficiency and reduce the difficulty of modeling and prototyping, many scholars have proposed the fiber-reinforced soft actuator. Singh et al. [27] developed a cylindrical fiber reinforced soft pneumatic actuator (FRSPA) with a single air chamber, which reduces the number of air chambers and has a torsion effect and can be used in rehabilitation medical scenarios. Wirekoh et al. [28] and Rakhtala et al. [29] proposed rehabilitation gloves with FRSPA structure, both of which have good bending effects and practical application. However, the effect of the number of fibers turns on the bending property and needs further study.

The soft mobile robot, composed of a soft actuator, can enter narrow spaces that traditional robots cannot enter by virtue of its good bending performance and flexibility. Therefore, it can be widely used

in various engineering scenarios [30]. Chen et al. [8] produced a three-segment soft mobile robot with multiple air chambers, which has two motion modes: straight line and cross. The coordinated motion of each segment enables it to complete obstacle avoidance. However, the linear motion mode does not have the ability to turn or lateral crawl. Wang et al. [14] proposed a type of Ω multi-segment soft crawling robot modeled on inchworm, in which all segments complete forward and turn locomotion, and the locomotion of each locomotion can be independent of each other. The maximum forward speed is 5.11 mm/s and the maximum turning speed is 0.76 degree/s. But whether it has an obstacle avoidance function is not given in the motion mechanism. Yeh et al. [19] designed a soft crawling robot that could be used for pipeline inspection. It could move quickly in horizontal and vertical pipelines, but its compact structure could not make it move in other complex terrains. Shepherd and Tolley et al. [31,32] first proposed a quadruped pneumatic crawling robot without rigid connectors, which has two gaits, wavy and crawling, and can complete corresponding locomotion across obstacles according to the inflated state of each foot. But lack of kinematic mechanism analysis. Fei et al. [33] built a modular soft mobile robot composed of three deformable spherical units. By controlling the input pressure, the diameter of each module is changed so as to complete the locomotion in narrow or low complex channels. But the same as [8], it cannot turn and walk laterally. Compared to our previous proposal of a soft actuator with a multi-segment corrugated tube structure [34]. The multi-segment three-dimensional-fiber reinforced soft pneumatic actuator (3D-FRSPA) designed in this paper, while ensuring the same bending and axial elongation functions, simplifies its internal air chamber structure from a complex corrugated tube structure to a cylindrical air chamber structure. The 3D-FRSPA is wrapped with double-sided symmetrical fiber lines on the outside, limiting radial deformation and improving the deformation ability of bending and axial elongation so that the energy generated by the input air pressure is more used for the deformation of the actuator.

The main contributions of this paper are as follows: (1) Based on the inspiration of bionics, a novel multi-segment 3D-FRSPA with independent and omni-directional bending performance is designed. (2) The theoretical models of statics and kinematics for single and multi-segments 3D- FRSPA are given, respectively. (3) The effects of fibers, length of the air chamber and segment, radius of circular air chamber of 3D-FRSPA on the mechanical performance of the actuator are discussed, respectively. (4) The crawling soft robots composed of three-segment of 3D-FRSPAs are numerically studied, and the simulation results of crawling in the different spaces are given, respectively.

The remainder of the paper is organized as follows. In Section 2, the new multi-segment 3D-FRSPA is introduced. Section 3 deduces the theoretical models of statics and kinematics of 3D-FRSPA in detail. In Section 4, a detailed parameter analysis is carried out. The bending behavior under different parameters is discussed. In Section 5, the application simulation of crawling of soft mobile robots in different environments is given. Finally, Sections 6 and 7 contain the discussions, main conclusions, and outlook for this paper.

2 Design of 3D-FRSPA

In real nature or world, there are a large number of creatures with multi-joint lower limb structures, such as snow crab claw's lower limb [35] and human fingers [36] structures. The above structures have three or more segments, and each segment can rotate freely to achieve the functions of crawling or grasping objects and other functions. Inspired by the above biological structure and its related sizes, this paper proposes a multi-segment 3D-FRSPA as shown in Fig. 1. It is composed of three segments (Fig. 1a). The structural sizes of each segment can be adjusted according to the actual application scenario, and the sizes between segments are independent of each other. Each segment is connected

with a bellow-shaped joint (Fig. 1b). By adjusting the number of bellow bulges of the joint, the bending trajectory of SPA can be changed [34]. In the non-joint area, that is, in the bending part of the soft body, two equal length and symmetrical fiber lines are wound along the axis (Fig. 1e). Thus, 3D-FRSPA is formed. FRSPA makes up for the fact that the structure of multi-joint organisms in the real world can only bend in one direction and cannot extend axially. The application scope of the crawling robot is greatly expanded by reasonably controlling the input air pressure and selecting the appropriate air chamber position. Such as crossing narrow channels, etc. By winding the fibers, the radial deformation is limited, the stiffness of the 3D-FRSPA is increased, the bending angle is larger, and the utilization efficiency of air pressure energy is improved.



