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## Novel Methodologies for Preventing Crack Propagation in Steel Gas Pipelines Considering the Temperature Effect

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### ABSTRACT

Using the software ANSYS-19.2/Explicit Dynamics, this study performed finite-element modeling of the large-diameter steel pipeline cross-section for the Beineu-Bozoy-Shymkent gas pipeline with a non-through straight crack, strengthened by steel wire wrapping. The effects of the thread tensile force of the steel winding in the form of single rings at the crack edges and the wires with different winding diameters and pitches were also studied. The results showed that the strengthening was preferably executed at a minimum value of the thread tensile force, which was 6.4% more effective than that at its maximum value. The analysis of the influence of the winding diameters showed that the equivalent stresses increased by 32% from the beginning of the crack growth until the wire broke. The increment in winding diameter decelerated the disclosure of the edge crack and reduced its length by 8.2%. The analysis of the influence of the winding pitch showed that decreasing the distance between the winding turns also led to a 33.6% reduction in the length of the straight crack and a 7.9% reduction in the maximum stresses on the strengthened pipeline cross-section. The analysis of the temperature effect on the pipeline material, within a range from  $-40^{\circ}\text{C}$  to  $+50^{\circ}\text{C}$ , resulted in a crack length change of up to 5.8%. As the temperature dropped, the crack length decreased. Within such a temperature range, the maximum stresses were observed along the central area of the crack, which were equal to 413 MPa at  $+50^{\circ}\text{C}$  and 440 MPa at  $-40^{\circ}\text{C}$ . The results also showed that the presence of the steel winding in the pipeline significantly reduced the length of crack propagation up to 8.4 times, depending on the temperature effect and design parameters of prestressing. This work integrated the existing methods for crack localization along steel gas pipelines.

### KEYWORDS

Crack propagation; finite-element; internal pressure; prestressing; steel gas pipeline; temperature effect



## 1 Introduction

The problem of crack propagation along main steel gas pipelines, in the form of avalanche fractures (destructions), is very common. Research has been carried out to solve this issue, most of which reinforced the pipe material and changed the operating conditions of the pipelines. These solutions, however, are not economically viable. For instance, in the Republic of Kazakhstan, this problem was aggravated by the fact that more than 70% of the national gas pipelines were in a deteriorated state [1]. According to Zhangabay et al. [2], open sources [3–6] showed that the main factors that influenced the operation of gas pipelines included pipe material corrosion, stress concentration in structural parts, and structural defects, as well as external influences that resulted in internal pressure and exposure to destructions [7–10]. These factors have led to accidents and human and environmental disasters [11–14]. As mentioned above, the solutions to such problems have been studied by various scientists and specialists. For example, Ilină [15] presented a case study of the bursting of a buried gas pipeline, which aimed to identify the cause of the incident and improve the strength of the pipeline under external loads. It evaluated whether a 20-inch pipeline near road crossings could withstand external loads under corrosion conditions. Such a study was performed based on a finite-element (FE) method where a solution was ultimately proposed to improve the pipeline strength characteristics by introducing protection sleeves equipped with spacer rings. However, only underground pipelines were considered in their work. Mahmood et al. [16] conducted a study based on a review of several gas pipeline accidents, identifying the material and age as critical factors threatening pipeline integrity and leading to corrosion damage. As a result, they proposed a targeted risk mitigation strategy to avoid structural failure, which was of fundamental importance for minimizing the risks associated with pipeline networks. Biagio et al. [17] investigated the roles of structural pipeline geometry, material ductility, and strain hardening on the propagation of ductile cracks [18]. Yang et al. [19] performed a theoretical and experimental study of crack modeling, investigating the crack propagation velocity and gas decompression. Their results demonstrated that, under certain conditions, the crack propagation rate was below the gas pressure break rate, which implied that the prototype pipeline could terminate its breaking. Conversely, Zhang et al. [20] numerically investigated the steel grade of gas pipelines with different corrosion depths that were subjected to internal pressure, showing the influences of the characteristics of the pipe material and cracking varieties that might minimize the damage. Kaputkin et al. [21] proposed a method by changing the configuration of the pipeline cross-section to deal with crack propagation, while Kaputkin et al. [22] provided the classification of the fractures concerning the temperature effect. Shtremel et al. [23] studied the rate of crack propagation on a large pipeline cross-section, with a diameter of 1420 mm, finding that on the large cross-section, the rate of crack propagation became smaller than the initial value by 1.5–2 times. This was due to a decrease in speed which led to the stoppage of crack development in the longitudinal direction. Based on the results of the aforementioned studies, it is concluded that numerical modeling of crack propagation in steel gas pipelines should be conducted based on dynamic deformation models that consider the plastic properties of the materials. Additionally, the strain rate influence on the matter failure and plastic yield process should be considered [24]. To account for the influence of temperature on the process of structure failure, the mathematical model of the problem should incorporate the dependency of stresses on deformations as functions of time and temperature [25]. The problem of deformation and failure of reinforced steel gas pipelines was not previously investigated in such a formulation. According to the literature review, the task that aims to implement methods for the localization of crack propagation in large-diameter pipeline cross-sections under different temperatures requires further studies. Furthermore, in previous works and the existing national [26–28] and international standards [29–33], cost-effective methods for the localization of crack propagation were not proposed. Only changes in the geometric parameters of the pipes and corresponding operational conditions were proposed, which were not acceptable in most cases.

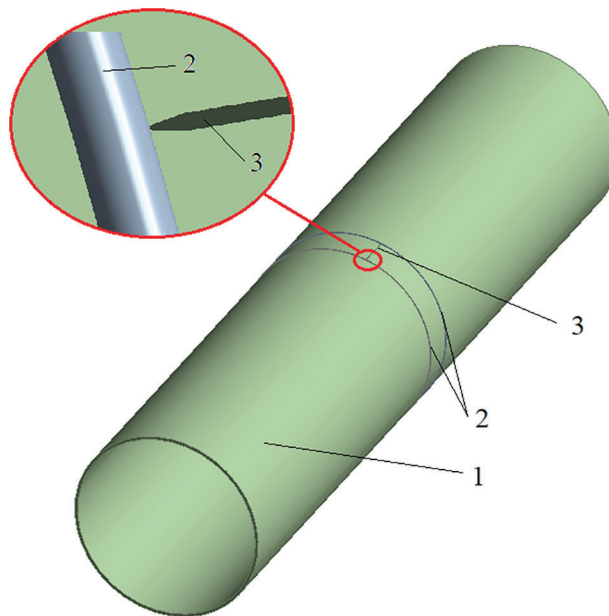
This work is a continuation of a series of investigations dealing with the deformation and failure of main gas pipeline cross-sections with cracks. First, the stressed-deformed state, the break of the steel pipeline cross-section with an initial crack, as well as those in the crack tip area, were investigated by Zhangabay et al. [34]. In the second stage, Zhangabay et al. [35] investigated crack propagation under conditions of temperature loading on the pipe material. Subsequently, targeting the outcomes obtained from the analysis of crack propagation, an approach for the localization of avalanche fractures was proposed according to the prestressing method, which was described in Zhangabay et al. [36–39]. This work is dedicated to developing numerical methodologies for analyzing the influences of the parameters of wire-wound sections of pipes with cracks on the longitudinal crack propagation in steel gas pipelines. Numerical modeling of the dynamic strength was based on a mathematical model that considered the nonlinear relationship between stress, strain, strain rate, and temperature. The deformation process was modeled in four stages: elastic deformation, small plastic deformation, large plastic deformation, and local material failure. Particularly, the material failure was modeled based on the maximum stress criterion (Von Mises's criterion). The mechanical properties of the material, dependent on the ambient temperature, were investigated as working temperature ranged from  $-40^{\circ}\text{C}$  to  $+50^{\circ}\text{C}$ .

## 2 Materials and Methods

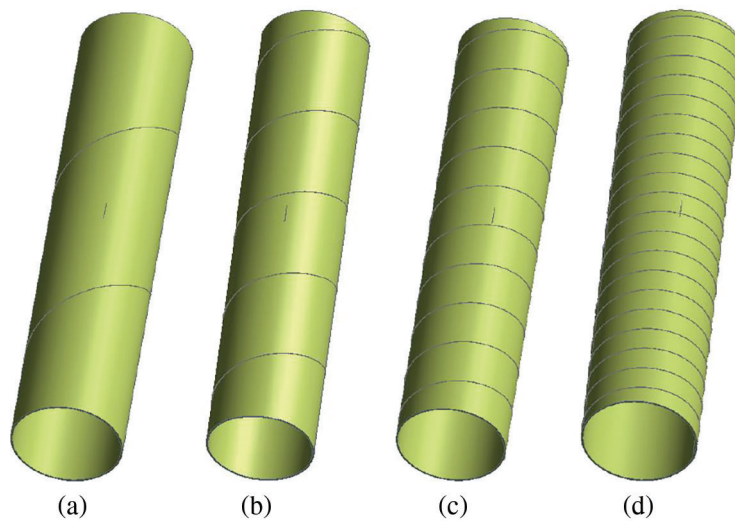
The stressed-deformed state of the gas pipeline was studied by examining the thermo-elastic-plastic features of its steel matter [40]. The constitutive equation of the steel's elastic state took the following form:  $\sigma_{\text{eq}} = E(T) \cdot \varepsilon_{\text{eq}}$  for  $0 \leq \sigma_{\text{eq}} \leq \sigma_y(T)$ , where  $\sigma_{\text{eq}}$  and  $\varepsilon_{\text{eq}}$  are the Von Mises's equivalent stress and strain, respectively. If  $\sigma_{(y)}(T) \leq \sigma_{\text{eq}} \leq \sigma_U(T)$ , the steel's model plastic response was used, where  $\sigma_U(T)$  is the tensile strength for a specified temperature. The steel's plastic response was depicted by the bilinear isotropic hardening (BIH) model [31,32]. The constitutive equation of the material BIH–plastic state took the following form:  $\sigma_{\text{eq}} = \sigma_{(y)}(T) + H(\varepsilon_{\text{eq}} - \sigma_{(y)}(T)/E(T))$ , where  $H = d\sigma_{\text{eq}}/d\varepsilon_{\text{eq}}$  is the hardening modulus,  $E(T)$  is the elastic modulus for a specified temperature, and  $\sigma_y(T)$  is the yield strength for a specified temperature. When the strain rate surpassed the magnitude of  $10^{-2} \text{ s}^{-1}$ , the steel's plastic response was depicted by the Cowper-Symonds Strength (CSS) model, as suggested in references [31,32]. The application of this model afforded the examination of the mechanical characteristics of the X70 class steel depending on the strain rate. The constitutive equation of the steel CSS–plastic state was expressed by  $\sigma_{\text{eq}} = \left[ A(T) + B(T) \cdot (\varepsilon_{\text{eq}}^{\text{pl}})^n \right] \cdot \left[ 1 + \left( D^{-1} \cdot \partial\varepsilon_{\text{eq}}^{\text{pl}}/\partial t \right)^{1/q} \right]$ , where  $A(T)$  is the yield stress at zero plastic strain for a specified temperature,  $B(T)$  is the strain hardening coefficient for a specified temperature,  $\varepsilon_{\text{eq}}^{\text{pl}}$  is the plastic strain,  $n$  is the strain hardening power,  $\partial\varepsilon_{\text{eq}}^{\text{pl}}/\partial t$  is the plastic strain rate, and  $D$  and  $q$  are the strain rate hardening factors [40]. The steel's matter failure was simulated according to the maximal stress criterion (Von Mises's criterion). Notably, when the lumped stresses were greater than the ultimate stress, i.e.,  $\sigma_{\text{eq}} > \sigma_U(T)$ , the breakdown of the local pipeline matter occurred.

