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# Analysis of Maximum Liquid Carrying Capacity Based on Conventional Tubing Plunger Gas Lift

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## ABSTRACT

China's unconventional gas fields have a large number of low-productivity and low-efficiency wells, many of which are located in remote and environmentally harsh mountainous areas. To address the long-term stable production of these gas wells, plunger-lift technology plays an important role. In order to fully understand and accurately grasp the drainage and gas production mechanisms of plunger-lift, a mechanical model of plunger-liquid column uplift in the plunger-lift process was established, focusing on conventional plunger-lift systems and representative wellbore configurations in the Linxing region. The operating casing pressure of the plunger-lift process and the calculation method for the maximum daily fluid production rate based on the work regime with the highest fluid recovery rate were determined. For the first time, the critical flow rate method was proposed as a constraint for the maximum liquid-carrying capacity of the plunger-lift, and liquid-carrying capacity charts for conventional plunger-lift with different casing sizes were developed. The results showed that for  $2^{3/8''}$  casing plunger-lift, with a well depth of shallower than 808 m, the maximum drainage rate was  $33 \text{ m}^3/\text{d}$ ; for  $3^{1/2''}$  casing plunger-lift, with a well depth of shallower than 560 m, the maximum drainage rate was  $75.14 \text{ m}^3/\text{d}$ . This research provides a foundation for the scientific selection of plunger-lift technology and serves as a decision-making reference for developing reasonable plunger-lift work regimes.

# **KEYWORDS**

Unconventional well; conventional tubing string; plunger gas lift; critical flow rate; liquid carry-over chart version

# Nomenclature

С

Critical liquid carry velocity coefficient

# 1 Introduction

In plunger-lift, the plunger acts as a solid interface, separating the liquid above from the gas below in the wellbore, thus providing a seal to reduce liquid slippage above the plunger and prevent gas channeling below the plunger. This improves operational efficiency and gas production from the wells.

Researchers such as Foss et al. [1] proposed the first plunger-lift model based on field application experiences from the Ventura Avenue gas field. This model predicted critical plunger-lift parameters



using data from high gas-liquid ratio plunger-lift wells and calculated the minimum pressure required to ensure contact between the plunger and the liquid column. Lea [2] developed the first dynamic model for the plunger-rising phase, considering the force balance between the plunger and the liquid column and the changing velocity of the plunger ascent. He et al. [3] divided the plunger-lift process into three stages based on the study of plunger motion characteristics and casing pressure variations. They established dynamic models for each stage and analyzed the influencing factors of plungerlift, categorized as power, resistance, and volume. The impact of gas-liquid ratio, formation pressure, pipeline pressure, and well depth on production rates was also analyzed, and it was suggested to adjust casing pressure and flowback time to achieve higher production. Tang et al. [4] developed a dynamic model considering changes in oil and casing pressure, liquid accumulation, liquid fallback, and plunger resistance to describe plunger motion. Based on field test data, the characteristics of oil and casing pressure in plunger-lift wells were quantitatively described. Gasbarri et al. [5] developed a plungerlift model for low-producing gas wells that do not achieve the desired production rate due to liquid accumulation. This model combines the dynamic characteristics of mechanical plunger-lift systems with reservoir dynamics, considering the frictional effects of liquid segments above and below the plunger and the influence of separator and pipeline pressure on plunger operation, including modeling the transient production behavior after the liquid segment reaches the surface.

Hashmi et al. [6,7] proposed a simplified model of wellbore and formation plunger operation, still based on the Forsgel model, which presupposes that the tubing and jacket annulus can be filled with fluid by closing the production valve. Gupta et al. [8,9] developed a novel plunger gas-lift dynamic model based on the pressure and flow dynamics of fluids in the annulus and the central tubing section of the well, as well as the dynamics of plunger descent and rise in tubing. It is the first example to model a complex plunger lifting system using the standard Hybrid System Model framework.

Zhu et al. [10,11] provided a computationally efficient flow model to account for interactions between loading liquid, producing gas and the lifting plunger in gas wells. Tan et al. [12] built a full-size visible plunger lift equipment to study the process of single flowing plunger lift (SFPL) and normal plunger lift (NPL). Moreover, a transient model was built to simulate SFPL and NPL in different situation. Results show that the SFPL can prevent the liquid from dropping, decrease the bottomhole pressure, and enhance gas production. The SFPL performs well in a certain range, and the SFPL would be out of operation first. A plate has been built to help engineers optimize the SFPL. The paper helps clear the situation of SFPL and provides the theoretical basis for researchers.