Figure 1: Numerical modeling of the proposed FRSPA. (a) Geometric details of the FRSPA. (b) Geometric sizes of connecting joints. (c) Cross section geometric details of FRSPA, where r_1 represents the radius of circular air chamber. (d) Perspective view of circular air chamber (The fibers are not shown). (e) Structure of fiber wound outside the actuator. (f) Bending trajectory and stress cloud of the FRSPA, involving 31004 elements. (g) Results of fiber stress distribution of the soft actuator bending 360° under input pressure is $P_{in} = 0.442$ MPa

Fig. 1 shows the numerical model of 3D-FRSPA. Its actuating principle is as follows: different air chambers are inflated to produce bending or elongation deformation. To simplify the description, the four air chambers are labeled as Ω_1 , Ω_2 , Ω_3 and Ω_4 , respectively (Fig. 1c). The main bending configurations of different air chambers can be divided into the following categories: (1) When $\Omega_1 \& \Omega_2$ are inflated at the same time, the actuator will produce bending deformation in the direction of $\Omega_3 \& \Omega_4$. (2) In particular, when Ω_1 inflates, or $\Omega_1 \& \Omega_2 \& \Omega_4$ inflates simultaneously, it will bend towards the position of Ω_3 . However, in the two cases, the bending angle is different because the axial elongation will occur when the air chamber in the opposite position is inflated. (3) When $\Omega_1 \& \Omega_3$, or $\Omega_1 \& \Omega_2 \& \Omega_3 \& \Omega_4$ are inflated, the actuator will produce axial elongation. (4) When the air pressure P_{in} of $\Omega_1 \& \Omega_2$ is greater than P_{in} of $\Omega_3 \& \Omega_4$, the actuator will bend in the direction of $\Omega_3 \& \Omega_4$, and the actuator length will extend. Therefore, when the same pressure P_{in} are applied, the actuator in a single segment has eight bending modes and one stretching mode. For the three-segment structure in Fig. 1a, the total types of bending configurations are $9^3 = 729$. Obviously, when the pressure P_{in} are different, the number of deformation configuration will far exceed 729.

In Fig. 1a, each segment are denoted as H_1 , H_2 and H_3 , and its length are S_1 , S_2 and S_3 , respectively. The number of winding fiber is n_1 , n_2 and n_3 . LS_{12} , LS_{23} are expressed as the length of the bellows joint, respectively. And its other sizes are shown in Fig. 1b. The cross section of FRSPA is a square with a fillet radius of $r_2 = 2$ mm and the side length of it is 2a. The radius of the air chamber is r_1 . The spacing is 2b, and the spacing between the front and end bottom of the air chamber and the segmented soft section surface is c. The geometric dimensions of FRSPA are respectively set as: $S_1 = 40$ mm, $S_2 = S_3 = 50$ mm, $n_1 = 24$, $n_2 = n_3 = 30$, $LS_{12} = 10$ mm, $LS_{23} = 6$ mm, a = 7.5 mm, b = 4 mm, c = 5 mm, $r_1 = 2$ mm. The bending stress distribution diagram of the actuator in Fig. 1f and the stress distribution diagram of the fiber in Fig. 1g can be obtained by applying equal air pressure P_{in} to $\Omega_1 \& \Omega_2$, respectively. With the increase of P_{in} , all segments of FRSPA are bent and the bending angle becomes larger. When $P_{in} = 0.442$ MPa, the total bending angle of FRSPA reaches 360° .

3 Theoretical Model

In this section, the theoretical expression of the proposed 3D-FRSPA structure is derived, and the statics kinematics are analyzed. The bending models of single- and multi-segment 3D-FRSPA are presented, the accuracy of the models is verified, and the errors of different mesh structures are analyzed to ensure calculation accuracy and improve calculation efficiency.

3.1 Bending Theoretical Model

Since the bending motion of FRSPA is a nonlinear large deformation caused by P_{in} , compared with the rigid body model, the deformation motion and mechanical properties can be obtained more accurately from energy. The body of the 3D-FRSPA uses the hyper-elastic material Elastosil M4601 [35]. For the investigation of the bending theoretical model of single segment 3D-FRSPA, the second-order Yeoh hyperelastic theory [37] is used as this work's strain energy density function. Since silicon rubber is incompressible ($\lambda_1 \lambda_2 \lambda_3 = 1$), W can be simplified as follows:

$$W = C_{10} \left(I_1 - 3 \right) + C_{20} \left(I_1 - 3 \right)^3 \tag{1}$$

where I_1 is the first invariant of stress tensor. C_{10} and C_{20} are independent elastic coefficients. According to [37], let $C_{10} = 0.11$, $C_{20} = 0.02$.

$$I_1 = \lambda_1^2 + \lambda_2^2 + \lambda_3^2 = 1$$
(2)

where λ_1 , λ_2 and λ_3 are the axial, circumferential, and radial principal stretch ratios, respectively.

It can be seen from Section 2 that the 3D-FRSPA of the S-th (i = 1, 2, 3) single segment still have multiple bending configurations. To simplify the complexity of bending deformation analysis, the bending configuration schematic of the FRSPA side of a single segment is taken as shown in Fig. 2. Fig. 2a shows the bending configuration of the side when a uniform P_{in} is applied to Ω_1 and Ω_2 in a free space environment. Since the fiber in Fig. 1g limits the radial deformation of FRSPA, then $\lambda_2 = 1$ [38], the actuator only extends axially and bends in a certain direction. The right end face is fixed. Due to the anisotropy of Elastosil M4601, the free end bends downward under the action of positive pressure P_{in} . The inflated actuator bends towards the side of the uninflated chamber with a radius R and angle θ . Take the first part- II_1 as the analysis object. The arcs AB and EF represent the lengths of the stretched upper side and the unstretched lower side of the actuator, respectively. They are $S_1, S_1 + \Delta S_1$, respectively. Furthermore, the length of arc AB remains unchanged during bending deformation. The arc CD is the neutral layer of the actuator with a length of $S_1 + s_1$. Fig. 2b shows the moment balance schematic of the free end of the actuator. M_{Pin} represents the driving moment of the air pressure on the actuator relative to the $\widehat{E}F$ on the bottom side. Since FRSPA is made of soft silica gel, the elasticity of silicon rubber itself will produce a resistance moment M_R under the action of air pressure P_{in} . When the FRSPA reaches the stable bending state, the torque reaches the equilibrium state.