The large-diameter cross-section of the Beineu-Bozoy-Shymkent steel gas pipeline with a straight crack was considered in this study. The crack was located in the central part of the pipeline and on its outer surface away from the supports. The thickness of the pipe cross-section was  $H$ , and the crack depth was  $a = 0.5 H$ . Assume  $H = 15.9 \text{ mm}$ , and  $a = 8.0 \text{ mm}$ . The crack was directed in the longitudinal direction of the pipeline and perpendicular to its axial section. Furthermore, its initial width was  $w = 1.0 \text{ mm}$ , while the angle at the crack tip equaled  $20^{\circ}$ . The initial length of the non-through crack was assumed to be  $l = 200 \text{ mm}$  [34,35]. Two variants of the reinforcement of the steel gas pipeline were considered in the novel methodology. The first variant adopted steel single rings at the edges of the crack (Fig. 1).

The second variant involved the steel wire wrapping reinforcement with different winding pitches, i.e., 2.0, 1.0, 0.5, and 0.25 m (Fig. 2).



**Figure 1:** A schematic diagram of the steel gas pipeline with a straight crack reinforced by steel rings at the edges: 1) Pipeline; 2) Single steel rings; and 3) Straight crack



**Figure 2:** A schematic diagram of the steel gas pipeline with a straight crack reinforced by a steel wire wrapping with pitches of a) 2.0 m; b) 1.0 m; c) 0.5 m; and d) 0.25 m

Class X70 steel was used as the material of the pipeline cross-section, whilst class 65 steel was adopted for the winding material. Tables 1 and 2 present the mechanical characteristics of the pipe material (steel X70) and the winding material (steel 65) when the temperature range rose from  $-40^{\circ}\text{C}$  to  $+50^{\circ}\text{C}$  [41,42]. Such a temperature range was considered the operational zone for the Beineu-Bozoy-Shymkent gas pipeline.

It is noted that class X70 steel is a high-strength low-carbon micro-alloyed material with a high impact strength at low temperatures, so a temperature lower than  $-40^{\circ}\text{C}$  does not induce structural changes and metal brittleness in it [42]. Class 65 steel has similar properties. The effect of the tensile strength of the

steel rings on the development of a through crack, and the crack length along the pipeline at a critical increment in internal pressure were investigated. Four values of tensile strength with the following coefficients were assumed: 0.05, 0.25, 0.5, and 0.75. Such coefficients, multiplied by the value of the rupture strength of the steel wire, were used to calculate the different tensile forces of the steel winding. Notably, the tensile force of the steel rings and the wire wrapping  $N$ , based on the tensile strength for class 65 steel, was  $\sigma_U = 980$  MPa. Furthermore, the performance under steel wire rings with different diameters, i.e.,  $d_1 = 4.0$  mm,  $d_2 = 5.0$  mm, and  $d_3 = 6.0$  mm, was investigated. Specifically, the tensile force of the steel rings was determined by the following formula [39]:

$$N = 0.25\pi d^2 \cdot k \cdot \sigma_U, \quad (1)$$

where  $k$  is the coefficient from the critical breaking force of the ring. Considering Eq. (1), the tensile forces of the steel ring, for a given wire diameter, corresponding to the four variants of the prestressing force, are illustrated in Table 3.

**Table 1:** Mechanical characteristics of class X70 steel in the temperature interval of  $[-40^\circ\text{C}, +50^\circ\text{C}]$

Temperature, $T$ ( $^\circ\text{C}$ )	$-40^\circ\text{C}$	$-10^\circ\text{C}$	$+20^\circ\text{C}$	$+50^\circ\text{C}$
Tensile strength, $\sigma_U$ (MPa)	574	572	570	568
Yield strength, $\sigma_y$ (MPa)	525	515	505	495
Elastic modulus, $E$ (GPa)	211	209	206	203.5

**Table 2:** Mechanical characteristics of class 65 steel in the temperature interval of  $[-40^\circ\text{C}, +50^\circ\text{C}]$

Temperature, $T$ ( $^\circ\text{C}$ )	$-40^\circ\text{C}$	$-10^\circ\text{C}$	$+20^\circ\text{C}$	$+50^\circ\text{C}$
Tensile strength, $\sigma_U$ (MPa)	984	982	980	978
Yield strength, $\sigma_y$ (MPa)	795	790	785	780
Elastic modulus, $E$ (GPa)	212	211	210	209

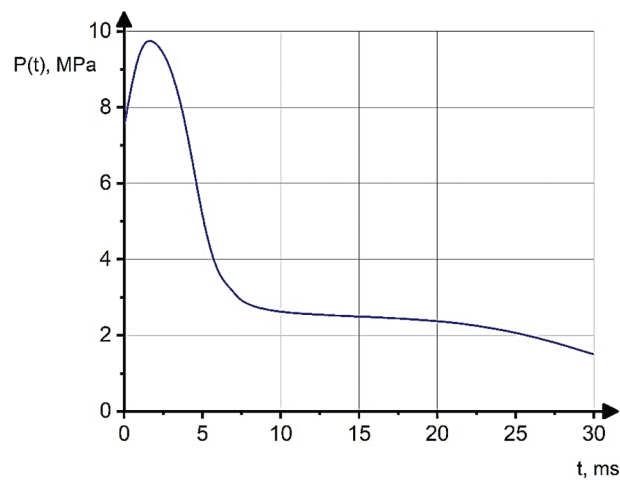
**Table 3:** Tensile forces of the steel rings

Estimated case	$d_1 = 4.0$ mm	$d_2 = 5.0$ mm	$d_3 = 6.0$ mm
0.05 $\sigma_U$	0.62 kN	0.96 kN	1.38 kN
0.25 $\sigma_U$	3.08 kN	4.81 kN	6.93 kN
0.50 $\sigma_U$	6.16 kN	9.62 kN	13.85 kN
0.75 $\sigma_U$	9.24 kN	14.43 kN	20.78 kN

Unsteady internal pressure was defined according to Fig. 3 and based on the experimental data by Nordhagen et al. [43]. Its variations with time corresponded to three stages of loading. The first stage was when the growth of the working pressure was critical. The second stage had a constant critical pressure, whilst the third was when the pressure dropped due to the gas outflow from the through crack.

A numerical solution to the problem was carried out using the FE software package ANSYS-19.2/Explicit Dynamics. As mentioned earlier, this work is a continuation of a series of investigations on

avalanche destruction of main steel gas pipelines. The issue of longitudinal crack propagation along a steel pipeline was addressed by Zhangabay et al. [34]. During the construction of the FE model based on cylindrical shell elements, the solution convergence was observed. According to the relevant results, the FE mesh was fabricated with the “Element Size” set at a magnitude of 0.01. Additionally, the cylindrical shell FE quantity of the pipeline’s thickness was determined with the “Sweep Method” option. Consequently, a uniform mesh was generated, as depicted by Zhangabay et al. [34]. These results were utilized for constructing the FE mesh of the wire. Besides, the FE mesh of the wire was created with the “Element Size” parameter set at 0.001. The FE model, which described the steel pipe section’s dynamic stressed-deformed state and damage (considering the thermal action on the matter’s machinal properties), was outlined by Zhangabay et al. [35]. The FE modeling results of the crack development in the pipe without the wire were then used for validating this investigation.



**Figure 3:** Variations of the unsteady internal pressure with time

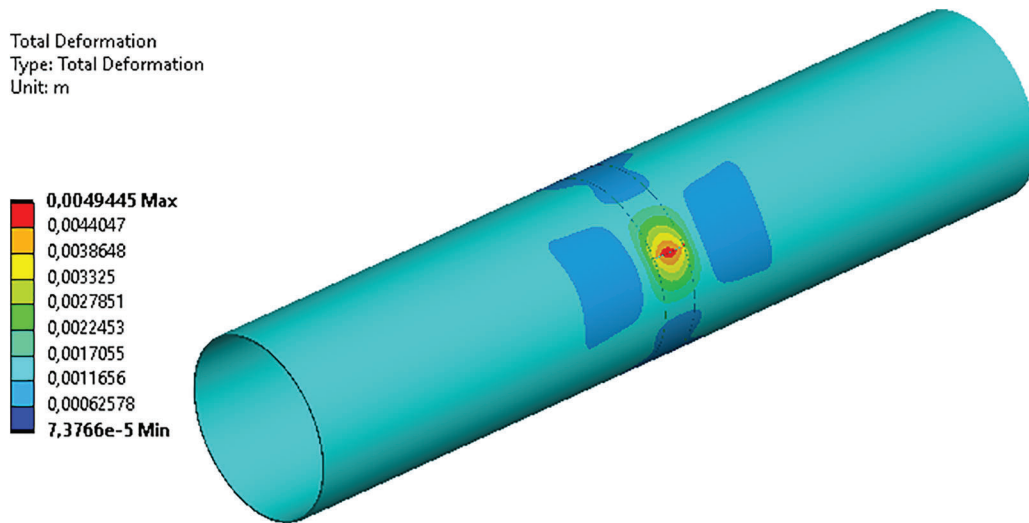
### 3 Results and Discussion

#### 3.1 Influence of the Tensile Force of the Wire Winding Filament on the Crack Propagation on the Steel Rings Located at the Ends of a Non-Through Crack