Based on different research emphases, scholars have proposed various calculation or optimization models [13–15]. In summary, researchers have conducted extensive studies on the mechanisms, processes, and operating regimes of plunger-lift technology. However, there has been limited research on the drainage capacity of plunger-lift processes. This study aims to address this gap by providing a theoretical basis for the adaptability analysis and selection of plunger-lift processes.

Currently, the oil tubing sizes commonly selected for plunger-lift operations are  $2^{3/8''}$ ,  $2^{7/8''}$ , and  $3^{1/2''}$ . These corresponding plunger-lift processes are referred to as "conventional plunger-lift with different casing sizes" in this paper.

## 2 Plunger Gas Lift Process Lifting Model

The cyclic operation of the plunger lift can be divided into four stages: the plunger ascending stage during the well opening, the continuous flow stage during the well opening, the plunger descending stage during well closing, and the pressure build-up stage during well closing. Among them, the "plunger ascending stage during the well opening" is the most crucial stage, as it determines the success of the plunger lift process and the effectiveness of fluid removal.

## 2.1 Ascending Mechanical Model

The plunger ascending stage refers to the process where the plunger, under the influence of the gas in the formation and the gas in the annulus, leaves the imported locking device and moves upward from the bottom of the well to discharge the liquid from the wellbore. The plunger ascending during well opening consists of two stages: the ascent of the plunger-liquid column in the wellbore and the fluid discharge at the wellhead. The mechanical model of the plunger-liquid column ascent is shown in Fig. 1.



Figure 1: Plunger-liquid column upward movement and force diagram

The mechanical model of the plunger-liquid column in the upward direction can be observed in Fig. 1.

$$F_{\rm p} - F_{\rm r} - (G_{\rm l} + G_{\rm p}) - F_{\rm f} = (m_{\rm l} + m_{\rm p}) \frac{d^2 S}{dt^2}$$
(1)

Among them:  $F_{\rm f} = f_{\rm g} + f_{\rm l}$ 

Where:  $F_p$ -Plunger lower thrust, N;  $F_r$ -Liquid column pressure, N;  $G_l$ -Liquid column weight, N;  $m_p$ -Plunger weight, N;  $m_l$ -Liquid column mass, kg;  $m_p$ -Plunger mass, kg;  $F_f$ -Friction force, N;  $f_g$ -Gas column friction force (Negligible), N;  $f_l$ -Liquid column friction force, N; S-Plunger displacement, m; t-Plunger operating time, s.

## 2.2 Running Casing Pressure Model

During operation, the gas energy accumulated in the annular space of the oil sleeve is capable of exerting pressure on the plunger-liquid column, lifting it to the wellhead. This casing pressure at the wellhead is the minimum operating pressure required for the plunger lift process. At this pressure, the "automatic fluid discharge" requirement can be met, as shown in Fig. 2.



Figure 2: The minimum operating pressure of the plunger lift process

From Fig. 2, it can be observed that if the liquid column reaches the wellhead and remains stationary, the pressure in the oil sleeve is in equilibrium. Therefore, the following relationship holds:

$$P_{\rm c} + \Delta P_{\rm cg} = P_{\rm t} + \Delta P_{\rm tg} + \frac{\left(m_{\rm l} + m_{\rm p}\right)g}{A_{\rm t}}$$
<sup>(2)</sup>

where:  $P_c$ -Casing pressure, Pa;  $\Delta P_{cg}$ -Gas column pressure inside the casing, Pa;  $P_t$ -Wellhead pressure, Pa;  $\Delta P_{tg}$ -Gas column pressure inside the tubing (gas column pressure inside the tubing below the plunger), Pa;  $A_t$ -Tubing cross-sectional area, m<sup>2</sup>.

The length of the liquid column relative to the length of the tubing can be neglected, therefore,  $\Delta P_{cg} \approx \Delta P_{tg}$ .

According to Fig. 2, once fluid discharge starts at the wellhead, the mass of the liquid column continuously decreases, causing the plunger-liquid column to accelerate its ascent. From this observation, it is evident that the energy from the gas well is sufficient to lift the plunger-liquid column to the wellhead and complete the fluid discharge. Therefore, we can determine the minimum casing pressure required for the operation of the plunger lift, denoted as  $P_{cmin}$ .

Combining Eqs. (1) and (2), we have the following relationship:

$$\begin{cases}
P_{\rm cmin} = \frac{F_{\rm p}}{A_{\rm t}} \\
P_{\rm tmin} = P_{\rm tf} = \frac{F_{\rm r}}{A_{\rm t}}
\end{cases}$$
(3)

where:  $P_{\text{cmin}}$ -Minimum casing pressure, Pa;  $P_{\text{tmin}}$ -Minimum oil pressure, Pa;  $P_{\text{tf}}$ -Export pressure, Pa.