Figure 2: Bending theoretical model of the designed FRSPA. (a) The actuator bends when the Ω_1 and Ω_2 air chambers are inflated simultaneously. (b) Moments equilibrium at the end of the actuator

Due to the incompressibility of silicon rubber material, the total volume of silica gel remains unchanged before and after inflation. V_1 is set as the total volume of silicon rubber after deformation, which can be obtained according to Fig. 2a.

$$V_1 = \left[(2a - 2r_2)^2 + 4(2a - 2r_2)r_2 + \pi r_2^2 \right] S_1 - 4\pi r_1^2 (S_1 - 2c)$$
(3)

When $P_{in} > 0$, the size change of the actuator in the axial direction can be expressed as

$$\lambda_1 = \frac{L_{\widehat{CD}}}{L_{\widehat{EF}}} = 1 + \frac{a}{R} = 1 + \frac{2a\theta}{S_1} \tag{4}$$

where $L_{\widehat{CD}}$ and $L_{\widehat{EF}}$ are the length of arcs \widehat{CD} and \widehat{EF} . Because $\lambda_2 = 1$, then can get $\lambda_3 = \frac{1}{\lambda_1 \lambda_2} = \frac{S_1}{S_1 + 2a\theta}$.

After inflation, the chamber $\Omega_1 \& \Omega_2$ are bent and deformed. $\Omega_3 \& \Omega_4$ bend under the drive of $\Omega_1 \& \Omega_2$ but do not change the volume of the air chamber. Therefore, the total volume V of deformed FRSPA (Fig. 2a) is

$$V = \theta \left[a\lambda_3 \left(R + a + a\lambda_2 \right)^2 - a \left(\lambda_3 - 1 \right) \left(R + a \right)^2 - aR^2 \right] - r_2^2 \lambda_2 \lambda_3 \left(4 - \pi \right)$$
(5)

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Then, we can get the volume of the air chamber $\Omega_1 \& \Omega_2$ after inflation deformation under the P_{in} effect, we can set V_2 and the is described as

$$V_2 = V - V_1 - V_3 \tag{6}$$

where V_3 represent the volume of air chamber $\Omega_3 \& \Omega_4$ which not affected by P_{in} .

$$V_3 = 2\pi r_1^2 \left(S_1 - 2c \right) \lambda_1 \tag{7}$$

Since gravity has little effect on the 3D actuator relative to the input air pressure P_{in} . The size of part- II_1 is small, and the fibers improve the overall rigidity of 3D-FRSPA. Therefore, the influence of the self-weight on the bending result is ignored when the theoretical modal of the single segment actuator is derived. If there is no external force, the work done by P_{in} can be completely converted into bending deformation or axial elongation of the actuator. According to the principle of virtual work, we can get

$$P_{in}dV_2 = V_1dW \tag{8}$$

Since the bending angle θ is a dependent variable, the explicit expressions of $dV_2/d\theta$ and $dW/d\theta$ can be obtained according to the Eqs. (1)–(7), which can be substituted into the Eq. (8) to obtain the theoretical expression between the P_{in} and θ_1 for part- II_1 .

$$\theta_1 = f(n_1, S_1, P_{in}, a, b, c, r_1, r_2, C_{10}, C_{20})$$
(9)

For multi-segment FRSPA, the joint connected is a non-air chamber connection part without rotation function. Therefore, when the condition of the position of P_{in} is the same, the θ or elongation length of each segment can be calculated only according to the input air pressure and structure size of each segment, so as to obtain the total bending angle of 3D-FRSPA. Then the total bending angle can be set to $\Theta = \sum_{i=1}^{3} \theta_i$, where i = 1, 2, 3.

3.2 Kinematics Analysis

3D-FRSPA is composed of three segments, and each segment has different bending configurations. Therefore, the total end pose expression is the vector product of each segment. According to [35], the explicit expression of the end position and posture of the three-segment 3D-FRSPA proposed in this paper is

$${}^{O}\mathbf{T}_{O3} = {}^{O1}\mathbf{T}_{O3} = {}^{O1}\mathbf{T}_{O2} \times {}^{O2}\mathbf{T}_{O3}$$
(10)

$$\boldsymbol{\Theta} = \sum_{i=1}^{3} \boldsymbol{\theta}_{i}^{j} \tag{11}$$

where i = 1, 2, 3 and j = 1, 2, 3, 4 indicate the number of each segment (part- II_1 , II_2 and II_3) and each direction of the single segment actuator, respectively. When j = 5, it means that $\Omega_1 \& \Omega_2 \& \Omega_3 \& \Omega_4$ of this segment are inflated simultaneously, that is, axial elongation.

In Eqs. (10) and (11), ${}^{o}\mathbf{T}_{o3}$ denotes the the end pose of FRSPA, $\boldsymbol{\Theta}$ represents the total bending angle. In particular, if the aeration position of each segment is different, the final bending angle is different. Thus, the forward kinematic expression of FRSPA can be obtained.