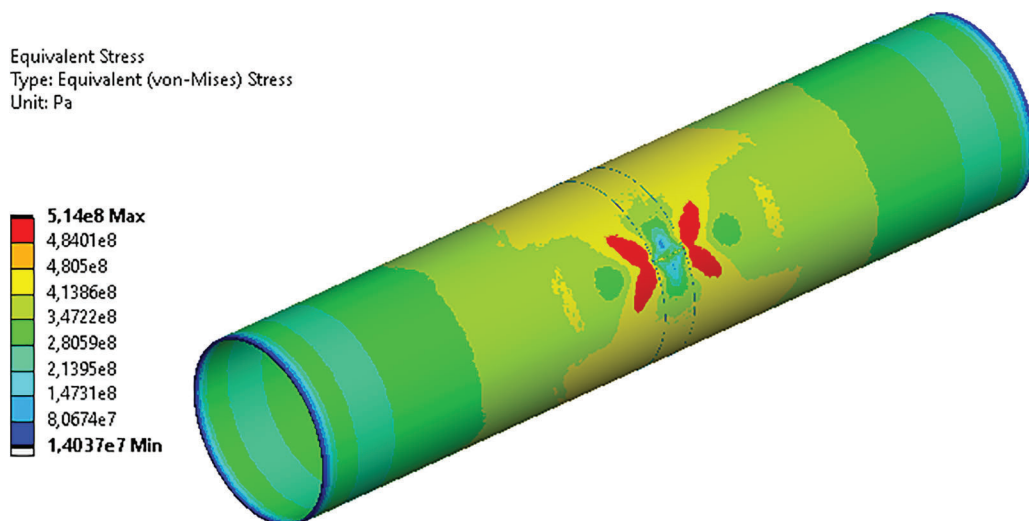
The FE section modeling results of the steel gas pipeline with a longitudinal crack (not reinforced by a steel wire) at temperatures of  $-40^{\circ}\text{C}$ ,  $-10^{\circ}\text{C}$ ,  $+20^{\circ}\text{C}$ , and  $+50^{\circ}\text{C}$  were presented by Zhangabay et al. [35]. The results exhibited uniformity for all the design cases, so the findings for steel rings made by a wire with a diameter of 4.0 mm are summarized below as an example. The analysis of the stressed-deformed variation and the pipe cross-section fracture with a non-slip crack for  $k = 0.05$  are illustrated. It should be noted that, in the presence of steel rings at the ends of the crack, no plastic strains occurred in the damaged area of the pipeline at an internal pressure that reached 9.0 MPa. At the same time, the steel rings retained their integrity and restrained the crack opening. Consequently, the crack remained non-penetrating. Figs. 4–6 respectively show the deformations, stresses, and plastic strains of the pipeline cross-section with a crack at the beginning of its growth. These results corresponded to  $t = 0.8$  ms at which an internal pressure of 9.34 MPa was applied.

The maximum elongations did not exceed 5.0 mm but were localized to the crack area. At the same time, the crack remained non-through.

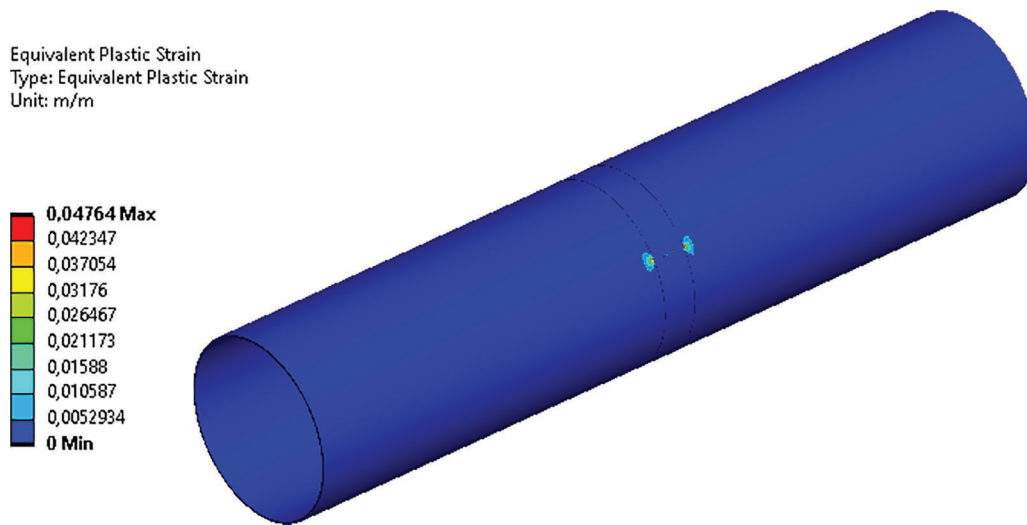
The localization of stresses and plastic strains at the ends of the crack was observed. In short, the steel rings retained their integrity and prevented the crack opening. As the pressure further increased to and maintained at the critical value, the crack became a through one. The gas pressure at the crack edges led to the crack opening, which, in turn, was restrained by the steel rings. Figs. 7–9 respectively show the elongations, stresses, and plastic strains at  $t = 2.0$  ms. By this point, the straight crack reached a length of 1.08 m. The steel rings maintained their integrity by restraining the crack opening, thus slowing down its growth along the longitudinal direction.



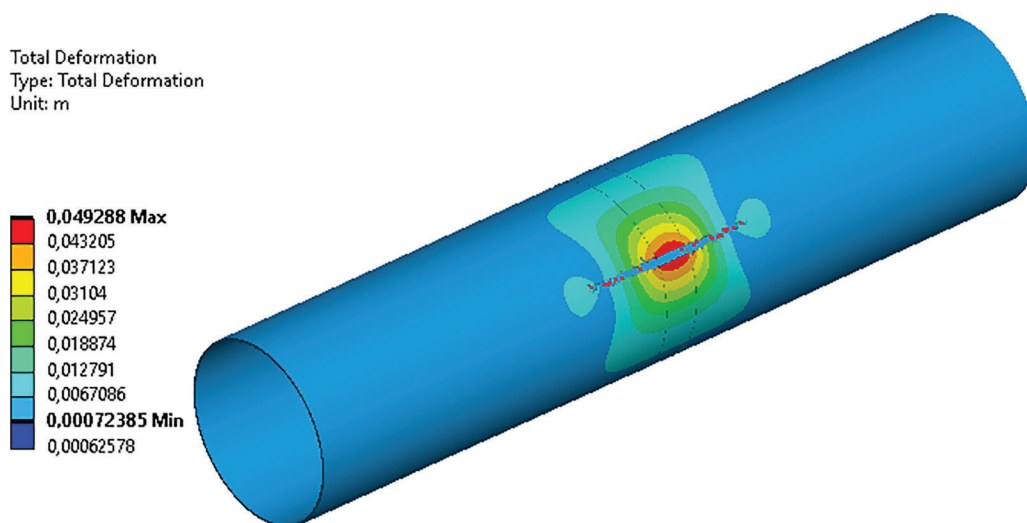
**Figure 4:** Elastic strains for  $P = 9.34$  MPa at  $t = 0.8$  ms ( $P$ : internal pressure)



**Figure 5:** Stresses for  $P = 9.34$  MPa at  $t = 0.8$  ms ( $P$ : internal pressure) (Units: MPa)



**Figure 6:** Plastic strains for  $P = 9.34$  MPa at  $t = 0.8$  ms ( $P$ : internal pressure)

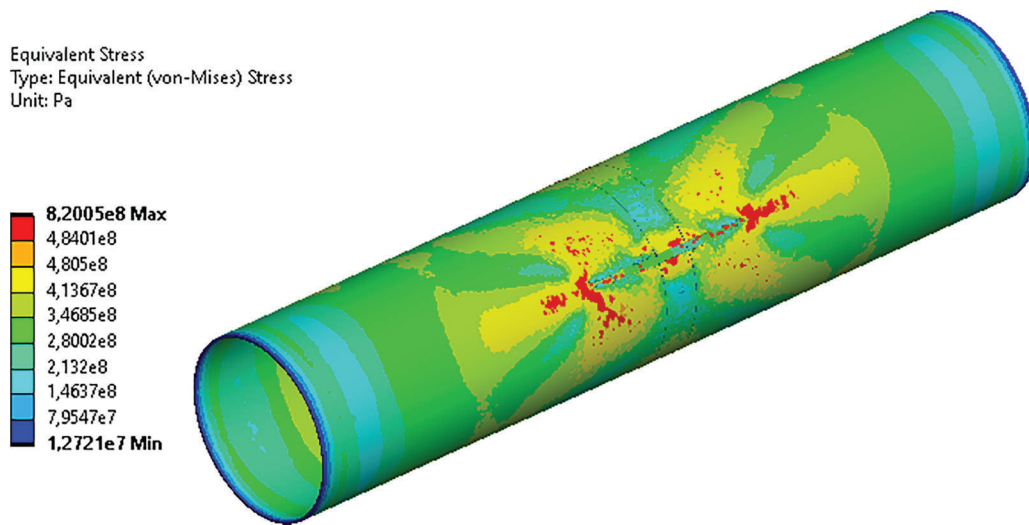


**Figure 7:** Elastic strains at  $t = 2.0$  ms

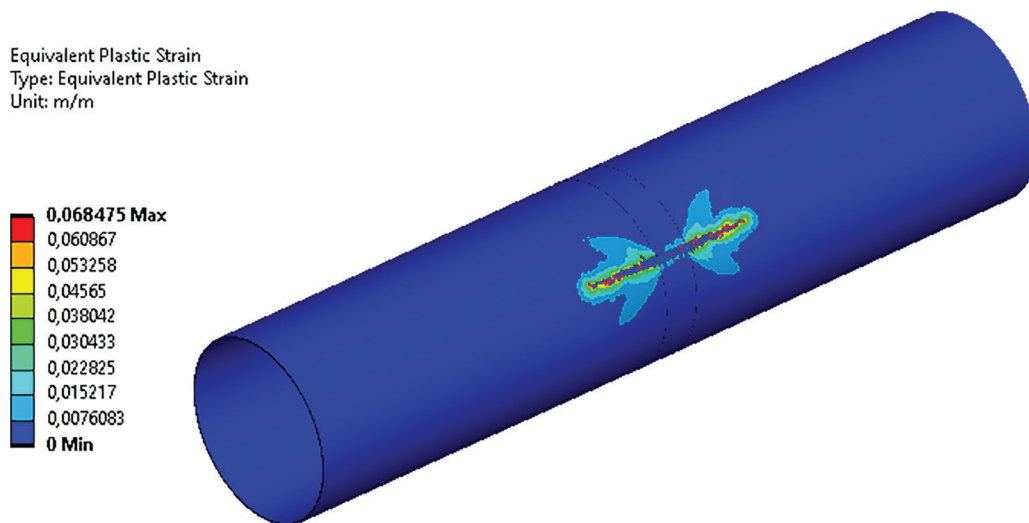
Subsequent development of the crack did not occur along the longitudinal direction, as shown in [Fig. 10](#). Over time, as the pressure decreased, the fracture of the damaged cross-section of the steel pipeline stopped.