Therefore, the minimum casing pressure required for plunger lift operation:

$$P_{\rm cmin} = P_{\rm tmin} + \frac{(m_{\rm l} + m_{\rm p})\,g}{A_{\rm t}} + \frac{F_{\rm f}}{A_{\rm t}} + \frac{(m_{\rm l} + m_{\rm p})}{A_{\rm t}}\frac{d^2S}{dt^2} \tag{4}$$

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Calculating the pressure  $P_{f1}$  generated by the frictional resistance of lifting a 1 m<sup>3</sup> liquid segment plug based on tubing inner diameter, plunger ascent velocity, and formation water density.

$$P_{\rm f1} = \frac{f_1 \rho_1 v_{\rm u}^2}{2d_1 A_1} \tag{5}$$

where:  $P_{f1}$ -1 m<sup>3</sup> Frictional pressure of the liquid column, Pa;  $v_u$ -Average upward speed of the plunger, m/s;  $d_t$ -Inner diameter of the tubing, m;  $f_1$ -Frictional coefficient of the liquid column;  $\rho_1$ -Fluid density, kg/m<sup>3</sup>.

Therefore, the frictional force  $F_{\rm f}$  in Eq. (4) is:

$$F_{\rm f} = \frac{f_1 v_{\rm u}^2}{2d_{\rm t}} m_{\rm l} \tag{6}$$

Since the minimum casing pressure is the result of gas expansion in the annulus at maximum casing pressure, the maximum casing pressure can be calculated using the gas state equation. Neglecting the differences in the deviation coefficients during gas expansion, the maximum casing pressure is:

$$P_{\rm cmax} = P_{\rm cmin} \left( 1 + \frac{A_{\rm t}}{A_{\rm c}} \right) \tag{7}$$

where:  $P_{cmax}$ -Maximum casing pressure, MPa;  $A_c$ -Casing area, m<sup>2</sup>.

## **3** Plunger Lift Process Parameters

#### 3.1 Operating Cycle Count

Operating cycle  $n_p$  is [16]:

$$n_{\rm p} = \frac{24 \times 60 \times 60}{t_{\rm r} + t_{\rm dg} + t_{\rm dl} + t_{\rm s} + t_{\rm c}} \tag{8}$$

where:  $n_p$ -Number of working cycles, cycle/d;  $t_r$ -Plunger rise time, s;  $t_{dg}$ -Downward time of the plunger in gas, s;  $t_{dl}$ -Downward time of the plunger in liquid, s;  $t_s$ -Dwell time of the plunger at the wellhead, s;  $t_c$ -Dwell time of the plunger on the imported locking device, s.

In which:

$$\begin{cases} t_{\rm r} = \frac{L_{\rm c}}{v_{\rm u}} \\ t_{\rm dg} = \frac{L_{\rm c} - h_{\rm l}}{v_{\rm dg}} \\ t_{\rm dl} = \frac{h_{\rm l}}{v_{\rm dl}} \end{cases}$$
(9)

where:  $L_c$ -The imported locking device depth, m;  $v_{dg}$ -Downward speed of the plunger in gas, m/s;  $h_1$ -Height of the liquid column, m;  $v_{dl}$ -Downward speed of the plunger in liquid, m/s.

The upward velocity of the plunger and the downward velocity of the plunger in gas and liquid are related to pipe dimensions, plunger type, etc. During design, they are generally determined based on field measurement data.

#### 3.2 Maximum Liquid Carrying Capacity

Minimum gas volume required per cycle  $V_{g}$  is [16]:

$$V_{\rm g} = 0.2892 F_{\rm gs} A_{\rm t} (L_{\rm c} - h_{\rm l}) \frac{P_{\rm cmax}}{ZT}$$
(10)

where:  $V_g$ -Minimum gas volume required per cycle,  $10^4$  m<sup>3</sup>;  $V_t$ -Volume of the pipe column plugged with liquid segment before well opening, m<sup>3</sup>;  $F_g$ -Slippage coefficient of gas through the plunger and liquid segment plug, typically taken as 1.15; Z-Dimensionless gas compression factor under operating conditions, nondimensional parameter; T-Average temperature of the wellbore, K.