For the soft crawling robot, it belongs to multiple lower limbs fixed on a rigid support, and the position and speed of the initial end face of each soft body are the same as that of the rigid support,

while the position and initial speed of the rigid support are known, so the inverse kinematics expression of 3D-FRSPA is not given here.

3.3 Finite Element Analysis of Theoretical Model

The consistency of the theoretical model and numerical calculation can ensure the rationality of the results of the FRSPA designed, and can provide accurate data reference for the follow-up locomotion mechanism of the soft crawling robot. In this section, the FRSPA of single and multisegment FRSPA in Fig. 1a will be verified numerically. The finite element model of the FRSPA was established and analyzed by ABAQUS/Standard. For the soft part of the model, the element type was selected as C3D10H, the material constant was set as $C_{10} = 0.11$ MPa, $C_{20} = 0.02$ MPa, and the density was 1130e-12 mg/mm³ of Elastosil M4601 [38,39]. The second order beam element (B32) was selected for the double-wound fiber. The Poisson's ratio, Young's modulus and density of Kevlar fiber were 0.36, 31067 MPa and 786e-12 mg/mm³, respectively [38,39]. The section radius of the beam element is set to 0.0889 mm [40]. Tie is used as the interaction relationship between the fiber and the soft in ABAQUS, and the joint without rotation function is not wrapped with fiber. Take Fig. 1a as the analysis object of multi-segment FRSPA, where part- II_1 as a single segment. Then, single- and multisegment analysis will be conducted. For single segment of FRSPA, the same pressure P_{in} is applied to the air chamber $\Omega_1 \& \Omega_2$ from 0 to 0.5 MPa as an increasing sequence of 0.05 MPa. For multi-segment, 0 to 0.4 MPa is applied, and other settings are the same as those for single segment. It can be seen from Fig. 3 that the numerical simulation results of single and multi-segments are in good agreement with the theoretical models. In particular, the results of the comparison and verification of each segment in the multi-segment can also be well matched.



Figure 3: (Continued)



Figure 3: Verification of ABAQUS results and theoretical values of FRSPA. The increased step size of P_{in} is 0.05 MPa. (a) Single segment (from 0 to 0.5 MPa). (b) Multi-segment (from 0 to 0.4 MPa). (c) Part- II_1 , II_2 and II_3 (from 0 to 0.4 MPa and without joints)

The structure of FRSPA has a high degree of nonlinearity, so it is necessary to consider the calculation efficiency while ensuring the accuracy of numerical calculation. Due to the element properties of C3D10H and B32 in ABAQUS, it is not possible to partition the number of elements specified for FRSPA accurately. Therefore, the method of controlling the number of elements by adjusting the mesh boundary size is adopted. The specific element boundary size and the total number of elements are shown in Table 1, where l_{soft-m} and $l_{fibre-m}$ represent the mesh boundary size of soft and fiber, respectively. NeS_{soft} and NeS_{fibre} denote the total number of elements of single segment soft and two fibers, respectively. There is a one-to-one correspondence between l_{soft-m} and $l_{fibre-m}$, and only the size of the mesh boundary of l_{soft-m} is indicated later. They increase from 1 and 0.3 mm, respectively, and the step size increases from 0.5 and 0.2 mm, respectively. Meanwhile, in order to ensure accuracy, the maximum values are set to 5 and 1.9 mm, respectively.

l_{soft-m} (mm)	$l_{\it fibre-m}~(mm)$	NeS_{soft}	NeS_{fibre}	NeM_{soft}	NeM_{fibre}
1	0.3	58938	11278	\	\
1.5	0.5	18972	6720	57056	18776
2	0.7	10768	5040	35518	14114
2.5	0.9	7358	3840	25020	10572
3	1.1	5781	3120	18672	8736
3.5	1.3	3847	2640	14105	7392
4	1.5	3268	2162	11571	6054
4.5	1.7	2686	2158	9612	6042
5	1.9	2441	1920	8852	5376

Table 1: Mesh boundary size and total number of elements for single and multi-segments of FRSPA

The experimental setup involves an Intel(R) Core(TM) i5-6500 CPU with 16 GB RAM. $l_{soft-m} = 1 \text{ mm}$ and $l_{fibre-m} = 0.3 \text{ mm}$ were used as reference results for accuracy verification. Define the *error* expression as

$$error = \left| \frac{l_{soft-m} - l_{\min}}{l_{soft-m}} \right| \times 100\%$$
(12)

where l_{\min} denote the minimum mesh boundary size of the soft and fiber.

The computation time of error analysis cloud chart and boundary size of each grid is shown in Figs. 4 and 5, respectively. It can be seen from Fig. 4 that for single and multi-segments of FRSPA, the error will gradually increase with the increase of mesh boundary size and input pressure P_{in} . In particular, when $P_{in} = 0.1$ MPa and $l_{soft-m} = 3.5$ mm for a single segment of FRSPA, a large error = 8.45% is presented, but when $P_{in} = 0.4$ MPa and $l_{soft-m} = 5$ mm for a multi-segment of FRSPA, a small error = 1.91% is presented. The reason is that the nonlinear effect leads to the fluctuation of bending accuracy. It can be seen from Fig. 5 that when the grid boundary size is $l_{soft-m} = 2.5$ mm, the calculation time decreases significantly. And when $l_{soft-m} = 4$ mm and $l_{fibre-m} = 1.5$ mm, the error is at a low level, which makes the precision and calculation efficiency achieve a good balance. Therefore, the mesh structure adopted in the subsequent calculation in this paper is: $l_{soft-m} = 4$ mm and $l_{fibre-m} = 1.5$ mm.