Hence, the presence of the steel rings, at the initial stage of the strain, restrained the crack opening and reduced its rapid growth along the longitudinal direction. However, the increment of the internal pressure within the pipe cross-section till the critical values led to the formation of a through crack (and opened its edges). The opened edges put pressure on the steel rings, which also caused their fracture. [Fig. 11](#) illustrates the plastic strains at  $t = 3.5$  ms, which corresponds to the beginning of the ring fracture. Such a fracture occurred at the opened edge of the crack. Moreover, this failure was observed for all the investigated values of the thickness and tensile force applied to the steel wire. Conversely, no crack development along the longitudinal direction was observed after the ring fracture. This could be explained by the accelerated gas outflow through the opened crack, which led to a rapid drop in the internal pressure.





**Figure 8:** Stresses at  $t = 2.0$  ms (Units: MPa)



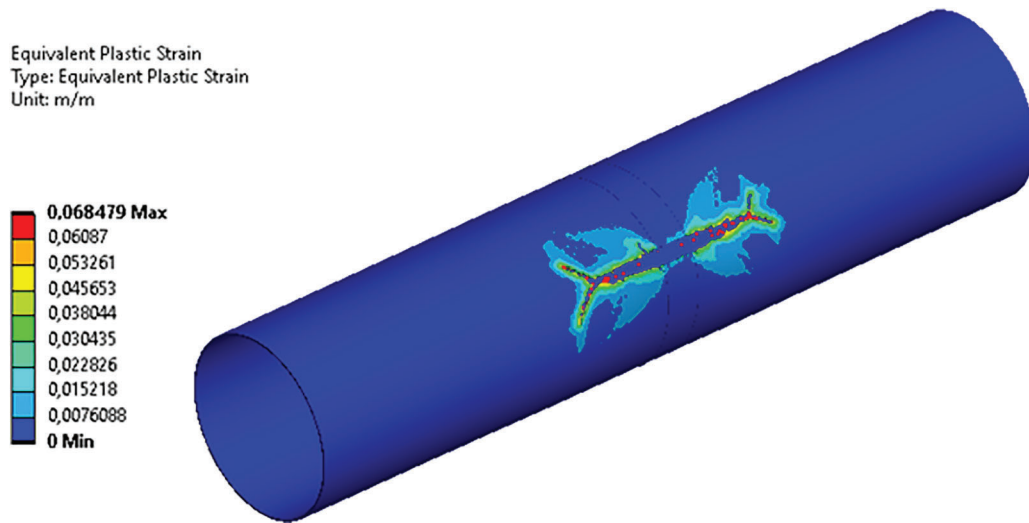
**Figure 9:** Plastic strains at  $t = 2.0$  ms

Table 4 shows the values of the starting time of the ring fracture and the maximum crack lengths for  $k = 0.05$ ,  $k = 0.25$ ,  $k = 0.50$ , and  $k = 0.75$ .

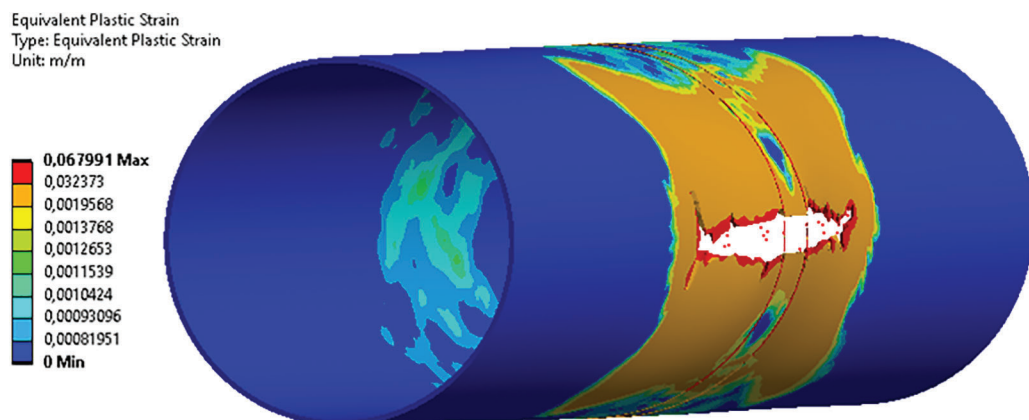
The aforementioned numerical analyses showed that the steel rings with a higher initial tensile force fractured faster. Therefore, it was preferable to strengthen the cracks with the steel rings that had a minimal tensile force of the steel wire thread, e.g., at  $k = 0.05$ .

### 3.2 Influence of the Diameter of the Wire Winding Filament on the Crack Propagation on the Steel Rings Located at the Ends of a Non-Through Crack

The effects of the filament diameter of the steel ring on the development of the through crack and the propagation distance of the through crack along the pipeline at the critical increment in internal pressure were investigated in this section. Specifically, three types of steel wire wrappings with different filament diameters,  $d_1 = 4.0$  mm,  $d_2 = 5.0$  mm, and  $d_3 = 6.0$  mm at  $k = 0.05$ , were considered (Table 5).



**Figure 10:** Plastic strains at  $t = 3.5$  ms



**Figure 11:** Plastic strains at  $k = 0.05$  and  $t = 3.5$  ms ( $k$ : coefficient for the tensile force of the steel winding in percentages)

**Table 4:** Time of the fracture of the steel ring and maximum length of the crack

Estimated case	$k = 0.05$	$k = 0.25$	$k = 0.50$	$k = 0.75$
Maximum crack length	1.46 m	1.48 m	1.54 m	1.56 m
Time of ring fracture	3.3 ms	3.1 ms	2.7 ms	2.1 ms

**Table 5:** Time of the fracture of the steel ring and maximum length of the crack at  $k = 0.05$

Estimated case	$d_1 = 4.0$ mm	$d_2 = 5.0$ mm	$d_3 = 6.0$ mm
Maximum crack length	1.46 m	1.42 m	1.34 m
Time of ring fracture	3.3 ms	3.4 ms	3.5 ms

The aforementioned analyses showed that an increment in the wire diameter led to an increment of the fracture with time and, as a consequence, to a slowing down of the edge crack opening, which eventually reduced the crack length. The obtained results were uniform for all the design cases, so the results for a steel wire with a diameter of 6.0 mm are summarized below as an example. Figs. 12–14 respectively show the stresses of the pipeline cross-section with a crack at the beginning of its growth, i.e., at  $t = 1.58$  ms; at the end of the crack growth along the longitudinal direction, i.e., at  $t = 3.15$  ms; and corresponding to the rupture of the steel rings, i.e., at  $t = 3.5$  ms.

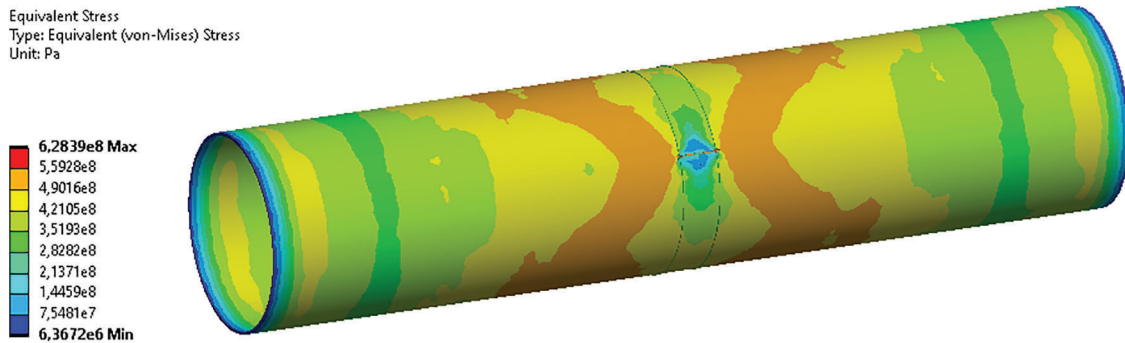


Figure 12: Stresses at  $t = 1.58$  ms (Units: MPa)

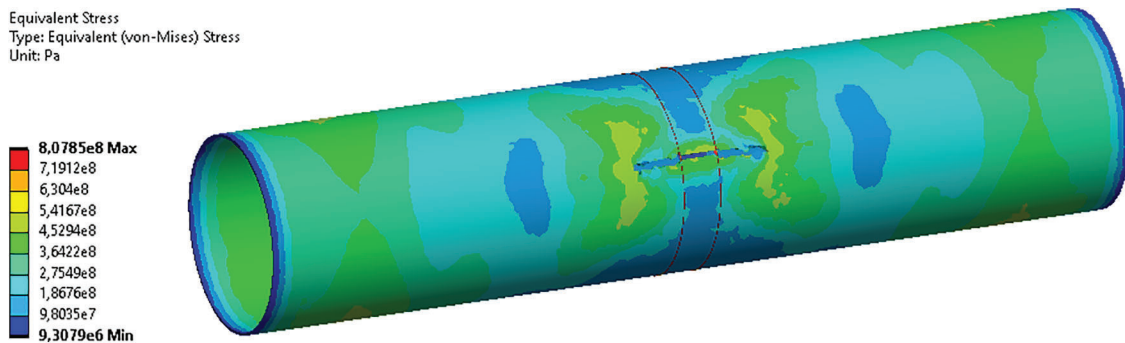


Figure 13: Stresses at  $t = 3.15$  ms (Units: MPa)