When the plunger's residence time at the wellhead  $t_s$  and the plunger's residence time on the imported locking device  $t_c$  are both zero in Eq. (9), the daily operating cycle count  $n_p$  of the plunger lift operation is maximized, resulting in the maximum daily liquid production under the same cycle liquid unloading conditions.

Daily gas demand  $Q_{\rm g}$  is:

$$Q_{\rm g} = n_{\rm p} V_{\rm g} \tag{11}$$

Daily maximum liquid discharge  $Q_1$  is:

$$Q_{\rm l} = n_{\rm p} \frac{m_{\rm l}}{\rho_{\rm l}} = \frac{24 \times 60 \times 60}{t_{\rm r} + t_{\rm dg} + t_{\rm dl}} \frac{m_{\rm l}}{\rho_{\rm l}}$$
(12)

where:  $Q_g$ -Daily gas demand, 10<sup>4</sup> m<sup>3</sup>/d;  $Q_1$ -Daily maximum liquid discharge, m<sup>3</sup>/d.

## **4** Compute Constraint Conditions

The gas-liquid carrying process is considered the most economical production method for gas wells. For gas wells that meet the requirements of gas-liquid production, the gas-liquid-carrying production method is chosen as the preferred option. This is also the theoretical basis for the development of the optimal tubing drainage gas production process.

The gas-liquid carrying process is based on the critical flow rate method to determine the critical flow rate, and then it is determined based on the relationship between the actual gas production rate and the critical flow rate.

The critical flow rate method suggests that the gas flow velocity determines the carrying capacity of the gas flow, and when the gas flow velocity reaches the critical carrying velocity, liquid droplets can be carried out by the gas.

There are various forms of critical flow rate models, and currently, the classical models are [17–20]:

$$v_{\rm cr} = c \left[ \frac{\sigma \left( \rho_{\rm l} - \rho_{\rm g} \right)}{\rho_{\rm g}^2} \right]^{0.25} \tag{13}$$

where:  $v_{cr}$ -Critical liquid carry velocity, m/s; *c*-Dimensionless critical liquid carry velocity coefficient;  $\sigma$ -Surface tension coefficient, N/m;  $\rho_g$ -Gas density, kg/m<sup>3</sup>.

When gas carries liquid continuously, liquid droplets are the primary form of the liquid phase. Scholars from different countries have obtained different model coefficients c-values based on different shapes of liquid droplets, theoretical derivations, and experimental regressions. The representative c-values for some models [21,22] are shown in Table 1.

Model	Turner	Coleman	Nosseir	He SL	Peng CY	LiM	WangZY
Droplet form	Spherical	Spherical	Spherical	Liquid film shape	Ellipsoidal	Ellipsoidal	Spherical cap shape
c	6.56	5.48	4.55	4.96	4.54	2.5	1.85

**Table 1:** Critical flow rate method liquid carrying model coefficient c value

At present, almost all the major gas fields in China have fitted the suitable liquid-carrying empirical model according to the production data, which shows the complexity of liquid liquid-carrying law. In gas-carrying liquid production, the shape of the droplet is relatively complicated. Through a large number of indoor gas-carrying liquid experiments, the actual shape of the droplet during gas-carrying liquid can be better reflected, and then the coefficient of the model can be obtained by the statistical method, which is closer to the actual situation of gas-carrying liquid production than the theoretical model assuming the shape of the droplet.

From the experimental model coefficients, it can be seen that it is between the Limin model and Coleman model, which indicates that the droplet shape of gas-carried liquid is between spherical and ellipsoidal. In this experiment, droplets are carried by gas through the liquid column, and the shape of droplets is related to the degree of fragmentation of the liquid surface under the action of airflow. Through a lot of experiments, the c value suitable for the target block is 2.90. The coincidence rate of fluid accumulation reached 87%, which is shown in Fig. 3.



Figure 3: The results of the field application test of the model

For the current conventional column of plunger lift, the critical liquid-carrying flow rate determined based on the maximum flow velocity method is shown in Fig. 4.





(c) 3<sup>1/2</sup>" Tubing critical liquid-carrying flow rate

Figure 4: The critical liquid-carrying flow rates under various operating conditions

In Eq. (13), if the density of natural gas is converted to the standard state, the calculation is relatively convenient.

$$\rho_{\rm g} = 3.4832 \times 10^3 \left( \frac{\gamma_{\rm g} p_{\rm wh}}{Z_{\rm wh}(t_{\rm wh} + 273.15)} \right) \tag{14}$$

where:  $\gamma_g$ -The relative density of natural gas, generally 0.59;  $p_{wh}$ -Wellhead oil pressure, MPa;  $t_{wh}$ -Wellhead temperature, °C;  $Z_{wh}$ -Natural gas deviation coefficient at given oil pressure and temperature, generally 0.6~1.1.