Figure 4: Mesh error analysis nephogram of FRSPA. The input air pressure P_{in} increases from 0 to 0.4 MPa at the step = 0.05 MPa. The boundary size of the fiber correspond to the size of the soft and are omitted. (a) Single segment. (b) Multi-segment



Figure 5: Computation time of FRSPA for different mesh sizes. (a) Single segment. (b) Multi-segment

4 Mechanical Performances of the FRSPA

The fiber reinforced soft actuator are sensitive to structural parameters. In order to better characterize the mechanical properties of FRSPA, this section systematically analyzes the fibers, air chamber shape and segment length, and circular air chamber radius.

4.1 Fibers

In this Section, the influence of fibers with different turns on the bending performance of 3D-FRSPA is first studied. Three different types of fiber winding methods for SPA were designed, as shown in Fig. 6a. They are non wound fibers, symmetrically wound for 30 turns in all three segments, and the number of turns in each segment is taken as $n_1 = 30$, $n_2 = 20$, $n_3 = 10$. The other structural parameters remain unchanged, and the total length of 3D-FRSPA is 162 mm. Fixing the rightmost end of the three types of SPAs and applying equal air pressure to the air chamber $\Omega_1 \& \Omega_2$ to bend the SPA, studying the effect of bending performance. And apply equal air pressure to all chambers $\Omega_1 \& \Omega_2 \& \Omega_3 \& \Omega_4$ simultaneously to study the effect of different fibers on their axial elongation.



Figure 6: (Continued)



Figure 6: Numerical results of the 3D-FRSPA of different fibers. (a) Geometric details of the 3D-FRSPA. Cloud diagram of bending configuration and stress distribution when applying different P_{in} (fibers are not shown): (b) $n_1 = n_2 = n_3 = 0$; (c) $n_1 = n_2 = n_3 = 30$; (d) $n_1 = 30$, $n_2 = 20$, $n_3 = 10$. Axial elongation: (e) $P_{in} = 0.1$ MPa; (f) $P_{in} = 0.2$ MPa

It can be clearly seen from Figs. 6b–6d that the SPA wrapped with fibers has a larger bending angle compared to the SPA without fiber winding when P_{in} is applied at the same time. When three types of SPA reach 360° bending, the required P_{in} values are 0.354, 0.404 and 0.402 MPa, respectively. Therefore, when P_{in} is too large, due to the limitation of fiber deformation, more air pressure is needed to achieve 360° bending, but for crawling motion, SPA is not required to achieve 360° bending. When the input air pressure is not too high, the fiber wrapped SPA has better bending performance.

For the axial elongation analysis of SPA, it can be seen from Figs. 6e and 6f that fiber reinforced SPA has a greater elongation than non fiber wrapped SPA when the same P_{in} is applied. When the number of fiber turns in all three segments is 30, the elongation is greater than that of SPA with three segments of turns of 30, 20, and 10, respectively. However, the radial deformation of SPA without fibers will gradually increase with increasing air pressure, which will lead to additional consideration of lateral spatial interference in subsequent crawling movements. Fiber reinforced SPA does not need to consider this problem. Therefore, fiber wound SPA has better axial elongation ability. In summary, from the quantitative comparison results in Fig. 6, it can be seen that fiber-reinforced SPA has better bending and axial deformation ability.

4.2 Pneumatic Chamber and Segment Length

In this section, we will study the influence of the pneumatic chamber and the segment length on the bending performance of the multi-segment FRSPA. Firstly, the research object is the chamber length, $L_c = 35$, 40 and 45 mm, respectively. The principle of unique variable is adopted. When only the length of the pneumatic chamber is changed, the 3D-FRSPA length is L = 162 mm, then the c = 7.5, 5 and 2.5 mm. Other geometric sizes are: $S_1 = S_2 = S_3 = 50$ mm, $n_1 = n_2 = n_3 = 30$, $L_{12} = L_{23} = 6$ mm, a = 7.5 mm, b = 4 mm, c = 5 mm, $r_1 = 2$ mm. As shown in Fig. 7a. P_{in} is applied to $\Omega_1 \& \Omega_2$ to

obtain the mechanical properties in the corresponding Z and Y directions as shown in Figs. 7b–7d. In order to facilitate the display, all fibers are not displayed but are involved in the simulation calculation.



Figure 7: Numerical results of the 3D-FRSPA of different of pneumatic chamber. (a) Geometric details of the 3D-FRSPA. Cloud diagram of bending configuration and stress distribution when applying different P_{in} (fibers are not shown): (b) $L_c = 35$ mm; (c) $L_c = 40$ mm; (d) $L_c = 45$ mm

It can be seen from Figs. 7b–7d that when the same input air pressure $P_{in} = 0.1$ and 0.2 MPa, the larger the length of the pneumatic chamber, the larger the bending angle, the more uniform the stress distribution and the smaller the stress value, which are mainly distributed near the pneumatic chamber $\Omega_1 \& \Omega_2$. When the three multi-segment FRSPAs reach 360° bending, the required air pressure P_{in} are 0.446, 0.403 and 0.359 MPa, respectively. Therefore, an appropriate increase in the length of the pneumatic chamber can increase the internal area under the action of P_{in} . It also can improve the bending performance of FRSPA, and make the expansion deformation more uniform. However, L_c should not be too large, that is, *c* should not be too small, otherwise it may cause the first end face of FRSPA to burst.