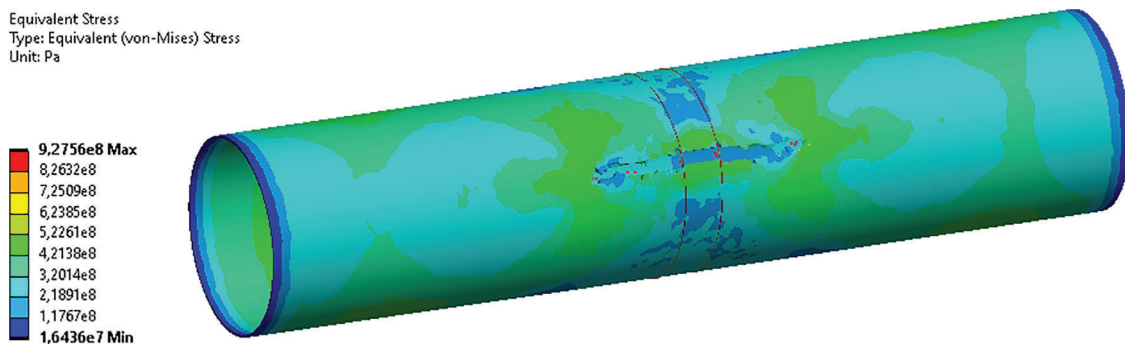
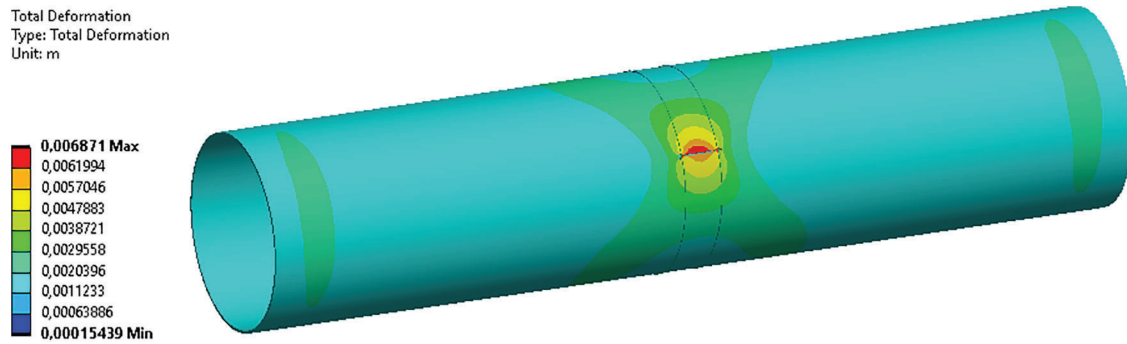
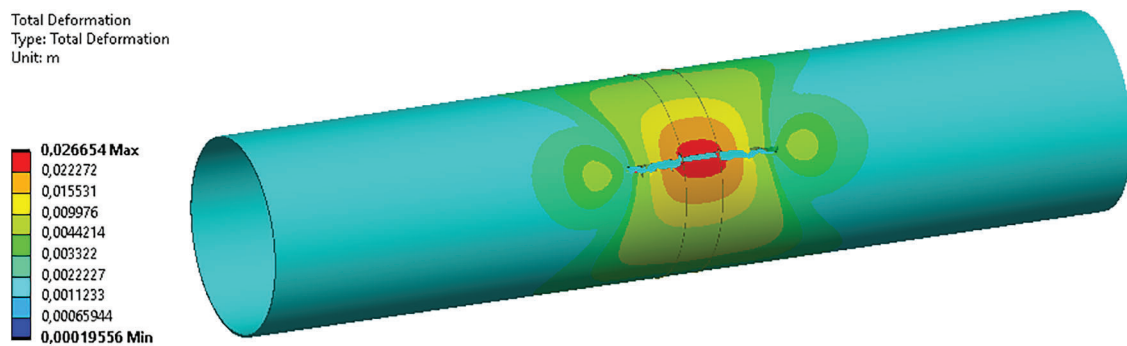


Figure 14: Stresses at  $t = 5.5$  ms (Units: MPa)

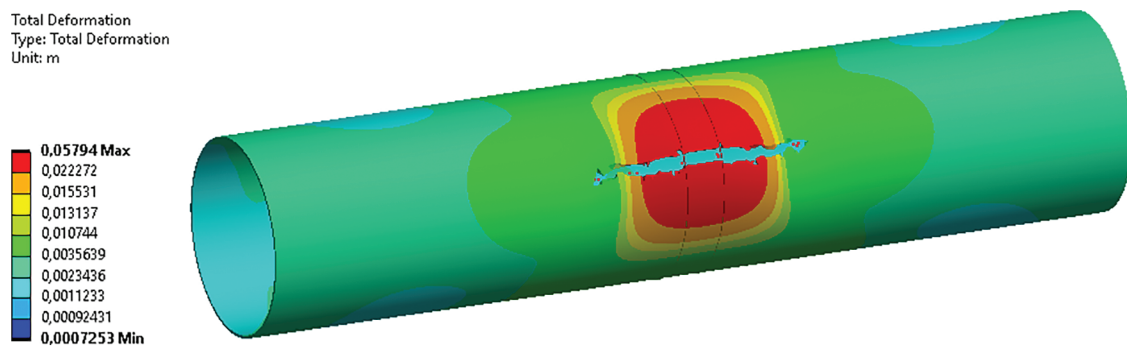
Figs. 15–17 respectively show the deformations of the pipeline cross-section with a crack at the beginning of its growth at  $t = 1.58$  ms,  $t = 3.15$  ms, and  $t = 3.5$  ms.



**Figure 15:** Elastic strains at  $t = 1.58$  ms



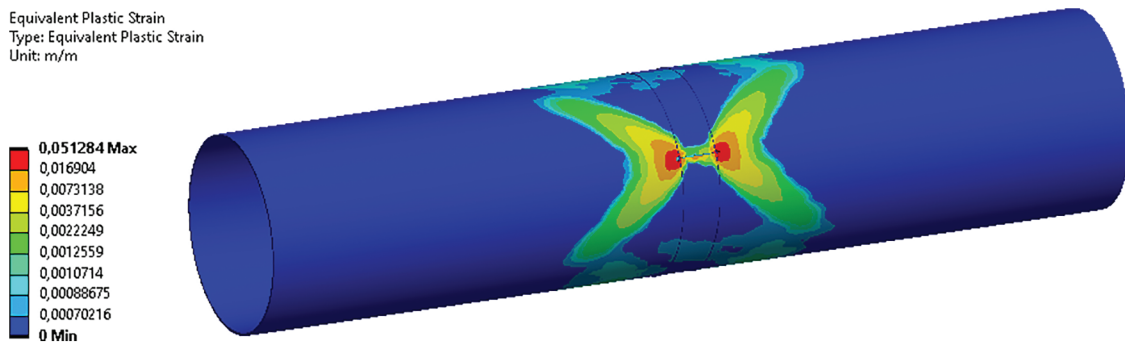
**Figure 16:** Elastic strains at  $t = 3.15$  ms



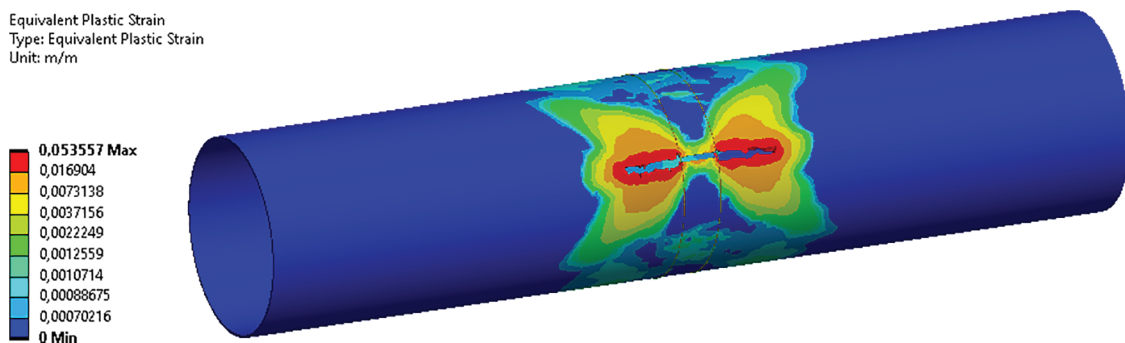
**Figure 17:** Elastic strains at  $t = 3.5$  ms

Figs. 18–20 respectively depict the plastic strains of the pipeline cross-section with a crack at the beginning of its growth at  $t = 1.58$  ms, at  $t = 3.15$  ms, and  $t = 3.5$  ms.

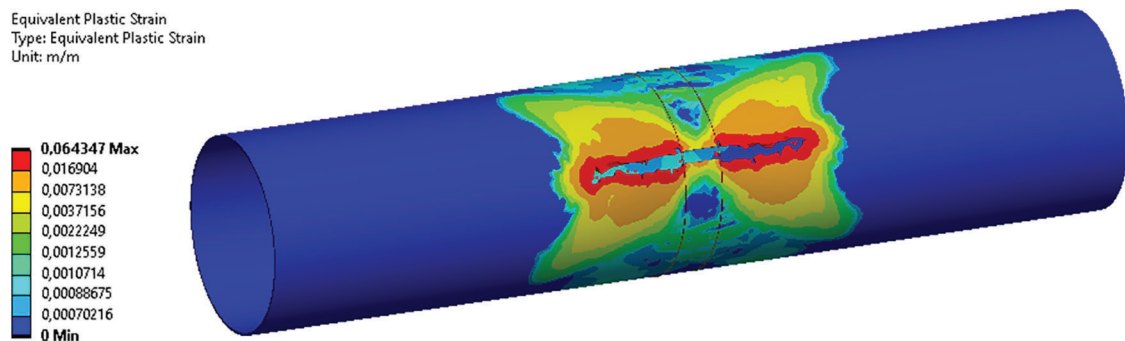
Based on the aforementioned analyses, it could be declared that the steel wire wrapping, under the form of rings at the ends of non-through straight cracks, prevented the avalanche fracture of steel gas pipelines at the working pressure. Furthermore, such a wire wrapping could reduce the crack propagation length in the longitudinal direction corresponding to the critical pressure. In this case, it was preferable to use a tensioned wire wrapping with a larger diameter and the lowest tensile force.