The lower the wellbore temperature and the higher the wellbore pressure, the greater the critical liquid-carrying flow rate. At 0°C, the critical liquid-carrying flow rates for different models of plunger lift columns corresponding to different wellhead pressures are illustrated in Fig. 5.



Figure 5: The critical liquid-carrying flow rates of various specifications at 0°C

Based on the field research data, it is possible for wellhead pipelines to experience freezing and blockage below 0°C. The wellhead network manifold pressure is generally less than 3 MPa. Therefore, for subsequent calculations, we will select the parameters from Fig. 4 for the scenario of "Wellhead Temperature: 0°C, Wellhead Pressure: 3 MPa" to perform the calculations.

#### 5 Plunger Lift Liquid Carrying Selection Chart

As evident from the preceding analysis, various operational parameters of the plunger lift process, such as pack-off pressure (the operating casing pressure) and number of operating cycles, are correlated with the speed of plunger movement. QSY01014-2017 provides recommended values for plunger velocities in conventional plunger lift setups: an average plunger ascent speed of 150 to 300 m/min, a plunger descent speed in gas of 60 to 150 m/min, and a plunger descent speed in liquid of 15 to 40 m/min.

Based on the plunger lift daily liquid discharge model equation (Eq. (12)), assuming abundant energy supply in the gas well (due to high gas production or supplemental gas injection), with both opening time for continuous flow and closing time for pressure restoration set to zero, and considering maximum values for plunger ascent and descent speeds, the maximum daily liquid discharge for different specifications of plunger lift columns at varying well depths can be obtained.

Calculation model: Well depth increments (/m) 0:200:5000, periodic liquid discharge increments (/m<sup>3</sup>) 0:0.05:  $Q_1$  (where  $Q_1$  represents the liquid volume of the full wellbore). This approach provides insight into the relationship between periodic liquid discharge and daily liquid discharge for different specifications of plunger lift columns at different well depths. Adhering to the constraints of the plunger lift process, a plunger lift liquid-carrying calculation program is developed to determine the maximum liquid-carrying capacity for columns of different diameters. Subsequently, a plunger lift liquid carrying selection chart for various column specifications at different well depths is plotted, as illustrated in Fig. 6.



(c) 3<sup>1/2</sup>" Tubing plunger gas lift daily maximum liquid discharge rate chart

Figure 6: Conventional tubing plunger gas lift liquid discharge rate chart

As shown in Fig. 5: (). For plunger lift systems with different column diameters, under conditions of sufficient gas volume and appropriate well depth, the larger the column diameter, the greater the

maximum daily liquid carrying capacity. (2). For plunger lift systems with the same specifications, there is a corresponding limit depth for the "maximum daily liquid unloading rate". Below this depth, the "maximum daily liquid unloading rate" can be achieved, and beyond this depth, the "maximum daily liquid unloading rate" gradually decreases. (3). For a  $2^{3/8''}$  column plunger lift system, when lifting wells with a depth shallower than 808 m, the maximum liquid unloading rate is 33 m<sup>3</sup>/d. (4). For a  $2^{7/8''}$  column plunger lift system, when lifting wells with a depth shallower than 742 m, the maximum liquid unloading rate is 50.15 m<sup>3</sup>/d. (5). For a  $3^{1/2''}$  column plunger lift system, when lifting wells with a depth shallower than 560 m, the maximum liquid unloading rate is 75.14 m<sup>3</sup>/d.

# 6 Conclusion

In the case of short periods of high production and long periods of low production, extending the lifespan of gas wells is a crucial task to optimize the cost-efficiency of tight gas wells. Plunger lift technology serves as a pivotal method for managing low-production periods in tight gas wells. Through this study, the following findings have been established:

A mechanical model for the plunger-liquid column ascent in the plunger lift process was developed, allowing for the determination of the operating casing pressure for plunger lift operations. By focusing on the maximum liquid unloading regime, a methodology for calculating the daily maximum liquid unloading rate was derived. The utilization of the critical flow rate method was introduced for the first time as a constraint for determining the maximum liquid carrying capacity in plunger lift operations, laying the groundwork for adaptive analysis of the plunger lift process.

In this paper, for the first time, the conventional plunger lifting liquid carrying chart under different casing sizes is established. This research provides a foundation for the scientific selection of plunger-lift technology and serves as a decision-making reference for developing reasonable plunger-lift work regimes.

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