In order to improve the applicability and generality of the 3D-FRSPA, different from the segment lengths used for different air chamber lengths, the segment lengths were taken as $S_1 = S_2 = S_3 = 45$, 55 and 65 mm, respectively. The corresponding turns of fiber are $n_1 = n_2 = n_3 = 27$, 33 and 39, respectively. The structure diagram is shown in Fig. 8a. If a single rotary joint is used, the length of

multi-segment FRSPA is L = 147 mm, 177 mm and 207 mm, respectively. Other geometric dimensions: $L_{12} = L_{23} = 6$ mm, a = 7.5 mm, b = 4 mm, c = 5 mm, $r_1 = 2$ mm.



Figure 8: Numerical results of the 3D-FRSPA of different length of segment. (a) Geometric details of the 3D-FRSPA. Cloud diagram of bending configuration and stress distribution when applying different P_{in} (fibers are not shown): (b) L = 147 mm; (c) L = 177 mm; (d) L = 207 mm

It can be seen from the numerical results that when the same P_{in} is applied, the longer the segment length, the better the mechanical properties of the multi-segment FRSPA, and the more uniform the stress distribution. When 360° bending is reached, the required P_{in} are 0.465, 0.385 and 0.303 MPa, respectively. However, the increase in segment length will lead to an increase of structure size, so it is necessary to choose a reasonable configuration to obtain the best bending performance.

4.3 Radius of the Circular Chamber

Next, we investigate the influence of the radius of $\Omega_1 \& \Omega_2$ on the bending angle and internal stress distribution. It can be seen from Section 4.2 that the larger the area of input pressure P_{in} , the better the mechanical properties, but the smaller the c, the more unstable the structure will appear. Similarly, the overall external sizes remain the same and a more suitable r_1 is sought. Therefore, this section investigates the bending behavior of actuators with $r_1 = 1.5$, 2 and 2.5 mm, respectively. Other geometric dimensions are: $S_1 = S_2 = S_3 = 50 \text{ mm}$, $n_1 = n_2 = n_3 = 30$, $L_{12} = L_{23} = 6 \text{ mm}$, a = 7.5 mm, b = 3 mm, and c = 5 mm, as shown in Fig. 9a. The other settings remain the same as in Section 4.2.

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The step size was increased by 0.05 MPa, and the P_{in} was gradually increased from 0 to obtain the analysis results shown in Figs. 9b–9e.

It can be seen from Fig. 9 that the larger the radius r_1 of the air chamber is, the greater the bending ability of FRSPA can be significantly improved. When the bending reaches 360°, the pressure required for the FRSPA with a chamber radius of $r_1 = 1.5$, 2 and 2.5 mm are $P_{in} = 0.526$, 0.403 and 0.301 MPa, respectively. Figs. 9c–9e show the internal chamber deformation and overall stress distribution of different r_1 when $P_{in} = 0.2$ MPa is applied. As can be seen from the above figures, the larger r_1 is, the greater the degree of deformation is, the greater the surface stress of the air chamber is, and the greater the stress of the externally wound fiber caused by extrusion is. Therefore, selecting a suitable r_1 , such as $r_1 = 2$ mm, can make the bending performance and internal stress distribution of FRSPA reach a good balance.



Figure 9: Numerical results of the 3D-FRSPA of different radius of the circular chamber. (a) Schematic of geometric details for $r_1 = 1.5$, 2 and 2.5 mm. (b) FEA modeling of FRSPA with the different P_{in} . The deformation diagram and stress distribution nephogram inside the air chamber after $P_{in} = 0.2$ MPa is applied to $\Omega_1 \& \Omega_2$ of different radius: (c) $r_1 = 1.5$ mm; (d) $r_1 = 2$ mm; (e) $r_1 = 2.5$ mm

5 Crawling Analysis in Different Environments

In the above sections, the theoretical model of 3D-FRSPA proposed was verified, and the appropriate mesh structure was proposed. At the same time, the parameters of single and multi-segments were analyzed in detail, and the static and dynamic analysis was carried out. According to the conclusions of the previous sections, this section will design two types of crawling locomotion under different road environments to verify the advantages and application of the soft crawling robot proposed in this paper.

5.1 Crawling on a Flat Road

The Section Introduction gives the research motivation and importance of flexible and multifunctional soft crawling robots. This section proposes a soft crawling robot composed of rigid support with a lightweight optimization design and eight lower limbs. According to Eq. (11) and the locomotion rule of biological gait, the horizontal and vertical spacing is set as $l_{EF} = 50$ mm and $l_{AE} = 130$ mm, respectively, and the structure is shown in Fig. 10. A single multi-segment FRSPA consists of three sections, with a total length of 166 mm. Figs. 10a and 10b show the geometric details of the soft lower limb and the rigid support and the schematic diagram of its structure. The eight three-segment FRSPA are numbered as A, B, ..., D. Fig. 10c shows a flat road with a size of 600 mm × 300 mm, placed horizontally, used to simulate the barrier-free crawling simulation of a soft crawling robot.