**Figure 18:** Plastic strains at  $t = 1.58$  ms



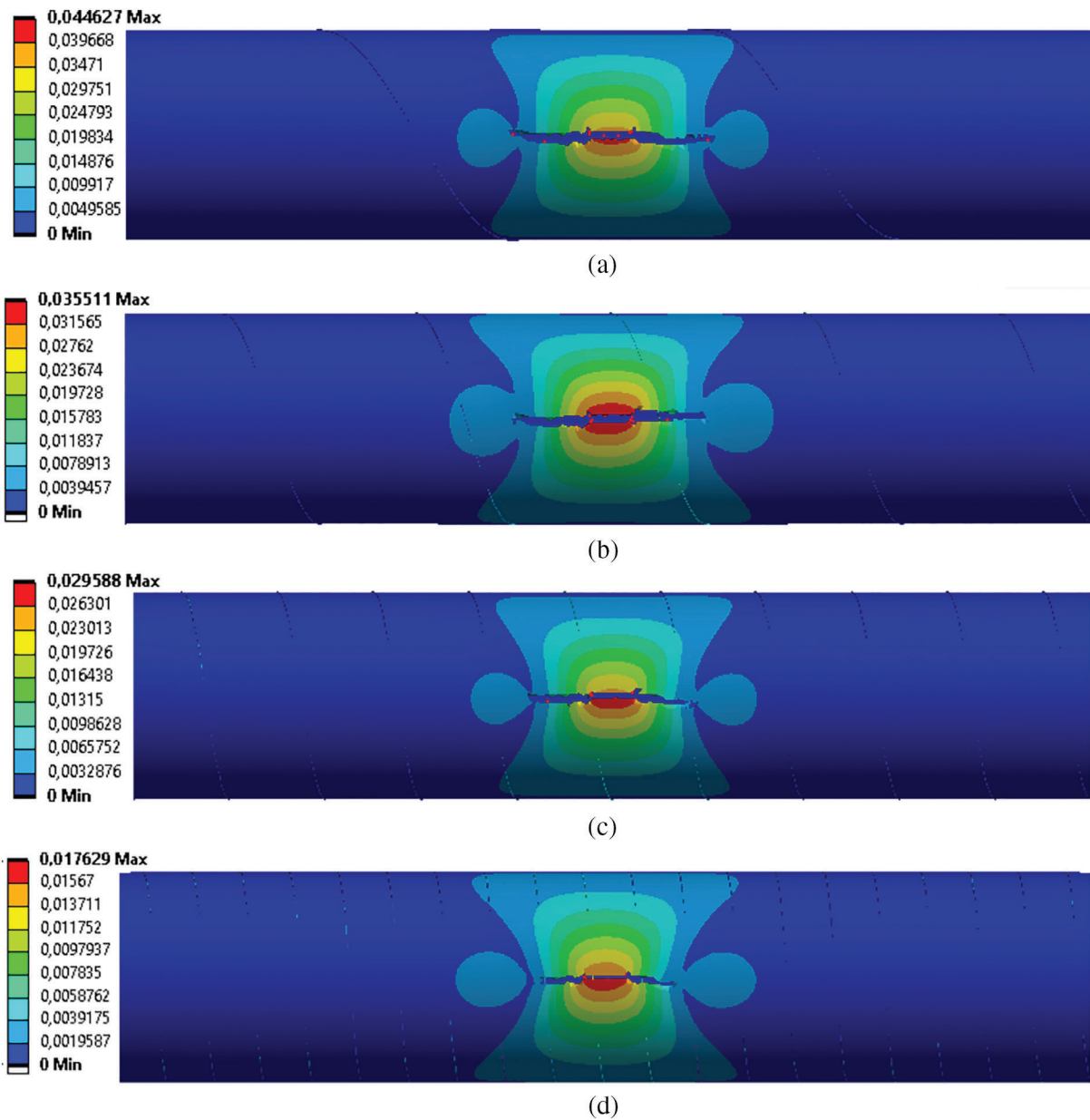
**Figure 19:** Plastic strains at  $t = 3.15$  ms



**Figure 20:** Plastic strains at  $t = 3.5$  ms

### 3.3 Influence of the Winding Pitch of the Wire Winding Thread on Crack Propagation

The cross-section of the steel gas pipeline with a non-through crack strengthened by a steel wire winding was investigated in this section. The wire winding was wound with a uniform pitch, whilst the mutual arrangement of winding turned and the crack was arbitrary. Particularly, the influence of the winding pitch of the steel wire winding thread on the strength and integrity of the cracked pipe cross-section was considered. For this purpose, four winding pitches, 2.0, 1.0, 0.5, and 0.25 m, were considered (Fig. 2). A steel wire with a diameter  $d = 6.0$  mm, tensioned with a force of 1.38 kN, was considered in this case. The unsteady load was set according to Fig. 3. Fig. 21 shows the deformations to the cross-section of the hardened pipeline with a crack. The results were determined at  $t = 3.5$  ms, which corresponded to the maximum development of the crack along the longitudinal direction.



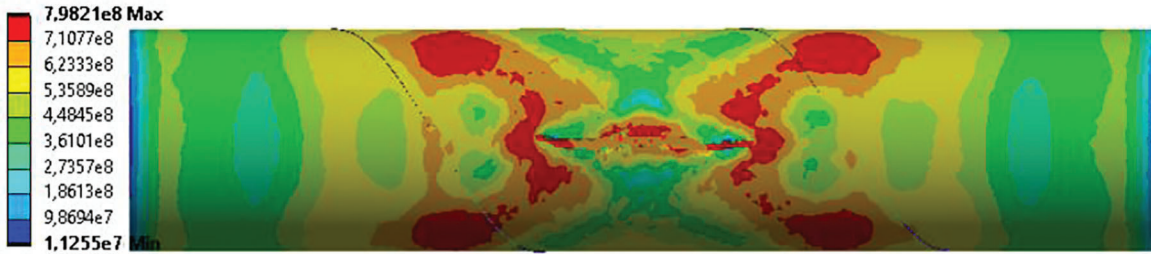
**Figure 21:** Elastic strains of the steel gas pipeline cross-section with a straight crack and reinforced by a steel wire wrapping with a pitch of (a) 2.0 m, (b) 1.0 m, (c) 0.5 m, and (d) 0.25 m

The analysis of the results showed that the reduction of the wire winding pitch in the steel winding led to the reduction of the maximum deformations in the pipeline cross-section. The area of maximum deformations corresponded to the initial location of the non-through crack. However, the critical load led to the formation of a through crack during the strain process, which developed in the longitudinal direction over time. In this case, the presence of the winding reduced the crack length. Table 6 summarizes the values of the straight crack length for each of the investigated winding pitches. Besides, reducing the spacing between the turns of the steel wire winding reduced the length of the linear crack.

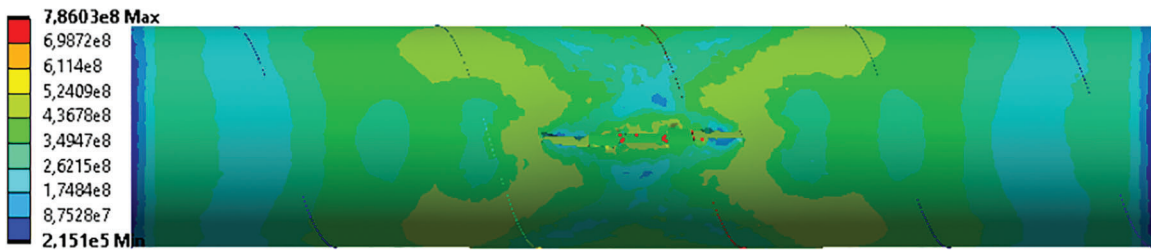
Fig. 22 shows the stresses corresponding to the cross-section of the reinforced steel gas pipeline with a crack at  $t = 3.5$  ms.

**Table 6:** Maximum crack length of the reinforced pipeline cross-section

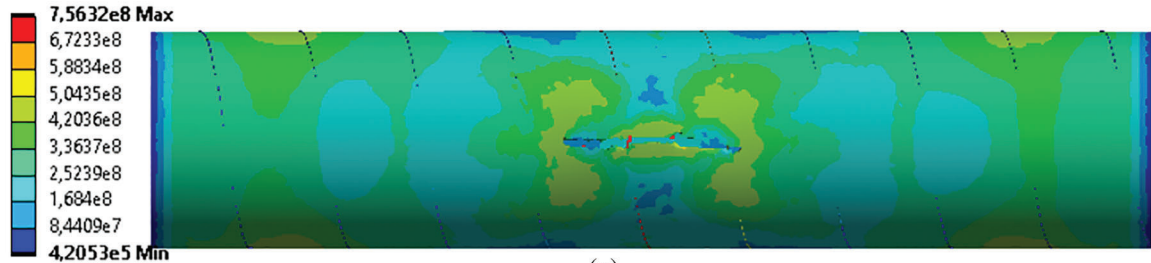
Winding pitch	2.0 m	1.0 m	0.5 m	0.25 m
Crack start coordinate (z)	1.40 m	1.47 m	1.53 m	1.61 m
Crack end coordinate (z)	2.41 m	2.45 m	2.32 m	2.28 m
Maximum length of the straight crack	1.01 m	0.98 m	0.79 m	0.67 m



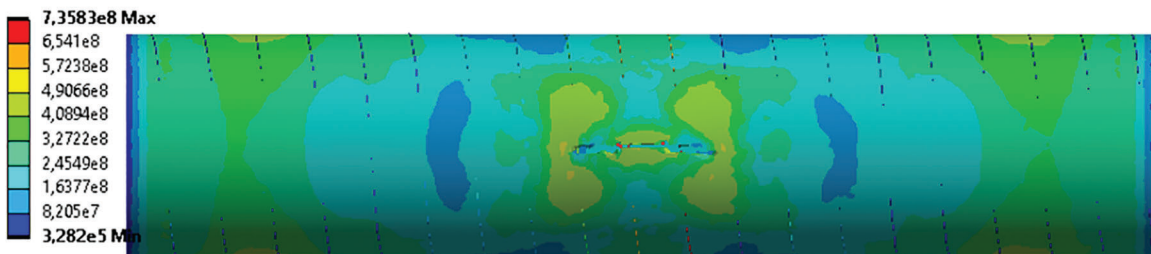
(a)