Figure 10: A soft crawling robot for crawling in different road locomotion. (a) Three-segments FRSPA structure and rigid fixture. (b) A soft crawling actuator consisting of eight identical FRSPAs. (c) Flat road. Irregular road surface with obstructions: (d) vertical view; (e) oblique view

The soft crawling robot is composed of multi-segment FRSPAs, and the control of each segment and each air chamber is independent of each other. In order to realize crawling along the length direction on the flat road shown in Fig. 10e, it is necessary to set a suitable crawling gait. Fig. 11 shows the time variation of pressure applied by each air chamber in the crawling process of the soft crawling robot. According to the air chamber position diagram in Fig. 1c and Fig. 10b, there are a total of $8 \times 3 \times 4 = 96$ air chambers in eight three-segment FRSPAs. 1 to 96 are defined in sequence, representing the number of each air chamber. For example, *Label*-22 and 91 stand for FRSPA-B-*II*₃- Ω_2 and FRSPA-H-*II*₂- Ω_3 , respectively.



Figure 11: Minimum input of P_{in} required for each segment of FRSPA when crawling on a flat road (MPa). The abscissa indicates that every 1 s on the horizontal axis represents the completion of a crawl, and the total time is 9 s to complete the entire distance. The total number of *Labels* in the ordinate ranges from 1 to 96, which in turn represents the number of each air chamber in each segment of each FRSPA

The distance between the soft crawling robot and the starting point on the flat road and the boundary is 45 mm, and the total length of the rigid support is 210 mm. At the same time, according to Eqs. (9)–(11), in order to improve the crawling efficiency, that is, to complete the whole process with the least number of steps, and ensure the stability of the whole crawling process, the contact

with the previous FRSPA is not allowed. Therefore, in the process of each creeping step, the input pressure of part- $II_1 \& II_2$ of FRSPA in each multi-segment can be determined as $P_{in} = 0.18$ MPa, then the distance of part- II_3 moving along the x direction is 39 mm. The FRSPA-ACFH and BDEG are made to bend and move forward alternately. When pressure is released, the soft climbs the actuator without changing the position of contact with the ground, thus driving the whole actuator forward. Since FRSPA bends with constant curvature deformation, alternating loads can be applied repeatedly to achieve continuous motion.

Fig. 12 shows the crawling process of the actuator on a flat road. And its sub processes are: (a) Positional relationship. (b) Crawling sequence: (1) 0–1 s, the chamber FRSPA-ACF- II_1 - $\Omega_3 \& \Omega_4$ and FRSPA-ACFH- II_2 - $\Omega_1 \& \Omega_2$ and FRSPA-ACFH- II_2 - $\Omega_1 \& \Omega_2$ were charged with the same pressure $P_{in} = 0.18$ MPa, and the position at the bottom bend is 39 mm; (2) 1–2 s, chamber FRSPA-ACFH- II_1 - $\Omega_3 \& \Omega_4$ and FRSPA-ACFH- II_2 - $\Omega_1 \& \Omega_2$ release pressure, recover to the initial state, and drive the soft body to crawl the actuator 39 mm in the *x* direction. At the same time, the chamber FRSPA-BDEG- II_1 - $\Omega_3 \& \Omega_4$ and FRSPA-BDEG- II_2 - $\Omega_1 \& \Omega_2$ and FRSPA-BDEG- II_2 - $\Omega_1 \& \Omega_2$ are injected with the same pressure $P_{in} = 0.18$ MPa; (3) 2–8 s, all the corresponding air chamber are inflated alternately to make the soft crawling actuator crawl along the *x* direction with a step size of 39 mm; (4) 8–9 s, release all the air and reach the final position.



Figure 12: Numerical investigation of the process of crawling on a flat road for soft crawling robot. (Unit: mm)

A specific load is applied at the specified position to control the step size, thus completing the crawling simulation of the specified distance. Table 2 shows the forward distance of each moving step during the crawling process. The actuator crawls 312 mm along the x direction to reach the end position.

Time of crawling sequence (s)	The movement distance of rigid support (mm)		
	x	У	Ζ
0	0	0	0
1	0	0	0
2	39	0	0
3	78	0	0
4	117	0	0
5	156	0	0
6	195	0	0
7	234	0	0
8	273	0	0
9	312	0	0

Table 2: The movement	distance of a sc	ft crawlin	ng actuator f	or crawling mo	tion on a flat road
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5.2 Crawling on the Road with Obstacles

In the process of horizontal crawling in Section 5.1, only part- II_1 and II_2 need to bending and deformation, while part- II_3 remains unchanged and only plays a supporting role. However, the road condition in this section is L-shaped pavement with obstacles as shown in Figs. 10d and 10e. The soft crawling actuator continues to adopt the structure of Fig. 10b. When climbing the step, three-segments need to act simultaneously to ensure its stability in the process of locomotion. When climbing the step, which increases the stability compared with the tip and surface contact in [41]. The FRSPA in the bending state will appear to be bending and have a small amount of axial elongation along the central axis. However, the elongation is small, so it can be ignored. According to the structural size of the irregular pavement, the loading mode of each air chamber in the process of locomotion as shown in Fig. 13. The total time is 32 s, which represents 32 bending and recovery motion modes.



Figure 13: Minimum input of P_{in} required for each segment of FRSPA when crawling on the road with obstacles (MPa). The abscissa indicates that every 1 s on the horizontal axis represents the completion of a crawl, and the total time is 9 s to complete the entire distance. The total number of *Labels* in the ordinate ranges from 1 to 96, which in turn represents the number of each air chamber in each segment of each FRSPA. For example, *Label*-28 and 70 represent FRSPA-C- II_1 - Ω_4 and FRSPA-F- II_3 - Ω_2 , respectively

The simulation process of a soft crawling robot crawling on the road with obstacles is shown in Fig. 14. Fig. 14 shows the steps of the climbing process in detail. Combined with the specific structural sizes of Figs. 10d and 10e, and calculated by Eqs. (9) and (10), the minimum pressure required by axial elongation and bending motion of FRSPA can be obtained. During the 0-3 s in Fig. 14b, since the step is 40 mm higher than the ground of the soft crawling robot, it is necessary to use the FRSPA-BCFG to inflate each air chamber 0.3 MPa and raise the actuator 50 mm overall (Table 3). Then, the FRSPA-BCFG is deflated, and the first two lower limbs of the soft crawling robot along the crawling direction can be placed on the step. Similarly, according to the position and space relationship, constantly adjust the size of the pressure and the position applied, and gradually realize the locomotion of the center of gravity to the direction of the step. When t = 12 s, it means that FRSPA-BCDEGH is all on the step, and the center of gravity is already above the step, so it only needs to follow the horizontal gait in Section 5.1. Since the lower step and the upper step belong to symmetrical motion, it is not shown in detail. After crossing the obstacle, the actuator finally crawls along the z positive direction. The distance between the two actuators along the z direction is $l_{AE} = 130$ mm. In order to improve the locomotion speed, the load size was set at 0.25 MPa and the displacement was 65 mm. After four gaits, the final position was reached. Table 3 shows the displacement of each time node.

Section 5.1 mainly shows the locomotion of the actuator along the length of the rigid support without side bending and elongation. On the basis of Section 5.1, Fig. 14 further discusses the axial production of the actuator and the coordinated locomotion of the side bending so as to complete more complex crawling locomotion and be more realistic.



Figure 14: Numerical investigation of the process of crawling on the road with obstacles for soft crawling robot. (Unit: mm) (a) Positional relationship. (b) Crawling sequence: (1) 0–1 s, chamber-BCFG inflated, overall rise; (2) 1–2 s, chamber-ADEH bending; (3) 2–3 s, chamber- BCFG let off air and start to climb the obstacle; (4) 3–4 s, chamber-ADEH vent, the first two actuators up the obstacle; (5) 4–14 s, repeat the above locomotion to make the actuator crawl each FRSPA of the actuator up the steps successively; (6) 14–24 s, down the obstacle; (7) 24–29 s, complete the descending step and crawl along the *z* direction; (8) 29–32 s, complete the required distance in the z direction, release all pressure, and reach the final position

Time of crawling sequence (s)	The movement distance of rigid support (mm)			
	X	У	Z	
0	0	0	0	
1	0	-50	0	
2	0	-50	0	
3	0	-13	0	
4	26	-22	0	
5	26	-48	0	
7	52	-20	0	
11	130	-50	0	
12	170	-40	0	
14	230	-40	0	
24	492	-13	0	
29	646	0	90	
32	646	0	270	

Table 3: The movement distance of soft crawling actuator for crawling motion on the road with obstacles

6 Discussions

Sections 5.1 and 5.2 introduce the eight-legged crawling robot composed of 3D-FRSPA with a three-segment structure and simulate the numerical simulation research of crawling and obstacle avoidance on flat road and L-shaped road with steps, respectively. As can be seen from Fig. 10b, the FRSPA of each segment can independently and freely bend and extend along the axial direction. Compared with the unidirectional bending actuator [9], the performance is more flexible. Compared with the aforementioned work [39], the difficulty of modeling is simplified, and the two-sided winding fibers limit the radial deformation and improve the bending performance. And compared with [9,33], a soft crawling robot has multiple crawling directions, which is more suitable for complex road conditions and practical application (Fig. 14). The actuator with segmental bending control has better flexibility and further improved obstacle avoidance compared to [14,19]. Therefore, to sum up, the soft crawling robot composed of 3D-FRSPA proposed in this paper is a deep expansion of mobile robots, especially in the complex nonlinear obstacle avoiding crawling, with better motion mechanism and application advantages.

7 Conclusions and Outlook

Inspired by the multi-segment structure of crab claw and human finger, we propose a novel multi-segment 3D-FRSPA with four symmetrical cylindrical air chambers inside which each segment can independently bend in omni-directional. The double-wound fiber structure further improves the bending performance of the actuator. Based on this, an eight-leg crawling robot is designed, which is used for crawling in different road conditions.

The theoretical models of statics kinematics of single and multi-segments 3D-FRSPA are derived, and its mechanical behavior is discussed in depth. It provides good theoretical guidance for the design of the motion mechanism of crawling locomotion. Detailed parameter analysis is carried out for single and multi-segment FRSPA, involving fibers, length of air chamber and segment, and radius of circular air chamber. The effects of different parameters on the bending properties of FRSPA are also given. It provides the best design scheme for the soft mobile robot.

The finite element numerical simulation results of the crawling locomotion on the flat road and the L-shaped road with obstacles are given, respectively. By matching the appropriate motion mechanism and combining the results of the mechanical model, the soft mobile robot can complete the corresponding crawling under different environmental conditions.

In conclusion, this work makes up for the shortcomings of existing SPAs. It introduces a novel structure of 3D-FRSPA, which further improves actuators' bending performance and expands the soft crawling robot's application range. Furthermore, future research work will design targeted prototype experiments according to parameter analysis conclusions and numerical simulation results. While ensuring flexibility, the multi-segment FRSPA designed in this paper has a larger number of air chambers and applies more different air values. The structure will be optimized and designed in the future. Based on theoretical verification, the results will be further verified through experiments to improve the practical application range and fully explore the deep application value of the pipe in curved edge pipeline detection and complex terrain detection.

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