(b)



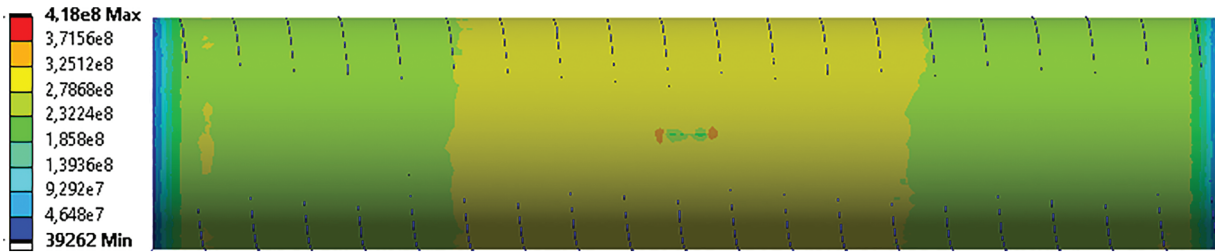
(c)



(d)

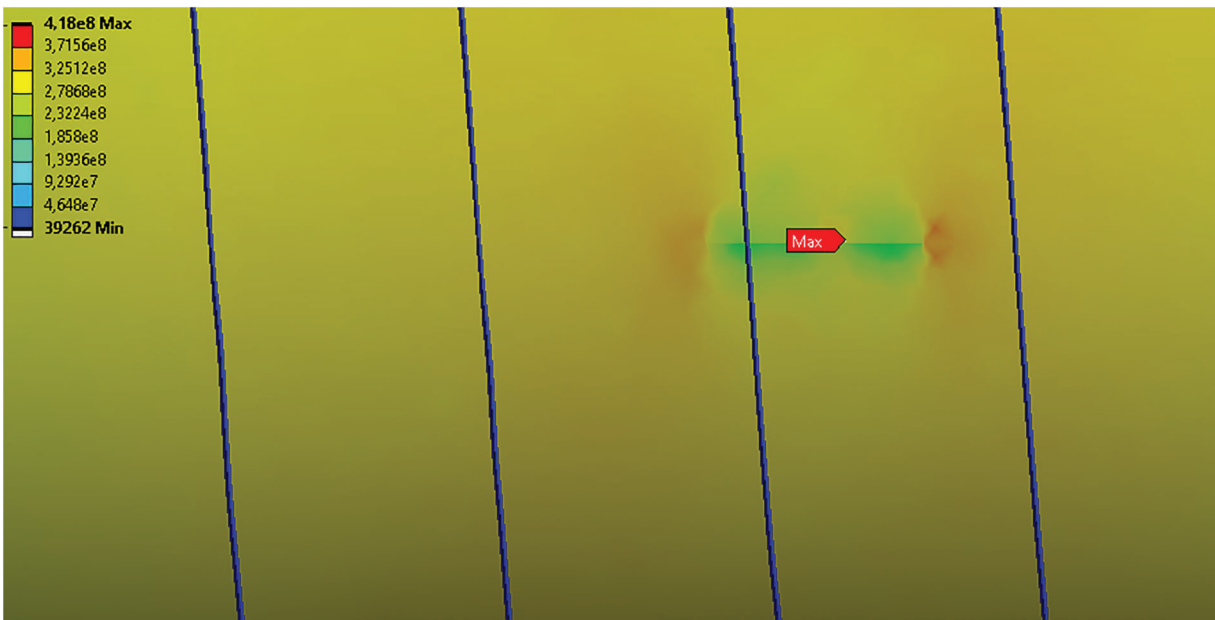
**Figure 22:** Stresses of the steel gas pipeline cross-section with a straight crack and reinforced by a steel wire wrapping with a pitch of (a) 2.0 m, (b) 1.0 m, (c) 0.5 m, and (d) 0.25 m (Units: MPa)

Notably, the reduction of the distance between the winding turns led to a slight stress reduction in the pipeline cross-section. An insignificant localization of the stresses was observed in the area of the crack tips. Investigations on the stress-state in the steel pipeline at the working pressure were performed here. Specifically, the pressure was given by a piecewise linear function:  $-P(t) = P_{kr} \cdot t$  when  $0 \leq t \leq 1.0$  ms, and  $-P(t) = P_{kr}$  for  $t \geq 1$  ms. It was found that the steel pipeline with a non-through crack reinforced by the wire winding with a pitch of 0.25 m retained its integrity. Fig. 23 shows the stresses in the damaged pipeline cross-section at  $t = 5.0$  ms.



**Figure 23:** Stresses of the steel gas pipeline cross-section with a straight crack and reinforced by a wire winding with a pitch of 0.25 m under the working pressure (Units: MPa)

The crack remained non-through and did not significantly affect the deflected shape of the steel pipeline. Fig. 24 shows the stresses surrounding the crack area. Notably, all the aforementioned findings were obtained at  $T = +20^\circ\text{C}$ .



**Figure 24:** Stresses surrounding the crack area at  $T = +20^\circ\text{C}$  (Units: MPa)

### 3.4 Analysis of the Temperature Effect on the Dynamics of the Wire Winding Reinforced Cross-Section of the Steel Gas Pipeline with a Non-Through Crack

The effect of temperature on the development of the non-through crack in the wire winding-reinforced cross-section of the steel pipeline was studied in this section. The winding had a pitch of 0.25 m and was

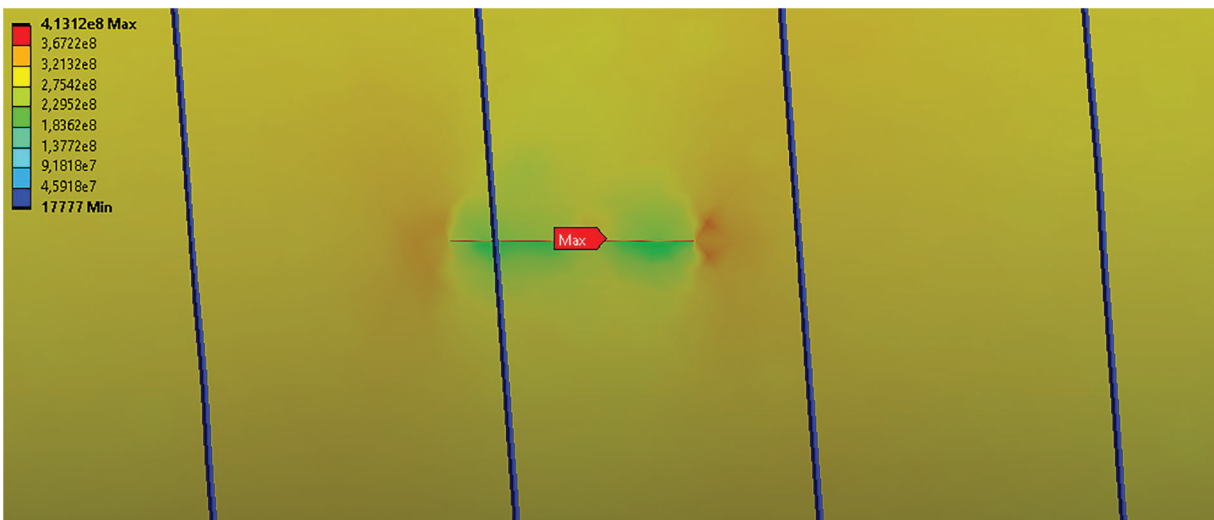


made by a steel wire with a diameter  $d = 6.0$  mm and stretched with a tensile force of 1.38 kN. These variables were chosen because the damaged pipeline cross-section retained its integrity at the operating pressure, and the non-through crack with a depth of half of the pipeline thickness did not develop into a through crack. Moreover, it did not lead to the avalanche fracture of the pipeline. The analysis showed that the critical pressure led to the pipeline fracture over the entire range of the temperature. Qualitatively, the obtained findings corresponded to the results above for a temperature of  $+20^{\circ}\text{C}$ . The variations in the start and end coordinates of the crack along the  $O_z$  axis and in the length of the straight crack as a function of temperature are illustrated in Table 7. For the steel winding reinforced cross-section pipeline, the change in temperature within the established range led to insignificant variations in the crack length.

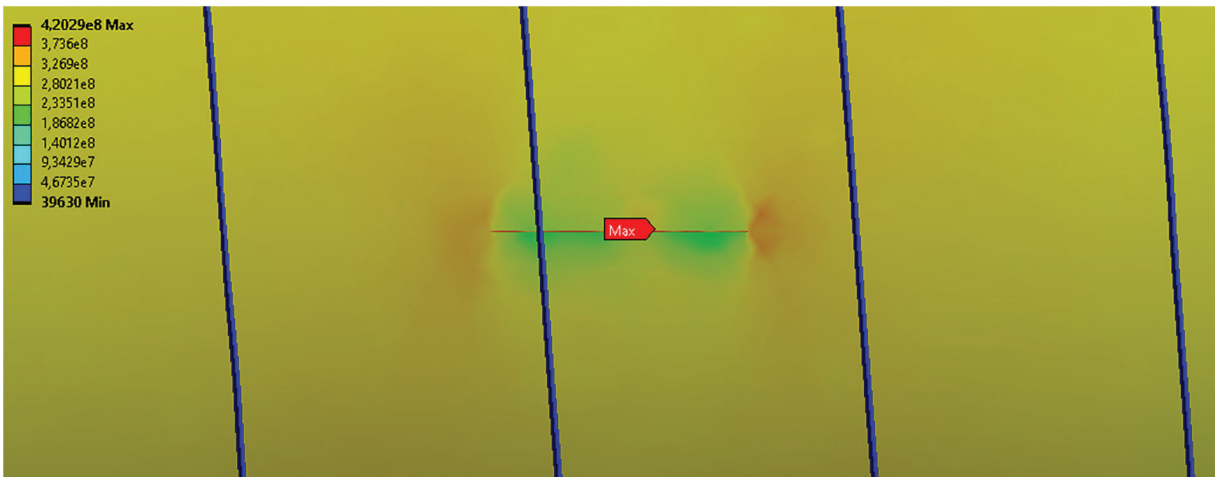
**Table 7:** Temperature effect on the maximum crack length of the pipeline reinforced under a 0.25 m winding pitch

Temperature	$-40^{\circ}\text{C}$	$-10^{\circ}\text{C}$	$+20^{\circ}\text{C}$	$+50^{\circ}\text{C}$
Crack start coordinate ( $z$ )	1.63 m	1.62 m	1.61 m	1.60 m
Crack end coordinate ( $z$ )	2.27 m	2.28 m	2.28 m	2.28 m
Maximum length of the straight crack	0.64 m	0.66 m	0.67 m	0.68 m

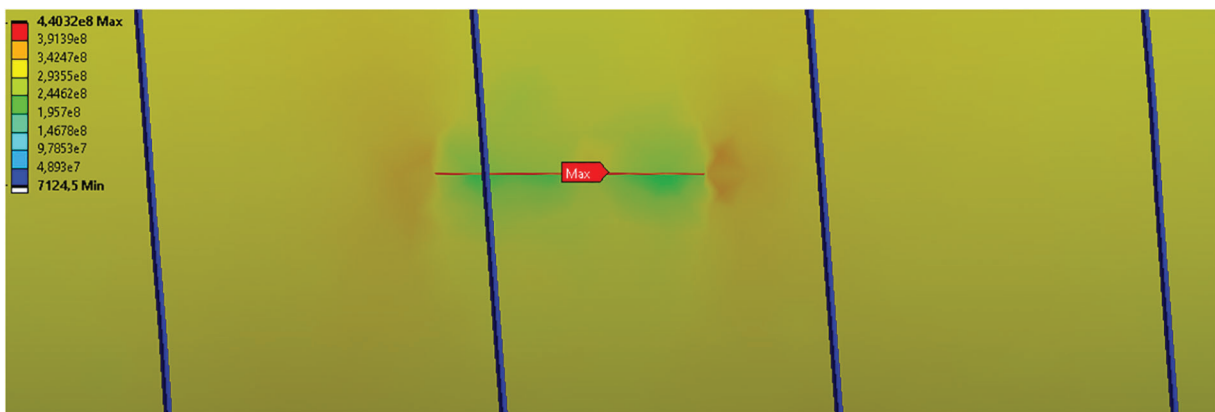
Similarly, the results obtained at the working pressure qualitatively corresponded to those at  $T = +20^{\circ}\text{C}$ . For the whole considered temperature range, the integrity of the pipeline cross-section with a non-through crack was preserved. Figs. 25–27 show the stresses surrounding the crack area for the assumed range of temperature. In all cases, maximal stresses were observed in the central part of the crack. For the remaining parts of the pipeline, no critical stress localization was present. Particularly, a slight increment in the maximum stresses as the temperature dropped was observed. Consequently, the maximum stress at  $+50^{\circ}\text{C}$  was 413 MPa, whilst the maximum stress at  $-40^{\circ}\text{C}$  was 440 MPa. The obtained values were definitively lower than the elastic limits which, in turn, corresponded to the absence of plastic strains, thus leading to the pipeline fracture in the non-through crack area.



**Figure 25:** Stresses surrounding the crack area at  $T = +50^{\circ}\text{C}$  (Units: MPa)



**Figure 26:** Stresses surrounding the crack area at  $T = -10^{\circ}\text{C}$  (Units: MPa)



**Figure 27:** Stresses surrounding the crack area at  $T = -40^{\circ}\text{C}$  (Units: MPa)

FE investigations of the large-diameter cross-section of the Beineu-Bozoy-Shymkent steel gas pipeline with a non-through straight crack, strengthened by a steel wire wrapping, were executed (Figs. 1–2). The numerical solution was implemented using the FE software package ANSYS-19.2/Explicit Dynamics, whilst the spatial discretization was based on the equation of motion [34,35]. The influence of the winding design parameters on crack propagation was analyzed considering the temperature effect on the pipeline material (Tables 1 and 2). Particularly, the influence of the tensile force applied to the steel wire winding filament was investigated (Table 3). The presence of the steel wire rings at the edges of the crack (at the initial stage of strain) restrained the crack opening and reduced its growth along the longitudinal direction (Figs. 4–6). However, the increase of the internal pressure within the pipeline cross-section until the critical values led to the formation of a through crack and opening of its edges. The opened edges pressed the steel wire and thus led to its rupture (Figs. 7–10). Therefore,  $t = 3.5$  ms corresponded to the beginning of the rupture of the steel wire rings. After such rings broke, no crack development in the longitudinal direction was observed. Specifically, the steel rings, with a higher initial tensile force, were destroyed faster. Considering these circumstances, it was preferable to strengthen the cracks with the rings when the tensile force of the wire thread was minimal ( $k = 0.05$ ), which was 6.4% more effective than that at the maximum value ( $k = 0.75$ ), as displayed in Table 4.

The influence of the diameter of the steel wire was also studied, showing that its increment from 4.0 mm to 6.0 mm (Table 5) led to the failure of the steel wire. The equivalent stresses over time are shown in Figs. 12–14. We noticed that such stresses increased by 32% from the beginning of the crack growth until the moment of the wire fracture. Furthermore, the increment in winding diameter slowed down the edge crack opening and led to a decrement in its length by 8.2% (Figs. 15–17). Consequently, the steel wire wrapping in the form of rings at the ends of non-through straight cracks prevented the avalanche failure of the steel pipeline at the operating pressure and reduced the length of crack propagation in its longitudinal direction at the critical pressure (Figs. 18–20). Regarding the influence of the winding pitch of the steel wire winding filament (Fig. 2), it was found that the decrement in the winding pitch led to a decrease in the maximum deformations in the pipeline cross-section (Fig. 21). Moreover, the critical load led to the formation of a through crack and its development in the longitudinal direction. Particularly, the winding led to a decrement in the crack length. Reducing the distance between the winding turns also led to a 33.6% reduction in the length of the straight crack (Table 6) and a 7.9% maximum stress decrease in the pipeline cross-section (Fig. 22). Under the working pressure, the crack remained non-through and, consequently, the pipeline cross-section did not fail (Figs. 23 and 24).

In conclusion, the temperature effect on the development of the non-through crack of the wire-wrapped hardened cross-section of the pipeline was studied within a temperature range of  $-40 - +50^{\circ}\text{C}$  (Figs. 25–27). The critical pressure led to the pipeline fracture within the whole temperature range. The temperature variations led to insignificant variations in crack length, which equaled a maximum difference of 5.8% (Table 7). As the temperature dropped, the crack length decreased. Besides, at the operating pressure, the integrity of the pipeline cross-section with the non-through crack was maintained. Within the temperature range, the maximal stresses were observed surrounding the central part of the crack, where the maximum stress was 413 MPa at  $+50^{\circ}\text{C}$  and 440 MPa at  $-40^{\circ}\text{C}$  (the difference was 6.1%). For the remaining parts of the pipeline, no localization of critical stresses was observed. Considering the obtained results and the previous findings [34,35], it was proved that the presence of the winding significantly reduced the length of crack propagation up to 8.4 times, depending on the temperature effect and design parameters of prestressing. Notably, the results of this work could be used in the design of new steel gas pipelines and the strengthening of existing ones. It is also worth noting that a related patent has been obtained in the Republic of Kazakhstan [44]. Cracks characterized by a regular shape were assumed in this study, whilst cracks having irregular shapes would be considered as a continuation of this research. Furthermore, the authors have planned to expand the scope of this work and analyze steel gas pipelines reinforced by different composite materials to solve the problem of localization of avalanche failures.

#### 4 Conclusions

A localization method for non-through straight cracks along steel gas pipelines by applying steel wire wrappings was considered in this study. Novel methodologies for selecting the wrapping parameters, e.g., the pitch between turns, the thread tensile force of the steel wire in the turn, and the thickness of the wire, were proposed. The methods were based on FE modeling of the structural integrity and failure of the wire-wrapped reinforced cross-section of the steel pipe with a crack. A refined nonlinear dynamic model of the pipeline's stress-strain state was utilized for the FE modeling. It considered the effects of temperature on the mechanical characteristics, yield, and tensile limits of the steel pipe and wires. Also, the influence of the strain rate on the dynamic yield strength of the steel materials was considered. Moreover, the FE modeling assumed that the local failure of the steel materials was due to the large plastic deformations. The stress-strain state in the cross-section pipe, due to the thread tensile force of the wire, was simulated according to an accurate geometric model of the wire wrapping. The numerical studies yielded several practical results. According

to the results of previous studies and the current work, the following conclusions were drawn regarding the prevention of crack propagation in steel gas pipelines considering the temperature effect:

- 1) Tensioning of steel pipelines by a single ring or wire windings was preferably carried out at the minimum value of the thread tensile force ( $k = 0.05$ ), which was 6.4% more effective than that at the maximum value ( $k = 0.75$ ).
- 2) The increment of the steel winding diameter from 4.0 to 6.0 mm increased its destruction with time. The values of the equivalent stresses increased by 32% from the beginning of the crack growth until the moment of steel wire breaking. Also, the increase in winding diameter slowed down the edge crack opening and reduced its length by 8.2%.
- 3) Decreasing the winding pitch led to a decrease in the maximum deformations in the pipeline cross-section. Furthermore, decreasing the distance between the winding turns also led to a 33.6% decrease in the length of the straight crack and a 7.9% reduction in the maximum stresses in the reinforced pipeline cross-section (Fig. 22).
- 4) Considering the effect of temperature on the mechanical characteristics of the steel in the pipeline, the crack length varied by a maximum difference of 5.8% from  $-40^{\circ}\text{C}$  to  $+50^{\circ}\text{C}$ . The crack length decreased as the temperature dropped. Moreover, within the temperature range, the maximum stresses could be observed surrounding the central area of the crack. In this work, the maximum stresses corresponded to 413 and 440 MPa at temperatures of  $+50^{\circ}\text{C}$  and  $-40^{\circ}\text{C}$ , respectively.
- 5) The analysis above concluded that the winding in steel gas pipelines significantly reduced the crack propagation length (up to 8.4 times), depending on the temperature effect and design parameters of prestressing.

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**Availability of Data and Materials:** The data that support the findings of this work are available from the corresponding authors upon reasonable request.

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